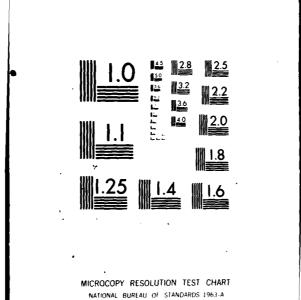
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20. ABSTRACT CONTINUED

- Thin-walled Hollow Cylinder tests using longitudinal and torsional cyclic excitation, both separately and in combination.
 - 3. Large scale shaking table tests on slope models, using horizontal and vertical cyclic excitation, both separately and in combination.

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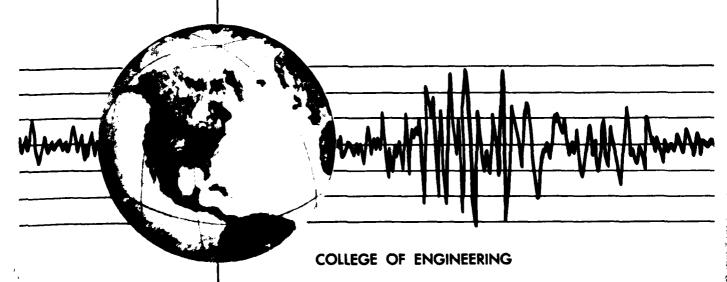
EARTHQUAKE ENGINEERING RESEARCH CENTER

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by

PATRICK M. GRIFFIN
WILLIAM N. HOUSTON

A Report on Research Sponsored by the U.S. Army Research Office



 $\textbf{UNIVERSITY} \ \ \textbf{OF} \ \ \textbf{CALIFORNIA} \ \ \cdot \ \ \textbf{Berkeley}, \textbf{California}$

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College of Engineering
University of California
Berkeley, California

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Abstract

An experimental research program based on laboratory test studies and scaled slope model tests was conducted with specimens of Monterey No. 0 sand. The principal objective of the research was to study the effects of interactive coupling during combined compression (normal) and shear loading on the response of sands to dynamic loading. The research program included the following experimental studies:

- Resonant Column tests on cylindrically shaped specimens using longitudinal and torsional excitation, both separately and in combination.
- Thin-walled Hollow Cylinder tests using longitudinal and torsional cyclic excitation, both separately and in combination.
- 3. Large scale shaking table tests on slope models, using horizontal and vertical cyclic excitation, both separately and in combination.

During the course of this research significant effects were observed during combined compression and shear cyclic loading. The primary observed effect of combined loading was the more rapid degradation of modulus with strain than would otherwise occur.

Two methods were developed and presented for calculating the degradation of compression modulus with strain under combined dynamic loading conditions. The first of these, called the Strain Ratio Method, requires the computation of either the instantaneous or an overall average ratio of shear strain amplitude to compression (normal) strain amplitude. The amount of additional degradation in compression modulus due to interactive

coupling may then be determined by reference to a set of typical strain ratio curves which are presented in Figure 7-7 on page 138.

The second method developed is called the Octahedral Shearing Strain Method. This method requires the computation of either the instantaneous or an overall average value of the octahedral shearing strain amplitude. This may be calculated from the strain tensor using either Equation 2-42 on page 18, or Equation 2-44 on page 19. Once this value is determined, the total degradation in compression modulus due to strain, including interactive coupling effects, may be determined by reference to a set of typical octahedral shearing strain curves such as those presented in Figure 7-6 on page 136. Alternatively, a specific set of octahedral shearing strain curves may be developed for any sand by converting the strain amplitude from conventional degradation curves to octahedral shearing strain by use of Equations 2-42 or 2-44 as outlined in the text.

These two methods provide a reasonable estimate of the effects of interactive coupling on the degradation of modulus with strain. Both methods lose accuracy as the straining progresses from elastic to plastic, and as specimens approach failure. Nevertheless, the use of either method under conditions of combined shear and compression loading represents a significant improvement over the practice of neglecting interactive effects.

A series of large scale shaking table tests were conducted upon slope models in an effort to determine when yielding in the slopes began. The slope specimens were subjected to horizontal, and combined horizontal and vertical cyclic excitation. The slopes were thoroughly instrumented to record accelerations and displacements during loading

An analysis method was developed and presented for predicting yield accelerations in granular slopes under combined vertical and horizontal loading conditions. When the shaking table test results were compared with the predicted values of yield acceleration, it was concluded that the predictions were at least as accurate as the empirical measurements could be made. These results were consistent with the conclusion that the effects of combined vertical and horizontal accelerations on yielding of granular slopes may be evaluated using simple superposition.

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LIST OF SYMBOLS

Roman Letters

ABO' A'B'O	Area of triangles ABO and A'B'O
A _Q	Area of hysteresis loop
a, ACC1, ACC2	Variable of acceleration, accelerometers 1 and 2
a _h , a _v	Horizontal and vertical accelerations
amax' ao' al	Peak acceleration response at frequencies f_{res} , f_0 ,
	and f ₁ , respectively
a ₀ , a _n	Acceleration amplitude of 0.th and n.th cycles
b	Length of free body sliding block in slope
С	Notation indicating combined vertical and torsional
	loading
D	Ratio of critical damping
DCDT1, DCDT2	DC linear differential transformers 1 and 2
D _R	Relative density
đ	Depth of free body sliding block in slope
E, E _{ps}	Dynamic Young's Modulus and Dynamic Young's Modulus
	in plane strain
E <<γ _{θz}	Dynamic Young's Modulus at very low shear strain
e, e _{min} , e _{max}	Void ratio, minimum and maximum void ratios
F _d , F _u	Lower and upper normal forces acting on free body
	sliding block in slope
f _z , f _T	Vertical and torsional frequency of loading
f _{res} , f ₀ , f ₁ , f ₂	Resonant frequency and frequencies 0, 1, and 2
G	Dynamic shear modulus

G s	Unit weight of soil solids
g	Acceleration of gravity
н, н _о , н _и	Height of specimen, old height, and new height
н .	Notation indicating horizontal acceleration alone
1 ₁ , 1 ₂	First and second invariants of strain
J, J ₀	Torsional moment of inertia of specimen and top cap
	system
ky	Yield acceleration
L	Length or height of cylindrical specimen
l, l _{AB} , l _{BC}	Length, length from A to B, and length from B to C
М	Constrained compression modulus
N, N _w	Normal stress on failure surface at impending failure and
	normal stress due to weight of sliding block on slope
n	A positive integer
P	Normal force exerted by passive wedge at toe of slope
R	Radius of specimen
R	Resultant force in diagram of forces on free body at toe of slope
R _{AVG} , R ₁ , R ₂	Average, inner, and outer radius of hollow
R _O , R _N	Old and new radius of cylinder specimen
r	Variable in radial direction
s, s _w	Shear stress on failure surface at impending failure and
	shear stress due to weight of sliding block on slope
s _i	Shear strength intercept on Mohr's circle diagram
T	Notation indicating torsional loading alone
T	Upward force applied to sliding block on slope
T	Thickness of hollow cylinder specimen

Variable of time t Displacement in radial direction Notation indicating vertical loading alone v_0, v_N Old volume and new volume v_p, v_s Velocity of propagation of "p-wave" and "s-wave" v, v_v, v_s Volume, volume of voids, and volume of solids Displacement in tangential direction W, W_s Weight of soil $\mathbf{W}_{\mathbf{TOP}}$ Weight of top cap system Displacement in vertical direction Variable in "x" direction Variable in "y" direction Variable in "z" or vertical direction

z

Greek Letters

α	A constant
α	Phase lag between vertical and torsional loading
α	Angle of slope
β	Angle of reorientation of the principal stresses
$^{Y}_{d}$	Unit weight of dry soil
Y _{OCT}	Octahedral shearing strain
Yxy' Yyz' Yxz	Shear strains within the "xy", "yz", and "xz" planes
δ	Damping calibration factor
Δ	Amplitude decay damping
ΔΖ, Δθ	Peak vertical and torsional displacement
$\Delta \gamma_{R\theta}$, $\Delta \gamma_{Rz}$, $\Delta \gamma_{\theta z}$	Peak-to-peak or single amplitude shear strain in
	"R θ ", "Rz", and " θ z" planes
$\Delta \varepsilon_{\mathbf{R}}$, $\Delta \varepsilon_{\mathbf{\theta}}$, $\Delta \varepsilon_{\mathbf{z}}$	Peak-to-peak or single amplitude normal strain in "R",
	" θ ", and "z" directions
$\Delta\sigma_{\mathbf{R}}$, $\Delta\sigma_{\mathbf{\theta}}$, $\Delta\sigma_{\mathbf{z}}$	Peak-to-peak or single amplitude normal stress in
	the "R", " θ ", and "z" directions
Δσ _a , Δσ _a 1 max	Driving stress when acceleration response is a and
1 max	a max
$\Delta \tau_{R\theta}$, $\Delta \tau_{Rz}$, $\Delta \tau_{\theta z}$	Peak-to-peak or single amplitude shear stress in "R θ ",
	"Rz", and " θ z" planes
ε, ε _χ , ε _χ , ε _z	Normal strain and normal strain in the "x", "y", and
·	"z" directions
Ect	Center of Mohr's circle in strain
[€] oct	Octahedral normal strain
ε _{Rθ} , ε _{Rz} , ε _{θz}	Shear strains within "R0", "Rz", and " θ z" planes

ε _{vol}	Volumetric strain
ε ₁ , ε ₂ , ε ₃	Principal strains
ε _{1c'} ε _{2c'} ε _{3c}	Principal strains during consolidation
λ	A constant
λ	Hysteretic damping
π	The constant "π"
ρ	A constant
ρ	Mass density of specimen material
μ	Poisson's ratio
σ, σ _x , σ _y , σ _z ,	Normal stress and normal stress in the "x", "y", "z",
σ _R , σ _θ	"R", and " θ " directions
^o ct	Center of Mohr's circle in stress
σ a	Variable of principal stress
$\sigma_{\mathfrak{m}}$	Mean confining stress
σ ₁ , σ ₂ , σ ₃	Principal stresses
σ _{1c} , σ _{2c} , σ _{3c}	Principal stresses during consolidation
φ, φ _{ps}	Angle of internal friction, angle of internal friction
	in plane strain
$^{\Phi}_{ exttt{max}}$	Maximum angle of internal friction
$^{\phi}$ sl	Factor in simplified yield acceleration equation
τ _{xy} , τ _{xz} , τ _{yz}	Shear stress in the "xy", "xz", and "yz" planes
τ _{θz}	Shear stress in the " θz " plane
θ	A constant
θ	Angle of passive wedge at toe of slope
ω	Angle of application of resultant acceleration
ω _z , ω _T	Rotational velocity in vertical and torsional directions

Angle of reorientation of the principal strains

Chapter 1

Introduction

The objective of this research was to study the interaction effects of combined compression and shear loading on the response of sands to dynamic loading. This research included the following experimental studies:

- Resonant Column tests on cylindrically shaped specimens of sand using longitudinal and torsional excitation, separately and in combination (Chapter 5).
- Thin-walled Hollow Cylinder tests on sand specimens using longitudinal and torsional cyclic excitation, both separately and in combination (Chapter 6).
- 3. Large scale shaking table tests on slope models constructed with sand, using horizontal and vertical cyclic excitation, both separately and in combination (Chapter 8).

A review of available theories for analyzing soil behavior under dynamic loading conditions was made, and the strain tensor (Chapter 2) and state of stresses (Chapter 3) were developed for the specimens under test in the first two of these studies. Formulae for calculating moduli and damping factors from the results of those studies were developed and presented in Chapter 4. The results of these first two studies were further analyzed and combined in Chapter 7.

The details of the large scale shaking table testing program, including the development of an analysis technique and evaluation of the test results, were presented in Chapter 8.

The mechanical and testing details of the various equipment used in

these experimental studies are presented in Appendix A. Derivations

(Appendix C), computer programs (Appendix D), and example test results

(Appendix B) are also included.

These experimental studies were performed on specimens constructed of Monterey No. 0 sand, a uniformly graded, fine grained quartz sand processed from beach sand. A gradation analysis of the sand used in these studies is shown in Figure 1-1. The maximum and minimum densities of this sand were found to be as follows:

$$\gamma_{\rm d \ max} = 1.707 \ \rm gm/cc,$$
 (1.1)

and
$$\gamma_{d \min} = 1.425 \text{ gm/cc},$$
 (1.2)

Experimental studies were performed at a variety of densities and confining pressures so that the effects of these variables could be evaluated as well.

Chapter 1

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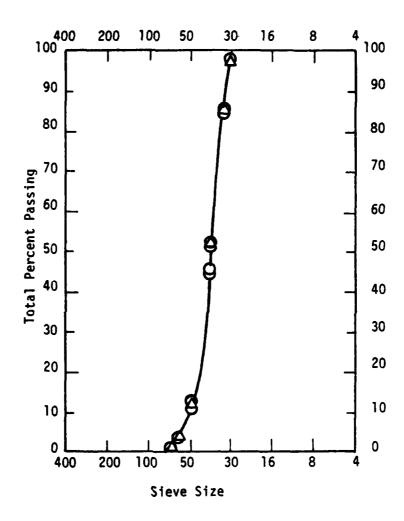


FIGURE 1-1 GRADATION ANALYSIS OF MONTEREY NO. O SAND USED IN THIS STUDY

Chapter 2

The Strain Tensor

Introduction

In both the triaxial resonant-column and the thin-walled hollow cylinder testing series forces and displacements are measured in the vertical and torsional directions. However, because of the geometry and complex loading of these specimens, the principal stresses and strains are not directly measurable. It is the purpose of this chapter to describe the strain tensor for these two testing series so that the complete state of strains may be determined from those strains which are directly measurable.

During the two testing series, the maximum strains measured were in the order of 10⁻¹%, so that a linear distribution of strains may be assumed. An elastic analysis of the stress-strain response of these soil specimens may be performed with the use of Hooke's Law.

Hooke's Law

In its simplest form, Hooke's Law for a homogeneous, isotropic body may be expressed as follows:

$$\begin{cases} \varepsilon_{\mathbf{x}} \\ \varepsilon_{\mathbf{y}} \\ \varepsilon_{\mathbf{z}} \\ \gamma_{\mathbf{yz}} \\ \gamma_{\mathbf{xz}} \\ \gamma_{\mathbf{xy}} \end{cases} = \begin{bmatrix} \frac{1}{E} & -\frac{\mu}{E} & 0 & 0 & 0 & 0 \\ -\frac{\mu}{E} & \frac{1}{E} & -\frac{\mu}{E} & 0 & 0 & 0 & 0 \\ -\frac{\mu}{E} & -\frac{\mu}{E} & \frac{1}{E} & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{2(1+\mu)}{E} & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{2(1+\mu)}{E} & 0 & \tau_{\mathbf{yz}} \\ 0 & 0 & 0 & 0 & 0 & \frac{2(1+\mu)}{E} & \tau_{\mathbf{xz}} \\ \end{cases}$$

$$(2.1)$$

The use of Hooke's Law for modeling the stress-strain behavior of a material implies several assumptions. The material is assumed to behave linearly and elastically during loading, so that the strains considered must be small enough so that changes in shape and size are negligible. Hooke's Law further implies that superposition of loading effects is valid, and that the principal stresses and strains are in the same directions.

Hooke's Law in Plane Strain

For the special case of plane strain in the xy plane, Hooke's Law reduces to the following expressions.

$$\begin{array}{cccc}
\varepsilon_{z} & \approx & 0 \\
\gamma_{xz} & \approx & 0
\end{array}$$

$$\gamma_{yz} & \approx & 0$$
(2.2)

 σ_z can be expressed as:

$$\sigma_{\mathbf{z}} = \mu + (\sigma_{\mathbf{x}} + \sigma_{\mathbf{y}}) \tag{2.3}$$

And:

$$\begin{cases}
\varepsilon_{\mathbf{x}} \\
\varepsilon_{\mathbf{y}} \\
\gamma_{\mathbf{x}\mathbf{y}}
\end{cases} = \begin{bmatrix}
\frac{1}{E}(1-\mu^2) & -\frac{1}{E}(\mu+\mu^2) & 0 \\
-\frac{1}{E}(\mu+\mu^2) & \frac{1}{E}(1-\mu^2) & 0 \\
0 & 0 & \frac{2(1+\mu)}{E}
\end{bmatrix} \begin{cases}
\sigma_{\mathbf{x}} \\
\sigma_{\mathbf{y}} \\
\tau_{\mathbf{x}\mathbf{y}}
\end{cases} (2.4)$$

Resonant-Column Testing

The strain tensor for the triaxial resonant-column testing series may be expressed in "cylindrical" coordinates in the following form:

$$\varepsilon = \begin{bmatrix} \varepsilon_{R} & \varepsilon_{R\theta} & \varepsilon_{Rz} \\ \varepsilon_{\theta R} & \varepsilon_{\theta} & \varepsilon_{\theta z} \\ \varepsilon_{zR} & \varepsilon_{z\theta} & \varepsilon_{z} \end{bmatrix}$$
 (2.5)

where the R, θ , and z directions are as shown in Figure 2-1(c) and $\varepsilon_{\theta z}$ = $\gamma_{\theta z}$, $\varepsilon_{R\theta}$ = $\gamma_{R\theta}$, and ε_{zR} = γ_{zR} . To maintain equilibrium, we can see from Figure 2-1(b) that $|\varepsilon_{R\theta}| = |\varepsilon_{\theta R}|$, $|\varepsilon_{Rz}| = |\varepsilon_{zR}|$, and $|\varepsilon_{z\theta}| = |\varepsilon_{\theta z}|$; and so [ε] must be a symmetrical matrix which is uniquely defined by six strains.

Strain Equations

If the displacements u, v, and w are defined as shown in Figure 2-1(c), the strains can be defined as follows:

$$\varepsilon_{\rm R} = \frac{\rm du}{\rm dr}$$
 (2.6)

$$\varepsilon_{\theta} = \frac{u}{r} + \frac{dv}{d\theta} \tag{2.7}$$

$$\varepsilon_{z} = \frac{\mathrm{d}w}{\mathrm{d}z} \tag{2.8}$$

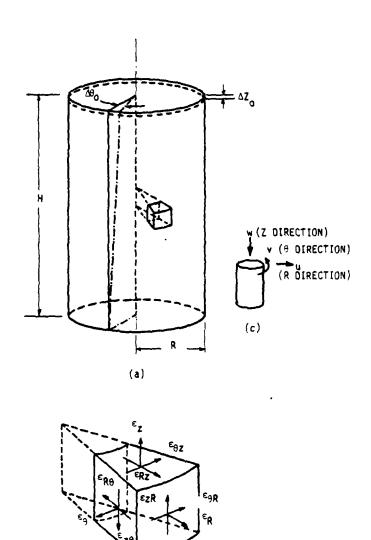


FIGURE 2-1 FREE BODY DIAGRAM AND STATE OF STRAINS FOR RESONANT COLUMN SPECIMENS

(b)

$$\varepsilon_{R\theta} = \varepsilon_{\theta R} = \frac{1}{2} (\frac{1}{r} \frac{du}{d\theta} + \frac{dv}{dr} - \frac{v}{r})$$
 (2.9)

$$\varepsilon_{z\theta} = \varepsilon_{\theta z} = \frac{1}{2} (\frac{dv}{dz} + \frac{1}{r} \frac{dw}{d\theta})$$
 (2.10)

$$\varepsilon_{Rz} = \varepsilon_{zR} = \frac{1}{2} (\frac{du}{dz} + \frac{dw}{dr})$$
 (2.11)

For this testing series, the following boundary conditions exist:

1.
$$w = \alpha \cdot z$$
, $\alpha = \varepsilon_{\overline{z}}$ (2.12)

2.
$$\mathbf{v} = \mathbf{r} \cdot \mathbf{z} \cdot \Delta \theta_0$$
 (2.13)

Equation 2.12, states that the vertical strain is known. This is true because it is directly calculated in all of the tests. The assumption expressed in Equation 2.13 is that the torsional displacement v varies linearly in both the r and z directions. This condition is illustrated in Figure 2-1(a).

· Strain Tensor

Applying boundary conditions 1 and 2 (Equations 2.12 and 2.13) to the six strains which define the strain tensor (Equations 2.6 through 2.11) yields the following:

$$\varepsilon = \begin{bmatrix} -\mu \varepsilon_{z} & 0 & 0 \\ 0 & -\mu \varepsilon_{z} & \frac{1}{2} r \Delta \theta_{0} \\ 0 & \frac{1}{2} r \Delta \theta_{0} & \varepsilon_{z} \end{bmatrix}$$
 (2.14)

A detailed derivation of the above expression appears in Appendix C-1.

Hollow Cylinder Testing

In the case of the thin-walled hollow cylinder testing series the strain tensor may be expressed in the same form as Equation 2.5, but the strains are now as shown in Figure 2-2.

Strain Equations

As with the resonant-column testing series, $\{\epsilon\}$ is a symmetric matrix about the normal strain diagonal, and is uniquely defined by six strains. From Figures 2-2 and 2-3 those strains can be defined as follows:

$$\varepsilon_{z} = \frac{\mathrm{d}\mathbf{w}}{\mathrm{d}z} \tag{2.8}$$

$$\varepsilon_{\theta} = \left[\frac{(1 - R_{AVG}/r)}{(1 - R_{AVG}/R_2)} \cdot \left(\frac{R_{2NEW}}{R_{2OLD}} - 1 \right) \right]_{R_{AVG} \le r \le R_2}$$
(2.15)

$$\epsilon_{\theta}|_{R_{AVG} \leq r \leq R_2} = -\epsilon_{\theta}|_{R_1 \leq r \leq R_{AVG}}$$

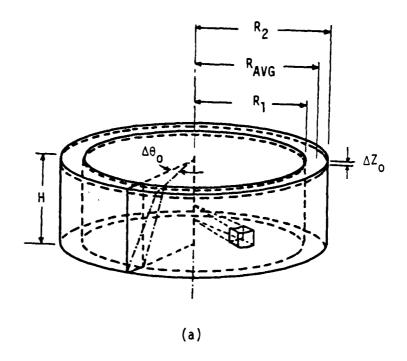
$$\varepsilon_{R} = \left[\frac{R_{2NEW} - R_{AVG}}{R_{2OLD} - R_{AVG}} - 1\right]$$

$$R_{1} \le r \le R_{2}$$
(2.16)

$$\varepsilon_{R\theta} = \frac{1}{2} \left(\frac{1}{r} \frac{du}{d\theta} + \frac{dv}{dr} - \frac{v}{r} \right) \tag{2.9}$$

$$\varepsilon_{z\theta} = \frac{1}{2} (\frac{dv}{dz} + \frac{dw}{d\theta}) \tag{2.10}$$

$$\varepsilon_{Rz} = \frac{1}{2} \left(\frac{du}{dz} + \frac{dw}{dr} \right) \tag{2.11}$$



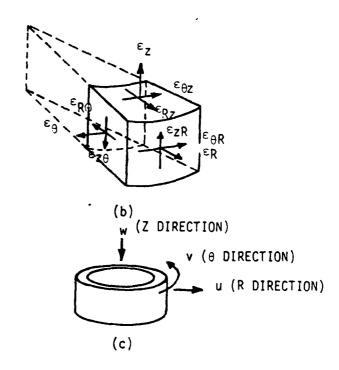
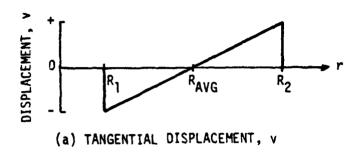


FIGURE 2-2 FREE BODY DIAGRAM AND STATE OF STRAINS FOR HOLLOW CYLINDER SPECIMENS



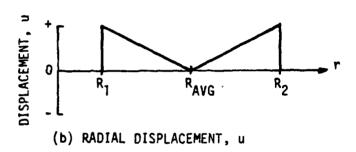


FIGURE 2-3 TANGENTIAL (a) AND RADIAL (b) DISPLACEMENT OF HOLLOW CYLINDER SPECIMENS UNDER LOAD AS A FUNCTION OF INITIAL RADIUS

Tangential Strain Assumption

Note that Equations 2.8 through 2.11 are identical to the expressions developed for the resonant-column testing series. The assumption implicit in Equations 2.15 and 2.16 and in Figure 2-3 is that the thin-walled hollow cylinder specimen will strain in such a way that the average radius, R_{AVG} , remains unchanged during loading. This assumption is equivalent to the assumption that ε_{θ} = 0. If $\gamma_{R\theta}$ = $\gamma_{Z\theta}$ = 0, such as during consolidation or vertical loading alone, a plane strain condition exists.

Because of the geometry and the end platen boundary conditions of the thin-walled hollow cylinder specimen, the assumption of ϵ_{θ} = 0 is reasonable. As the specimen is loaded compressively in the vertical direction, the specimen will compress vertically and bulge radially. This radial straining will appear as a bulging on both the inner and outer walls of the cylinder. At some radius within the specimen the radial (and tangential) displacement will be zero. If that radius is the average radius, $R_{\rm AVG}$, then the net tangential strain will be zero, as illustrated in Figure 2-3(a). If the no-strain radius is some other value, the net tangential strain will not be zero, but will be very small when compared with the net radial strain. Marachi et al, (1969) have shown that under this type of net strain condition the stress-strain response is essentially that of plane strain.

For this testing series the following boundary conditions exist:

1.
$$w = \alpha \cdot z$$
, $\alpha = \varepsilon_z$ (2.12)

2.
$$v = r \cdot z \cdot \Delta \theta_0$$

Note that these are identical to the boundary conditions for the triaxial resonant-column testing series.

Strain Tensor

Applying these boundary conditions to the six strain equations defining the strain tensor (Equations 2.8 through 2.11, 2.15, and 2.16) gives the following expression for the strain tensor:

$$\varepsilon = \begin{bmatrix} -\left(\frac{\mu + \mu^{2}}{1 - \mu^{2}}\right) \varepsilon_{z} & 0 & 0 \\ 0 & 0 & \frac{1}{2} R_{AVG}^{\Delta \theta} 0 \\ 0 & \frac{1}{2} R_{AVG}^{\Delta \theta} 0 & \varepsilon_{z} \end{bmatrix}$$
 (2.17)

A detailed derivation of the above expression appears in Appendix C-2. It was assumed that ϵ_A = 0 for this derivation.

Incremental Principal Strains

Incremental principal strains may be evaluated using the strain tensors developed in Equations 2.14 and 2.17.

Resonant Column

In the case of the triaxial resonant column specimens, whose strain tensor is given in Equation 2.14, the Mohr's circle in strain for the θz "plane" is shown in Figure 2-4(a). The condition illustrated in this figure is the case in which the maximum vertical normal strain, $\frac{\Delta \epsilon_z}{2}$, is occurring simultaneously with the average shear strain, $\frac{\Delta \gamma_{\theta z}}{4}$. As recorded, $\Delta \gamma_{\theta z}$ and $\Delta \epsilon_z$ are "peak-to-peak" or double amplitude values. Thus the single amplitude values are $\Delta \gamma_{\theta z}/2$ and $\Delta \epsilon_z/2$. Because the compressional and torsional strains were out of phase and at different frequencies, the average value of $\Delta \gamma_{\theta z}$ occurring simultaneously with $\Delta \epsilon_z$

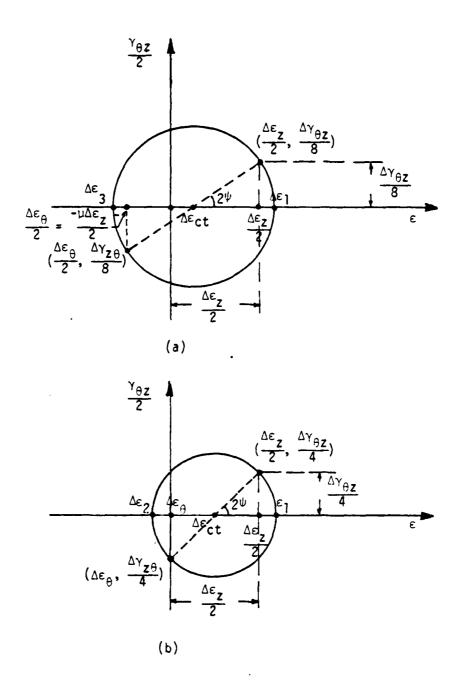


FIGURE 2-4 MOHR'S CIRCLE IN STRAIN FOR THE "0z PLANE" FOR RESONANT COLUMN (a) AND HOLLOW CYLINDER (b) SPECIMENS UNDER DYNAMIC LOADING CONDITIONS

was approximated as $\Delta\gamma_{\theta z}/2$. Thus $\Delta\gamma_{\theta z}/4$ is used as the average single amplitude value. Because the vertical axis of Mohr's circle is equal to half the shear strain in any direction, the plotted value of shear strain on the θz "plane" is $\frac{\Delta\gamma_{\theta z}}{8}$.

From Figure 2-4(a), and the strain tensor (Equation 2.14) the principal strains may be expressed as follows:

$$\Delta \varepsilon_1 = \frac{\Delta \varepsilon_z}{2} + \frac{\Delta \gamma_{\theta z}}{8} \cdot \tan \psi \qquad (2.20)$$

$$\Delta \varepsilon_2 = \frac{\Delta \varepsilon_R}{2} = -\mu \frac{\Delta \varepsilon_z}{2} \tag{2.21}$$

$$\Delta \varepsilon_3 = -(\mu \frac{\Delta \varepsilon_z}{2} + \frac{\Delta \gamma_{\theta z}}{8} \cdot \tan \psi) \qquad (2.22)$$

where, $\psi = \begin{cases} \frac{1}{2} \tan^{-1} & \frac{\Delta \gamma_{\theta z}}{2} \\ \frac{1}{2} \tan^{-1} & \frac{\Delta \gamma_{\theta z}}{8(\frac{z}{2} - \Delta \epsilon_{ct})} \end{cases}$ (2.23)

and, $\Delta \varepsilon_{ct} = \left(\frac{1-\mu}{4}\right) \Delta \varepsilon_{z}$ (2.24)

Hollow Cylinder

For the case of the thin-walled hollow cylinder testing series, whose strain tensor is presented in Equation 2.17, the Mohr's circle of strain for the θz "plane" is shown in Figure 2-4(b). In this figure the condition of maximum vertical normal strain and simultaneous maximum shear strain is illustrated. In the hollow cylinder series the measured value of $\Delta \gamma_{\theta z}$ is the double amplitude value, so $\Delta \gamma_{\theta z}/2$ is the single amplitude value. However, the compressional and torsional strains had

the same frequency. If they were also in phase, as depicted in Figure 2-4(b), then the maximum value of $\Delta \gamma_{\theta Z}$ would always occur when $\Delta \epsilon_{Z}$ is maximum. Thus it is not necessary to divide $\Delta \gamma_{\theta Z}$ by an additional factor of 2 to get the average value of the shear strain accompanying the maximum value of vertical normal strain, as was done for the resonant column specimens.

Because the vertical axis of the Mohr's circle in strain is equal to one-half the shear strain in any direction, the plotted value of shear strain in the θz "plane" in Figure 2.4(b) is $\frac{\Delta \gamma_{\theta z}}{4}$. From the figure the following equations may be written:

$$\Delta \varepsilon_1 = \frac{\Delta \varepsilon_z}{2} + \frac{\Delta \gamma_{\theta z}}{4} \cdot \tan \psi \qquad (2.25)$$

$$\Delta \varepsilon_2 = -\frac{\Delta \gamma_{\theta z}}{4} \cdot \tan \psi \qquad (2.26)$$

$$\Delta \varepsilon_3 = -\left[\frac{\mu + \mu^2}{1 - \mu^2}\right] \frac{\Delta \varepsilon_z}{2} \tag{2.27}$$

and,

$$\psi = \frac{1}{2} \tan^{-1} \left[\frac{\Delta \gamma_{\theta z}}{4 \left(\frac{z}{2} - \Delta \epsilon_{ct} \right)} \right]$$
 (2.28)

$$\Delta \varepsilon_{CL} = \frac{\Delta \varepsilon_{Z}}{4} \tag{2.29}$$

Non-Combined Loading

For the special case of vertical loading alone, Equations 2.20 through 2.22 reduce to the following:

$$\Delta \varepsilon_1 = \frac{\Delta \varepsilon_z}{2} \tag{2.30}$$

$$\Delta \varepsilon_2 = \Delta \varepsilon_3 = -\frac{\mu \Delta \varepsilon_z}{2} \tag{2.31}$$

and Equations 2.25 through 2.28 become:

$$\Delta \varepsilon_1 = \frac{\Delta \varepsilon_2}{2} \tag{2.32}$$

$$\Delta \varepsilon_2 = 0$$
 (2.33)

$$\Delta \varepsilon_3 = -\left[\frac{\mu + \mu^2}{1 - \mu^2}\right] \frac{\Delta \varepsilon_z}{2} \tag{2.34}$$

and $\psi = 0$ (2.35)

In the special loading case of torsional loading alone Equations 2.20 through 2.23 become:

$$\Delta \varepsilon_1 = \frac{\Delta \gamma_{\theta z}}{4} \tag{2.36}$$

$$\Delta \varepsilon_2 = 0$$
 (2.37)

$$\Delta \varepsilon_3 = -\frac{\Delta \gamma_{\theta z}}{4} \tag{2.38}$$

$$\psi = 45^{\circ} \tag{2.39}$$

In the absence of vertical loading it is not necessary to divide $\Delta\gamma_{\theta z} \text{ by two to get the average value associated with } \Delta\varepsilon_z. \text{ Thus the denominator in Equations 2.36 and 2.38 is 4 rather than 8.}$

Although torsional loading alone was not performed in the hollow cylinder test series, the applicable equations can be derived from Equations 2.25 through 2.29.

Because $\Delta \epsilon_z/2$ = 0, Equation 2.27 now defines $\Delta \epsilon_z$, which is zero, and

 $\Delta \epsilon_3$ is obtained from Equation 2.26. Thus the expressions for $\Delta \epsilon_1$, $\Delta \epsilon_2$, $\Delta \epsilon_3$ and ψ become identical to Equations 2.36 through 2.39.

Octahedral Strains

The nonlinearity of the stress-strain response for soil has been recognized for some time. Researchers have often referred to this non-linearity as the "degradation of modulus with increasing strain amplitude." The octahedral strains are a very useful tool in evaluating the effects of strain amplitude on modulus because they are both dimensionless and bring into consideration the complete state of strain, and are independent of the orientation of the coordinate system.

Octahedral Normal Strain

There are two octahedral strains: the hydrostatic component and the shearing component. The first of these, $\epsilon_{\rm OCT}$, is also called the octahedral strain or the octahedral normal strain; and may be computed from the following expression:

$$\varepsilon_{\text{OCT}} = \frac{I_1}{3} \tag{2.40}$$

where \mathbf{I}_{1} is the first strain invarient, and may be written:

$$I_1 = \varepsilon_x + \varepsilon_y + \varepsilon_z = \varepsilon_\theta + \varepsilon_R + \varepsilon_z = \varepsilon_1 + \varepsilon_2 + \varepsilon_3 = \varepsilon_{vol}$$
 (2.41)

Octahedral Shearing Strain

The second octahedral strain, $\gamma_{\rm OCT}$, is also called the octahedral shearing strain; and may be computed from the following:

$$\gamma_{\text{OCT}} = \frac{2\sqrt{2}}{3} \cdot \left[I_1^2 + 3I_2 \right]^{1/2}$$
 (2.42)

where \mathbf{I}_{2} is the second strain invarient, and may be written:

$$I_{2} = -(\varepsilon_{y}\varepsilon_{z} + \varepsilon_{z}\varepsilon_{x} + \varepsilon_{x}\varepsilon_{y}) + \frac{1}{4}\gamma_{yz}^{2} + \frac{1}{4}\gamma_{zx}^{2} + \frac{1}{4}\gamma_{xy}^{2}$$

$$= -(\varepsilon_{\theta}\varepsilon_{z} + \varepsilon_{z}\varepsilon_{R} + \varepsilon_{\theta}\varepsilon_{R}) + \frac{1}{4}\gamma_{\theta z}^{2} + \frac{1}{4}\gamma_{zR}^{2} + \frac{1}{4}\gamma_{\theta R}^{2}$$

$$= -(\varepsilon_{1}\varepsilon_{2} + \varepsilon_{2}\varepsilon_{3} + \varepsilon_{1}\varepsilon_{3}) \qquad (2.43)$$

In terms of the principal strains, Equation 2.42 may be written as follows:

$$\gamma_{\text{OCT}} = \frac{2}{3} \sqrt{(\varepsilon_1 - \varepsilon_3)^2 + (\varepsilon_1 - \varepsilon_2)^2 + (\varepsilon_2 - \varepsilon_3)^2}$$
 (2.44)

This shearing component has the effect of causing shearing or distorsional deformation of a material, whereas the octahedral normal strain has the effect of causing purely normal or compressive deformations.

When considering the process of degradation of modulus with increasing strain amplitude, it is clearly the shearing strain component of the state of strain which causes the degradation. The normal component of the strain tends to <u>increase</u> the modulus with increasing strain amplitude.

For soils assumed to be isotropic the octahedral shearing strain, $\gamma_{\rm OCT}$, would appear to be a very useful tool in evaluating the effects of strain on modulus for these testing series since it will bring into consideration both the effect of the dynamic shear strains, $\Delta\gamma_{\theta z}$, and the shear strain component of the anisotropic dynamic vertical loading, represented by the diameter of the Mohr's circle in strain in Figure 2-4.

Chapter 3

The State of Stresses

Introduction

It is the purpose of this chapter to describe the complete state of stresses of the various soil samples used in both the triaxial resonant-column and the thin-walled hollow cylinder testing series. This presentation will include the magnitude and direction of the principal stresses throughout the load history of the specimens.

Assuming that the material is behaving both linearly and elastically, the principal stresses and directions may be easily calculated if the values of all stresses on three mutually perpendicular planes are known. In the two testing series conducted, the stresses were measured in different manners and the specimens were subjected to different pre-loading consolidation stress histories; therefore, the state of stress will be discussed separately for each series.

Resonant-Column Testing

Test Procedure

In this testing series cylindrically shaped specimens of sand were isotropically consolidated to one of the following confining pressure: 0.5 KSC, 2.0 KSC, and 3.5 KSC (1 KSC = 1 kg/cm² = 98.07 kN/m²). The samples were then fixed at the base and excited from the cap with a pure sinusoidal force of very low amplitude. The frequency of this excitation was adjusted to the undamped natural frequency (also called the resonant frequency) of the soil column. The acceleration response of the

soil column was measured with a calibrated accelerometer mounted in the cap of the specimen.

The resonant frequency was obtained with the use of an oscilloscope by displaying a Lissajous figure of the excitation and the acceleration response. When the two axes of the elliptical Lissajous figure are in the same directions as the input axes (i.e.: the vertical and horizontal plates), then the two signals are at the same frequency but 90° out of phase. For the resonant-column sample, the frequency at which the input excitation and the acceleration response are 90° out of phase is by definition the undamped natural frequency.

From the calibrated acceleration response and the geometry of the specimen, the peak-to-peak strain response may be easily calculated.

During testing of a specimen the excitation may be axial (vertical), torsional, or both simultaneously. In a typical test the specimen will be loaded with a particular excitation for less than two minutes, then the excitation amplitude is increased and the process repeated.

Values of dynamic unconstrained compression modulus (also called dynamic Young's modulus) and dynamic shear modulus are calculated from the resonant frequency, the sample geometry and weight and the dynamic response characteristics of the testing apparatus. A more detailed discussion of this calculation is included in Appendix C-3.

Stress Equations

The net, peak-to-peak vertical and torsional stress may be calculated by multiplying the calculated modulus by the calculated peak-to-peak strain as follows:

$$\Delta\sigma_{z} = E \cdot \Delta\varepsilon_{z} \tag{3.1}$$

and
$$\Delta \tau_{z\theta} = G \cdot \Delta \gamma_{z\theta}$$
 (3.2)

where:

E = unconstrained compression modulus,

G = shear modulus,

 $\Delta \sigma_z$ = peak-to-peak vertical stress,

 $\Delta \varepsilon_{\tau}$ = peak-to-peak vertical strain,

 $\Delta \tau_{z\theta}$ = peak-to-peak torsional stress,

and $\Delta \gamma_{z\theta}$ = peak-to-peak torsional strain.

Vertical Loading Alone

The typical loading sequence for the case of vertical loading alone is shown in Figure 3-1. In this figure it can be seen that the calculation of the magnitude and direction of the principal stresses is greatly simplified. The principal stress directions are coincident with the directions of the cylindrical coordinate system shown in Figure 2-1; thus their values are equal to the values of σ_z , σ_R , and σ_θ as a function of time, where,

 σ_z = vertical normal stress,

 $\sigma_{\rm p}$ = radial normal stress,

 σ_A = tangential normal stress.

The mathematical expression for the vertical principal stress, ${}^{\sigma}{}_{a}$, is as follows:

$$\sigma_{\mathbf{a}} = \sigma_{1c} + \frac{\Delta \sigma_{\mathbf{z}}}{2} \cdot \sin(\omega_{\mathbf{z}} \cdot \mathbf{t})$$
 (3.3)

where σ_{lc} = vertical principal stress during consolidation

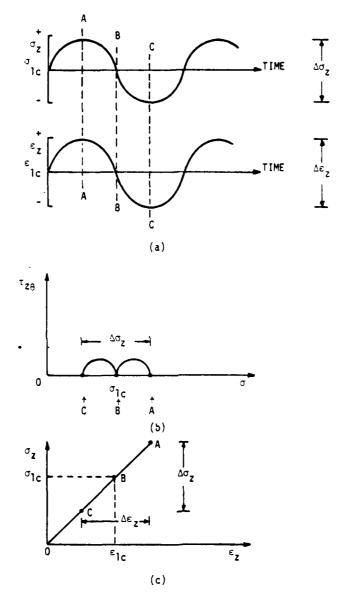


FIGURE 3-1 TYPICAL DYNAMIC LOADING SEQUENCE FOR RESONANT COLUMN SPECIMENS UNDER CONDITION OF VERTICAL LOADING ALONE: TIME HISTORIES OF STRESS AND STRAIN (a), MOHR'S CIRCLE DIAGRAM IN STRESS (b), AND STRESS VS STRAIN CURVE (c)

and
$$\omega_z = 2 \cdot \pi \cdot f_z$$
 (3.4)

with f_{z} as the vertical resonant frequency.

If a free-body is isolated as was done in Figure 2-1, and the maximum vertical stress condition is plotted upon it, the illustration in Figure 3-2(a) is obtained. Figure 3-2(b) shows the Mohr's circle plot for this loading condition. Note from this figure that:

$$\sigma_1 = \sigma_z = \sigma_R + \frac{\Delta \sigma_z}{2}$$
 (3.5)

$$\sigma_2 = \sigma_3 = \sigma_R = \sigma_\theta \tag{3.6}$$

where,

 σ_1 = major principal stress,

 σ_2 = intermediate principal stress,

 σ_3 = minor principal stress.

Torsional Loading Alone

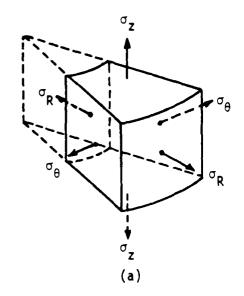
Under the condition of torsional stress alone, the maximum principal stress Mohr's circle is shown in Figure 3-3(a). With these loading conditions, the principal stresses in the θz "plane" are oriented 45° from the axes in cylindrical coordinates. These principal stress values may be expressed as follows:

$$\sigma_1 = \sigma_{1c} + \left[\frac{\Delta \tau_{\theta z}}{2} \cdot \sin(\omega_T \cdot t) \right]$$
 (3.7)

$$\sigma_2 = \sigma_{2c} = \sigma_{R} \tag{3.8}$$

$$\sigma_3 = \sigma_{3c} - \left[\frac{\Delta \tau_{\theta z}}{2} \cdot \sin(\omega_T \cdot t)\right]$$
 (3.9)

where $\sigma_{\rm lc},~\sigma_{\rm 2c},$ and $\sigma_{\rm 3c}$ are the principal stresses during consolidation.



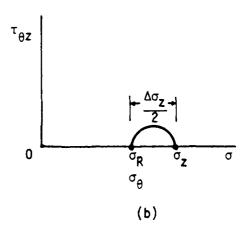


FIGURE 3-2 FREE BODY DIAGRAM (a) AND MOHR'S CIRCLE IN STRESS (b) FOR RESONANT COLUMN SPECIMENS UNDER CONDITION OF MAXIMUM INSTANTANEOUS VERTICAL STRESS

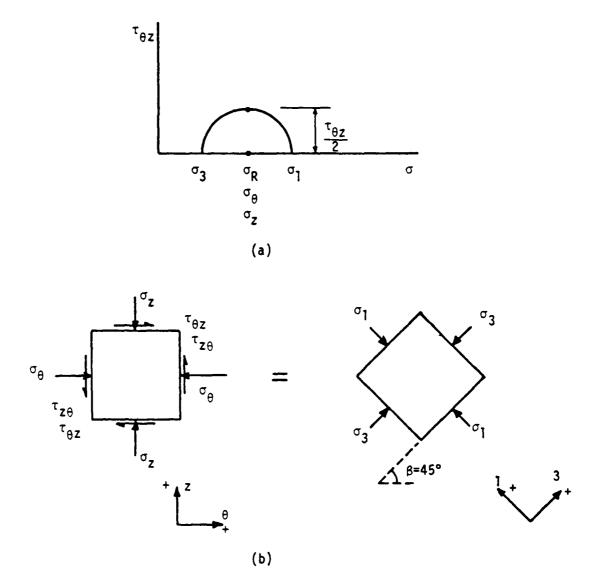


FIGURE 3-3 MOHR'S CIRCLE IN STRESS (a) AND TWO-DIMENSIONAL FREE BODY DIAGRAM (b) FOR RESONANT COLUMN SPECIMENS UNDER CONDITION OF MAXIMUM INSTANTANEOUS TORSIONAL STRESS

and
$$\omega_{\mathbf{r}} = 2 \cdot \pi \cdot \mathbf{f}_{\mathbf{r}}$$
 (3.10)

with f_T as the torsional resonant frequency. The relationship between the applied torsional stress and the principal stresses is further illustrated in Figure 3-3(b), which is a two-dimensional free-body diagram in the θz "plane".

Combined Loading

In the case of simultaneous vertical and torsional excitation conditions become more complicated. The resonant frequencies f_z and f_T are different, and are normally not an integer multiple of one another. As seen above, the principal stress directions are different for the two types of loading, and will vary as a function of time. Figure 3-4 shows a Mohr's circle diagram in the θz "plane" for the condition (and at the time of occurence) of the maximum σ_1 . The angle β is the angle of reorientation of the principal stresses in the θz "plane" from the θz axes.

If we establish a constant, ρ , such that:

$$\rho = \frac{f_T}{f_z} \tag{3.11}$$

the stresses may be expressed in the following mathematical form:

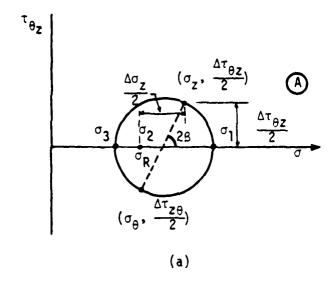
$$\sigma_{z} = \sigma_{R} + \left[\frac{\Delta \sigma_{z}}{2} \cdot \sin(\omega_{z} \cdot t)\right]$$
 (3.12)

and

$$\tau_{\theta z} = \frac{\Delta \tau_{\theta z}}{2} \cdot \sin(\rho \cdot \omega_{z} \cdot t)$$
 (3.13)

From these equations the following expressions were derived for the principal stresses and directions:

$$\sigma_1 = \sigma_R + \left[\frac{\Delta \sigma_z}{2} \cdot \sin(\omega_z \cdot t)\right] + \left[\frac{\Delta \tau_{\theta z}}{2} \cdot \sin(\rho \cdot \omega_z \cdot t) \cdot \tan(\beta)\right]$$
 (3.14)



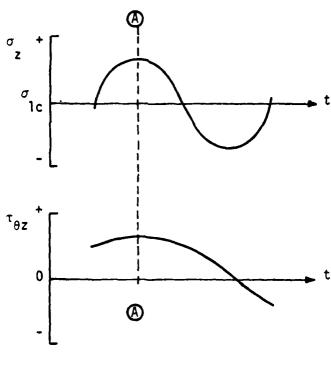


FIGURE 3-4 MOHR'S CIRCLE IN STRESS (a) AND STRESS TIME HISTORIES (b) FOR SIMULTANEOUS DYNAMIC LOADING OF RESONANT COLUMN SPECIMENS UNDER CONDITION OF MAXIMUM INSTANTANEOUS VERTICAL AND TORSIONAL STRESS

(b)

$$\sigma_2 = \sigma_R \tag{3.15}$$

$$\sigma_3 = 2\sigma_{ct} - \sigma_1 \tag{3.16}$$

$$\beta = \frac{1}{2} \tan^{-1} \left[\frac{\Delta \tau_{\theta z} \cdot \sin(\rho \cdot \omega_{z} \cdot t)}{\frac{\Delta \sigma_{z}}{2} \cdot \sin(\omega_{z} \cdot t)} \right]$$
(3.17)

and

$$\sigma_{\text{ct}} = \sigma_{\text{R}} + \left[\frac{\Delta \sigma_{z}}{4} \cdot \sin(\omega_{z} \cdot t)\right]$$
 (3.18)

where $\sigma_{\mbox{\scriptsize ct}}$ is the center of the θz Mohr's circle.

A more detailed derivation of the above expressions appears in Appendix C-4.

Hollow Cylinder Testing

Sample Preparation

In these testing series thin walled hollow cylinder specimens of sand were anisotropically consolidated with a principal stress ratio of 0.54. The minor principal stress during consolidation was set at one of the following values: 0.5 KSC, 2.0 KSC, or 3.5 KSC.

Consolidation Stress Equations

As discussed in Chapter 2, the load response of the hollow cylinder specimen is closely analogous to plane strain type loading in the zR plane during consolidation. If we apply the expressions for Hooke's Law in the special case of plane strain loading (Equations 2.2 through 2.4) to the present example, we obtain the following expression for the intermediate principal stress during consolidation:

$$\sigma_{2c} = \mu(\sigma_{1c} + \sigma_{3c}) \tag{3.19}$$

where:

 σ_{lc} = major principal stress during consolidation

 σ_{2c} = intermediate principal stress during consolidation

 σ_{3c} = minor principal stress during consolidation

and μ = Poisson's ratio

Because the directions of the principal stresses are coincident with the directions of the axes in cylindrical coordinates, and because the principal stress ratio during consolidation is known, the consolidation stress state for the hollow cylinder test may be stated as follows:

$$\sigma_{1c} = \sigma_{z} = 1.85 \cdot \sigma_{3c} \tag{3.20}$$

$$\sigma_{2c} = \sigma_{\theta} = 2.85 \cdot \mu \cdot \sigma_{3c}$$
 (3.21)

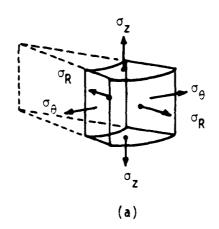
$$\sigma_{3c} = \sigma_{R} \tag{3.22}$$

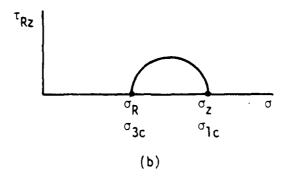
It is noteworthy that, should the Poisson's ratio be less than 0.35, then the tangential principal stress would actually be the minor principal stress and the radial stress would be the intermediate principal stress. This consolidation stress state is illustrated graphically in Figure 3-5.

Test Procedure

Following consolidation, the hollow cylinder specimens were excited at a frequency of approximately 0.33 hertz with a moderate amplitude sinusoidal force. The loading excitation was applied in the vertical direction, the torsional direction, or in both the vertical and torsional directions simultaneously.

During the excitation, which was applied for 10 to 20 cycles, the





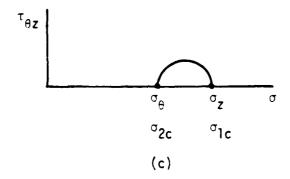


FIGURE 3-5 FREE BODY DIAGRAM (a) AND MOHR'S CIRCLES IN STRESS (b AND c) FOR HOLLOW CYLINDER SPECIMENS UNDER CONDITION OF ANISOTROPIC CONSOLIDATION

vertical stress and strain were recorded directly on an XY recorder, and the vertical stress, torsional stress, and torsional strain were plotted as a function of time on a strip chart recorder. Several typical load records are shown in Figure 3-6. After loading, the cyclic stress levels were adjusted, and the process repeated.

Vertical Loading Alone

The typical loading sequence for the case of vertical excitation alone is illustrated in Figure 3-7(a). In this figure we can observe several differences from the case of the resonant-column testing series that was shown in Figure 3-1. The peak stresses and strains occur essentially at the same time, but the zero crossings indicate a phase lag of strain behind stress. This is further illustrated in Figure 3-7(b), as a hysteretical stress-strain response. This hysteresis represents an energy loss during loading and is a measure of damping during the loading sequence. Although some non-elastic response occurs in all loading, it is relatively insignificant at the very low levels of loading encountered during the resonant-column testing series. At the higher strains seen in the hollow cylinder testing series, however, hysteretic stress-strain response is common. This damping effect will be discussed in more detail in Chapter 4.

A second peculiarity of this testing series is the anisotropic consolidation which the samples have undergone. This has the primary effect of moving the dynamic loading effects higher up the stress-strain response curve to a region where higher total strains (and the resultant non-linear, inelastic load response) are seen. In this region of the

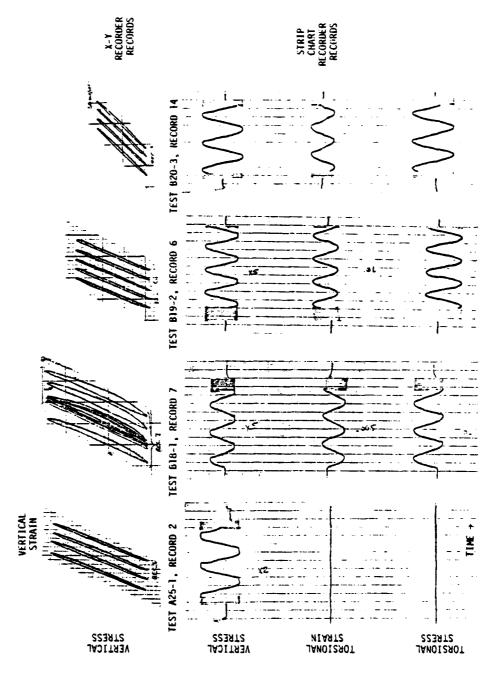
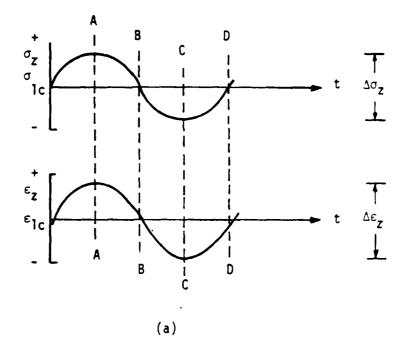


FIGURE 3-6 TEST RECHROS FOR A TYPICAL HOLLOW CYLINDER TESTS



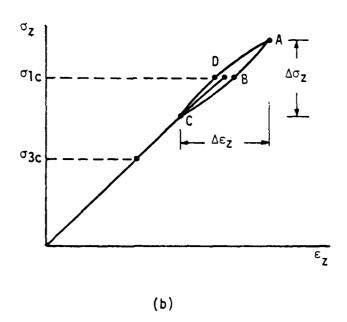


FIGURE 3-7 TYPICAL LOADING SEQUENCE FOR HOLLOW CYLINDER SPECIMENS UNDER CONDITION OF VERTICAL LOADING ALONE: TIME HISTORIES OF STRESS AND STRAIN (a) AND STRESS VS STRAIN CURVE (b)

material response, a significant level of load resisting stress has been mobilized in the specimen.

In addition, because of the nearly plane strain type load response of the hollow cylinder specimen, the tangential normal stress varies with the vertical normal stress and the Poisson's ratio. This effect is illustrated schematically in a Mohr's circle diagram in the θz "plane" in Figure 3-8.

The mathematical expressions for the principal stresses under this loading condition are as follows:

$$\sigma_{z} = \sigma_{1c} + \left[\frac{\Delta \sigma_{z}}{2} \cdot \sin(\lambda \cdot t)\right]$$
 (3.23)

$$\sigma_{\theta} = \sigma_{2c} + \left[\frac{\Delta \sigma_{\theta}}{2} \cdot \sin(\lambda \cdot t)\right]$$
 (3.24)

and
$$\sigma_{R} = \sigma_{3c}$$
 (3.25)

where λ is a constant. These equations further reduce to the following:

$$\sigma_{z} = 1.85 \cdot \sigma_{3c} + \left[\frac{\Delta \sigma_{z}}{2} \cdot \sin(\lambda \cdot t) \right]$$
 (3.26)

$$\sigma_{\theta} = 2.85 \cdot \mu \cdot \sigma_{3c} + \left[\mu \cdot \frac{\Delta \sigma_{z}}{2} \cdot \sin(\lambda \cdot t) \right]$$
 (3.27)

and
$$\sigma_{R} = \sigma_{3c}$$
. (3.25)

Torsional Loading Alone

Under the condition of torsional stress alone, the maximum principal stress Mohr's circle in the θz plane is shown in Figure 3-9(a). Unlike the case with the resonant-column testing series, the principal stresses in the " θz " plane are not oriented at an angle of 45° from the cylindrical coordinate axes. Indeed, the angle of orientation, β , varies both

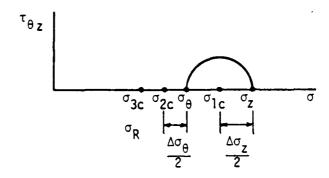


FIGURE 3-8 MOHR'S CIRCLE IN STRESS FOR HOLLOW CYLINDER SPECIMENS UNDER CONDITION OF MAXIMUM INSTANTANEOUS VERTICAL STRESS

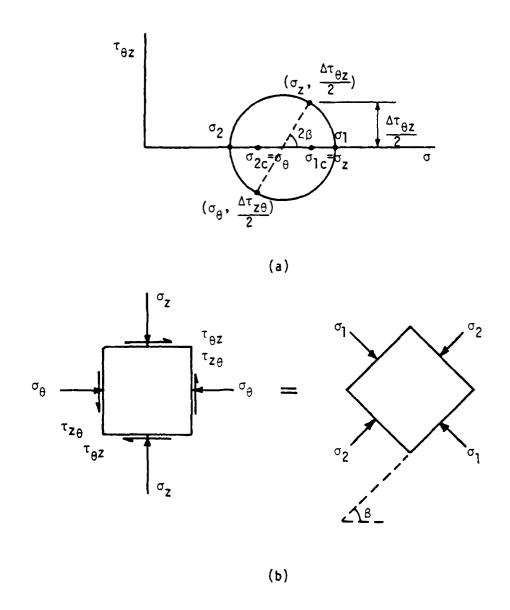


FIGURE 3-9 MOHR'S CIRCLE IN STRESS (a) AND TWO-DIMENSIONAL FREE BODY DIAGRAM (b) FOR HOLLOW CYLINDER SPECIMENS UNDER CONDITION OF MAXIMUM INSTANTANEOUS TORSIONAL STRESS

with Poisson's ratio and with time. This orientation effect is illustrated in Figure 3-9(b), which is a two-dimensional free-body diagram in the θz "plane".

Under these conditions of loading, the state of stresses is defined by the following equations:

$$\sigma_1 = \sigma_{1c} + \left[\frac{\Delta \tau_{\theta z}}{2} \cdot \sin(\lambda \cdot t) \cdot \tan\beta\right]$$
 (3.28)

$$\sigma_2 = \sigma_1 - 2(\sigma_1 - \sigma_{CT}) \tag{3.29}$$

$$\sigma_{3} = \sigma_{3c} \tag{3.30}$$

and
$$\beta = \frac{1}{2} \tan^{-1} \left[\frac{(\tau_{\theta z}/2) \cdot \sin(\lambda \cdot t)}{\sigma_{1c} - \sigma_{ct}} \right]$$
 (3.31)

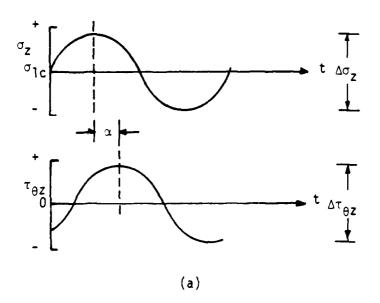
where
$$\sigma_{ct} = \left[\left(\frac{1 + \mu}{2} \right) \cdot \sigma_{1c} \right] + \left[\frac{\mu}{2} \cdot \sigma_{3c} \right]$$
 (3.32)

and
$$\sigma_{1c} = 1.85 \cdot \sigma_{3c}$$
. (3.20)

Combined Loading

In the case of simultaneous vertical and torsional excitation, the loading sequence is shown in Figure 3-10. The greatest complication in this stress state occurs because, although the two excitations are at the same frequency, they may be at different phases. This condition is shown in Figure 3-10(a). The maximum principal stress Mohr's circle in the θz "plane" is shown in Figure 3-10(b). The following equations define the state of stresses under these loading conditions:

$$\sigma_{1} = \sigma_{1c} + \left[\frac{\Delta \sigma_{z}}{2} \cdot \sin(\lambda \cdot t)\right] + \left[\frac{\Delta \tau_{\theta z}}{2} \cdot \sin[(\lambda - \alpha) \cdot t] \cdot \tan(\beta)\right]$$
(3.33)



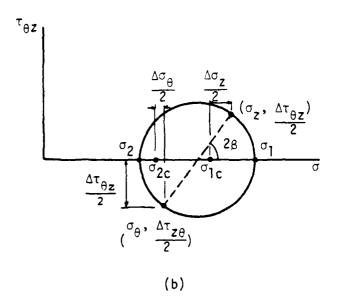


FIGURE 3-10 STRESS TIME HISTORIES (a) AND MOHR'S CIRCLE IN STRESS (b) FOR SIMULTANEOUS DYNAMIC LOADING OF HOLLOW CYLINDER SPECIMENS UNDER CONDITION OF MAXIMUM INSTANTANEOUS VERTICAL AND TORSIONAL STRESS

$$\sigma_2 = \sigma_1 - 2(\sigma_1 - \sigma_{ct}) \tag{3.34}$$

$$\sigma_3 = \sigma_{3C} \tag{3.35}$$

$$\beta = \frac{1}{2} \tan^{-1} \left[\frac{\Delta \tau_{\theta z} \cdot \sin[(\lambda - \alpha) \cdot t]}{2 \left[\sigma_{1c} + \frac{\Delta \sigma_{z}}{2} \cdot \sin(\lambda \cdot t) - \sigma_{ct} \right]} \right]$$
(3.36)

where

$$\sigma_{\text{ct}} = \left[\left(\frac{1 + \mu}{2} \right) \cdot \sigma_{\text{lc}} \right] + \left[\left(\frac{1 + \mu}{4} \right) \cdot \Delta \sigma_{z} \cdot \sin(\lambda \cdot t) \right] + \left[\frac{\mu \cdot \sigma_{3c}}{2} \right] (3.37)$$

$$\sigma_{1c} = 1.85 \cdot \sigma_{3c} \tag{3.20}$$

and $\boldsymbol{\alpha}$ is the phase lag of torsional loading to vertical loading.

A more detailed derivation of these equations appears in Appendix C-S.

Chapter 4

Soil Moduli and Damping Factors

Introduction

In this chapter the strain tensor developed in Chapter 2 and the state of stresses described in Chapter 3 will be further developed and combined to produce a complete picture of the stress-strain behavior of triaxial resonant-column and thin-walled hollow cylinder test specimens under dynamic loading conditions.

Resonant Column Testing

In the triaxial resonant-column testing series, the moduli are determined directly in the manner described in Chapter 3. Damping factors are also obtained during this testing series using either the amplitude decay technique or the logarithmic decrement approach.

Dynamic Interaction Theory

When evaluating the stress-strain response of these specimens under combined vertical and torsional loading, it is necessary to consider the effects of the reorientation of the principal stress directions during loading to evaluate the interaction.

As described in Chapter 3, the principal stresses and their directions will change throughout each combined loading test because the two excitations are at two independent frequencies. As a test is conducted, any interaction effects resulting from the reorientation of the principal stress directions will vary continuously, and the

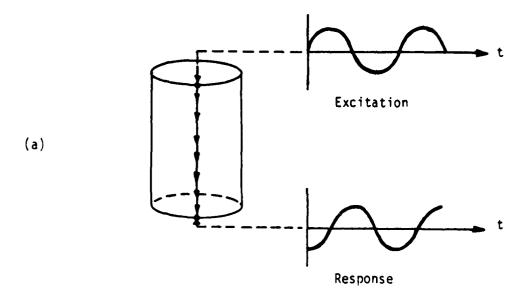
Lissajous figure observed on the oscilloscope will be an integration of the combined excitation and response over time. The modulus and damping factor calculated from this Lissajous figure will be "weighted average" values, averaging any interaction effects.

Because it is being assumed that the specimen is behaving as a linear, elastic, isotropic material, the direction of incremental strains during loading will coincide with the direction of incremental stresses, and the principal strain and stress directions will coincide. Consequently, the path of travel of the dynamic waves will be distorted to follow the direction of the principal stresses, and the effective velocity of propagation through the material will be decreased. This effect will evidence as a decrease in the resonant frequency of the specimen, and a decrease in the effective modulus of the material. This principle is illustrated graphically in Figure 4-1.

Data Reduction

A computer program has been developed which reduces the raw data from the resonant-column testing series to produce values of the moduli and damping factors under combined vertical and torsional dynamic loading conditions, and to calculate the values and directions of the maximum principal stresses during loading. This computer program, called Program RC, is included as Appendix D-1.

Example results of this testing series, as reduced by Program RC are presented in Appendices B-1 through B-3, and summaries of the results are given in Chapter 5. The complete results are available in Griffin (1980).



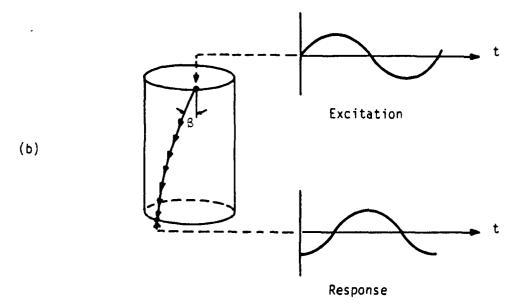


FIGURE 4-1 EFFECT OF THE DIRECTION OF THE INCREMENTAL STRESSES AND STRAINS ON PROPAGATION OF DISPLACEMENTS IN CYLINDRICAL RESONANT COLUMN SAMPLES

Hollow Cylinder Testing

In the thin-walled hollow cylinder testing series, the vertical and torsional stresses and strains were measured directly as described in Chapter 3. Moduli and damping factors may be calculated from this data provided the complete state of stresses and strains are known.

Consolidation Procedure

During each test, a test specimen was first isotropically consolidated to 0.5 KSC confining pressure by intercell vacuum. The specimen was then loaded vertically with an additional compressive stress to provide an anisotropic consolidation ratio, σ_{3c}/σ_{1c} , of 0.54. This occurred when the total vertical compressive stress was 0.925 KSC. While several tests were conducted upon specimens in this consolidation state, a number of tests received further consolidation preparation before dynamic testing was performed. As discussed earlier, tests were performed upon samples with consolidation lateral confining pressures of 0.5, 2.0, and 3.5 KSC.

For those tests conducted at the two higher confining pressures, the above consolidation process was continued in the same sequence as described above. First the effective cell pressure was increased from 0.5 KSC to 2.0 KSC, then the additional vertical compressive stress was increased from 0.425 KSC to the value causing an anisotropic consolidation ratio of 0.54. This additional vertical stress value was 1.7 KSC, resulting in a total vertical stress of 3.7 KSC.

Similarly, for specimens to be tested at a lateral confining pressure of 3.5 KSC, the effective cell pressure was increased to that

value, then the additional vertical stress was increased from 1.7 KSC to 2.975 KSC to create an anisotropic consolidation ratio of 0.54.

Consolidation Strains

During this consolidation process, the vertical displacement proximeter was installed immediately following the initial consolidation steps. It was therefore possible to record the vertical strain, ε_z , during the remaining consolidation steps where lateral confining pressures greater than 0.5 KSC were used. This record was made on the XY recorder for a number of tests, and the summary of these records are shown in Figures 4-2 through 4-7.

In Figures 4-2 and 4-3, the variation in vertical strain, ϵ_z , is shown for the increment of consolidation loading resulting from the change in intercell vacuum only. This change in stress corresponds to an isotropic increment of loading applied to a specimen which is already at some anisotropic state of stress. Since the XY recorder used to measure this strain increment was plotting it against the additional vertical stress change, which was zero for this increment of loading, it was not possible to show the true shape of the stress-strain response. The stress-strain response is therefore shown as a dashed line between the known values.

Figure 4-4 provides a summary of the data presented in Figures 4-2 and 4-3, and an extrapolation to allow for predicting the total consolidation strains. The predicted isotropic increments of vertical strain shown in Figure 4-4 are as follows:

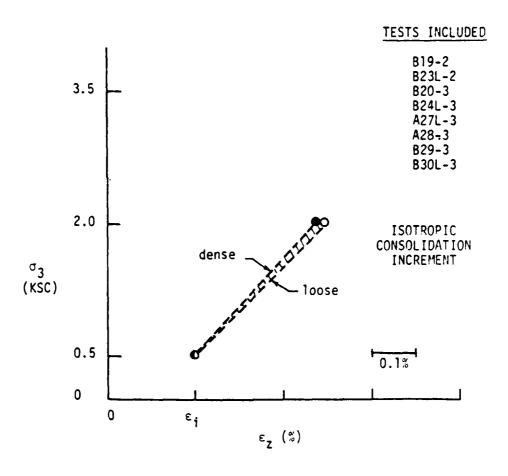


FIGURE 4-2 VARIATION OF ε_{Z} WITH ISOTROPIC INCREMENT OF LATERAL CONSOLIDATION LOADING, σ_{3} , FROM 0.5 TO 2.0 KSC, FOR SEVERAL TESTS

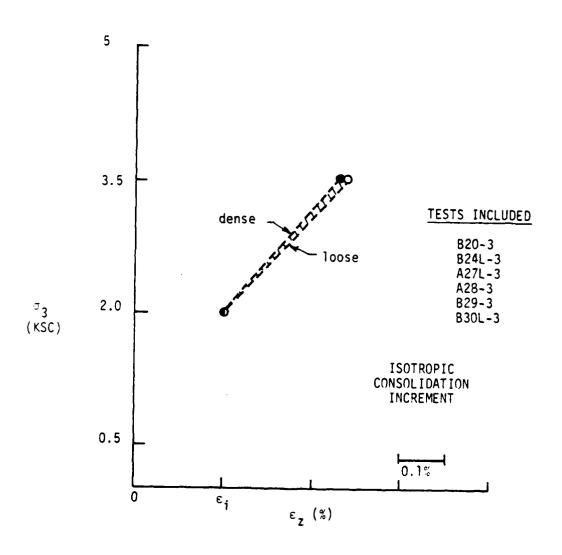


FIGURE 4-3 VARIATION OF $\varepsilon_{_Z}$ WITH ISOTROPIC INCREMENT OF LATERAL CONSOLIDATION LOADING, $\sigma_{_3}$, FROM 2.0 TO 3.5 KSC, FOR SEVERAL TESTS

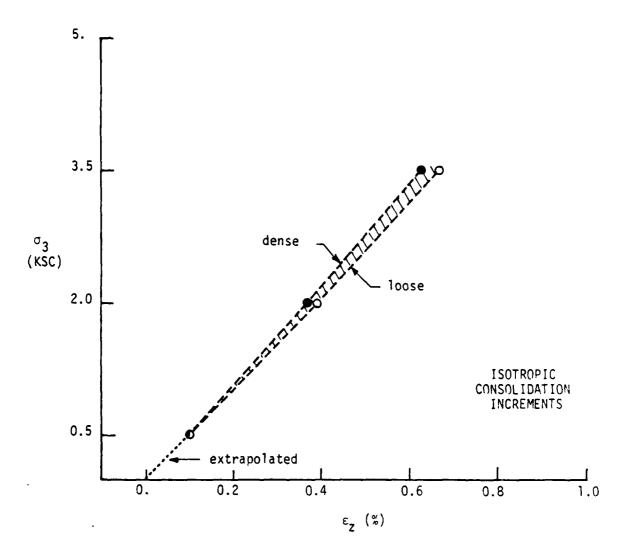


FIGURE 4-4 SUMMARY OF VARIATIONS OF ε_z WITH ISOTROPIC INCREMENTS OF LATERAL CONSOLIDATION LOADING, σ_3 : AND EXTRAPOLATION OF INITIAL INCREMENT OF STRAIN FROM 0 TO 0.5 KSC

$$\varepsilon_{z}^{(\sigma)} = 0.5 \text{ KSC} \approx 0.10$$
% (4.1)

$$\varepsilon_2 (\sigma_{3c} = 2.0 \text{ KSC}) \approx 0.37\%$$
 (4.2)

and
$$\varepsilon_z (\sigma_{3c} = 3.5 \text{ KSC}) \approx 0.65\% \tag{4.3}$$

The short extrapolated line in this figure is an estimate of the amount of vertical straining resulting from the initial application of 0.5 KSC of intercell vacuum upon the sample. This value could not be directly measured as the sample was still in the forming mold at the time of application of the intercell vacuum.

It should be emphasized that the stress-strain response approximated in Figure 4-4 is not truly representative of the stress-strain response of the samples to isotropic loading. Because all samples were in an anisotropic state of stress before the various isotropic loading increments were applied, the lateral stresses were increasing disproportionately with the vertical stress during the loading increment. The primary purpose of plotting this figure was to allow an extrapolation of the initial vertical strain increment value and the estimation of total consolidation strains.

For the anisotropic consolidation increments of stress, the vertical compressive stress was adjusted as described earlier to provide an anisotropic consolidation ratio of 0.54. Since both the change in vertical stress, $\Delta\sigma_z$, and the change in vertical strain, $\Delta\varepsilon_z$, were plotted on the XY recorder for this stress-strain increment, the actual shape of the stress-strain response is known.

In Figures 4-5 and 4-6 the anisotropic stress-strain increments are shown combined with the isotropic stress increments. The solid lines

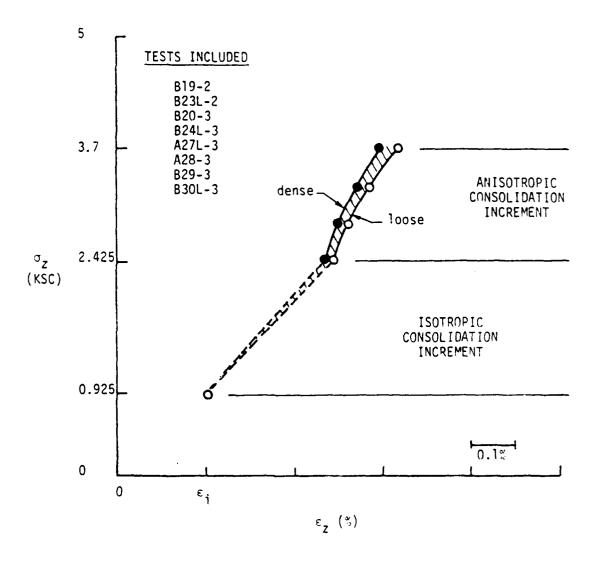


FIGURE 4-5 VARIATION OF ε_z WITH BOTH ISOTROPIC AND ANISOTROPIC INCREMENTS OF VERTICAL CONSOLIDATION LOADING, σ_z , FROM 0.925 TO 3.7 KSC, FOR SEVERAL TESTS

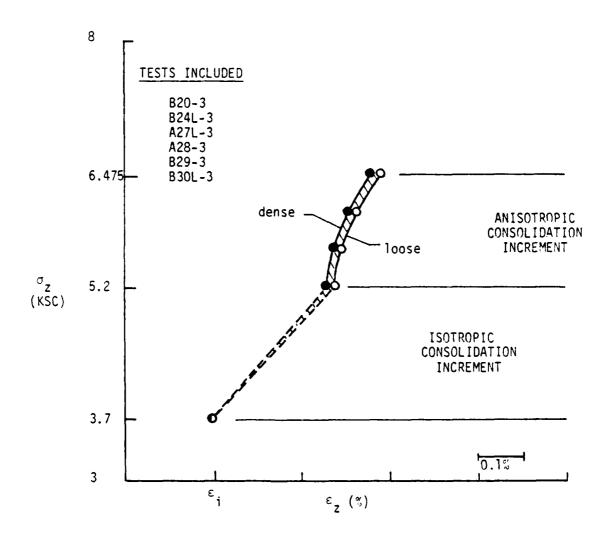


FIGURE 4-6 VARIATION OF ϵ_z WITH BOTH ISOTROPIC AND ANISOTROPIC INCREMENTS OF VERTICAL CONSOLIDATION LOADING, σ_z , FROM 3.7 TO 6.475 KSC, FOR SEVERAL TESTS

shown in these curves correspond to the actual anisotropic stressstrain response of the soil recorded by the XY recorder. The data
on these two figures are summarized in Figure 4-7, along with an extrapolation to estimate the total consolidation strain values. The
extrapolated line on Figure 4-7 includes the earlier isotropic increment
extrapolation from Figure 4-4, and an anisotropic increment extrapolation
deduced from the anisotropic loading increment data shown in the figure.
The total consolidation vertical strains may now be estimated as follows:

$$\varepsilon_{2}(\sigma_{3C} = 0.5 \text{ KSC}) \approx 0.15\%$$
 (4.4)

$$\varepsilon_{z}(\sigma_{3c} = 2.0 \text{ KSC}) \approx 0.56\%$$
 (4.5)

$$\varepsilon_z(\sigma_{3c} = 3.5 \text{ KSC}) \approx 0.93\%$$
 (4.6)

It is interesting to note that for all of the tests shown, which represent a wide range of densities, the recorded values of the total vertical strain under anisotropic consolidation conditions fall within a few percent of the average values in Equations 4.4 through 4.6.

Consolidation Modulus Calculation

From the strain tensor developed in Equation 2.17, we can calculate the volumetric strain as follows:

$$\varepsilon_{\text{vol}} = \left(1 - \frac{\mu + \mu^2}{1 - \mu^2}\right) \cdot \varepsilon_{z} \tag{4.7}$$

and from Hooke's Law (Equation 2.4),

$$\varepsilon_{\mathbf{z}} = \left(\frac{1 - \mu^{2}}{E}\right) \cdot \sigma_{\mathbf{z}} - \left(\frac{\mu + \mu^{2}}{E}\right) \cdot \sigma_{\mathbf{R}}$$
 (4.8)

and

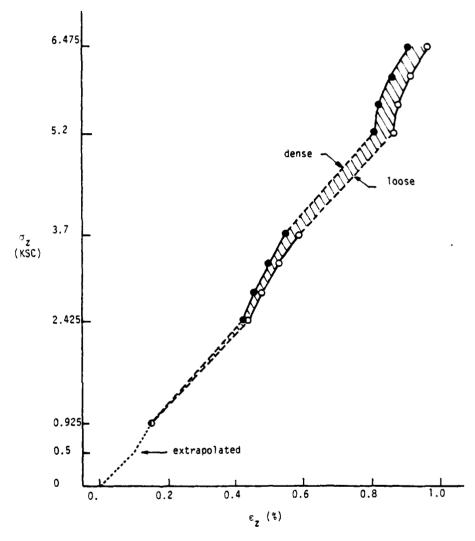


FIGURE 4-7 SUMMARY OF VARIATIONS OF ε_z WITH ISOTROPIC AND ANISOTROPIC INCREMENTS OF VERTICAL CONSOLIDATION LOADING, σ_z ; AND EXTRAPOLATION OF INITIAL INCREMENT OF STRAIN FROM 0 TO 0.925 KSC

$$\varepsilon_{R} = \left(\frac{1 - \mu^{2}}{E}\right) \cdot \sigma_{R} - \left(\frac{\mu + \mu^{2}}{E}\right) \cdot \sigma_{z}$$
 (4.9)

which reduces to:

$$\varepsilon_{\text{vol}} = \left(\frac{1 - \mu - 2\mu^2}{E}\right)(\sigma_z + \sigma_R) . \qquad (4.10)$$

Combining Equations 4.7 and 4.10 gives:

$$\left(1 - \frac{\mu + \mu^2}{1 - \mu^2}\right) \varepsilon_z = \left(\frac{1 - \mu - 2\mu^2}{E}\right) (\sigma_z + \sigma_R) \tag{4.11}$$

which reduces to:

$$E = \frac{(1 - \mu - 2\mu^2) \cdot (\sigma_z + \sigma_R)}{\left[1 - \left(\frac{\mu + \mu^2}{1 - \mu^2}\right)\right] \cdot \varepsilon_z}$$
(4.12)

The static compression modulus E was calculated from Equation 4.12 for a variety of values of Poisson's ratio and the results are shown in Table 4-1. The similarity in the calculated moduli for a specific Poisson's ratio is a measure of the similarity in the principal stress ratio during consolidation, and of linearity of the stress-strain response.

Tangent and Secant Moduli

During dynamic excitation, the dynamic vertical stress and strain will be superimposed upon the consolidation stress-strain condition.

Consequently, it is possible to view either the tangent modulus or the secant modulus under these loading conditions.

Vertical straining during dynamic loading will be in accordance with the strain tensor developed in Equation 2.17 and Hooke's Law as presented in Equation 2.4. The dynamic tangent compression modulus

TABLE 4-1
Static Unconstrained Compression Modulus (in psi)

μ	$E(\sigma_{3c} = 0.5 \text{ KSC})$	$E(\sigma_{3c} = 2.0 \text{ KSC})$	$E(\sigma_{3c} = 3.5 \text{ KSC})$
0.49	10,612.	11,370.	11,981.
0.44	11,261.	12,066.	12,714.
0.39	11,841.	12,687.	13,369.
0.34	12,351.	13,233.	13,944.
0.29	12,791.	13,704.	14,441.
0.24	13,161.	14,101.	14,859.
ĺ			

under vertical or combined vertical and torsional dynamic loading conditions may be calculated by the following expression:

$$E = (1 - \mu^2) \frac{\Delta \sigma_z}{\Delta \varepsilon_z}$$
 (4.13)

A derivation of this expression is given in Appendix C-6.

Similarly, Equation 4.12 may be rewritten to provide a means for calculating the dynamic secant compression modulus, producing the following:

$$E = \left[\frac{1 - \mu - 2\mu^2}{1 - \left(\frac{\mu + \mu^2}{1 - \mu^2}\right)}\right] \cdot \frac{\sigma_z + \sigma_R + \frac{\Delta \sigma_z}{2}}{\varepsilon_z + \frac{\Delta \varepsilon_z}{2}}$$
(4.14)

where $\sigma_z = \sigma_{lc}$, $\sigma_R = \sigma_{3c}$, and $\varepsilon_z = \varepsilon_{lc}$.

The secant compression modulus may be simply calculated using

Equation 4.14. Because this modulus is an overall average modulus from

the preconsolidation state, the values will not vary greatly from those

values in Table 4-1 except under conditions of very high dynamic straining.

If the soil sample is considered representative of soil conditions in the field, with anisotropic consolidation, then the compression modulus of interest is the tangent compression modulus determined using Equation 4.13. This modulus will reflect the dynamic stress-strain behavior of soil which has reached a steady-state consolidation condition, and whose strain will vary throughout the range of strains imposed in this testing program.

Effect of Torsional Loading on Equations for Moduli

Under conditions of torsional dynamic loading alone, the principal stress and strain directions will change during loading as discussed in Chapter 3. The Mohr's circles in strain and stress for this condition are indicated in Figure 4-8, with the reorientation of the principal strains in the θz "plane" indicated by the angle ψ . If the material behaves linearly and elastically, then ψ must equal β .

Note from Figure 4-8(a) that the principal strains may now be written as follows:

$$\varepsilon_1 = \varepsilon_{1c} + \left[\frac{\Delta \gamma_{\theta z}}{4} \cdot \sin(\lambda \cdot t) \cdot \tan\psi\right]$$
 (4.15)

$$\varepsilon_2 = 0 - \left[\frac{\Delta \gamma_{\theta z}}{4} \cdot \sin(\lambda \cdot t) \cdot \tan \psi \right]$$
 (4.16)

$$\varepsilon_3 = \varepsilon_{3c}$$
 (4.17)

and

$$\psi = \frac{1}{2} \tan^{-1} \left[\frac{\Delta \gamma_{\theta z} \cdot \sin(\lambda t)}{2\varepsilon_{1c}} \right]$$
 (4.18)

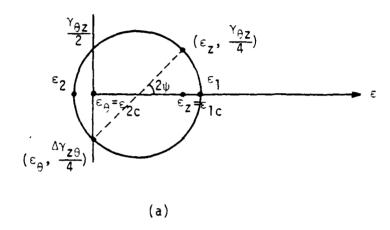
where ε_{1c} = ε_{z} , ε_{2c} = ε_{θ} = 0, and ε_{3c} = ε_{R} .

The volumetric strain may now be written:

$$\varepsilon_{\text{vol}} = \varepsilon_1 + \varepsilon_2 + \varepsilon_3 = \varepsilon_{1c} + \varepsilon_{3c}$$

$$= \left[1 - \frac{\mu + \mu^2}{1 - \mu^2}\right] \varepsilon_z .$$
(4.19)

Because Equation 4.19 is exactly equal to Equation 4.7, the expression for the volumetric strain during dynamic loading is shown to be independent of recrientation of the principal stress and strain directions. The expressions for the compression modulus (Equations 4.13 and 4.14) are derived from the expression for volumetric strain,



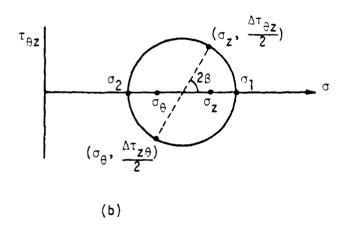


FIGURE 4-8 MOHR'S CIRCLE IN STRAIN (a) AND IN STRESS (b) FOR HOLLOW CYLINDER TEST SPECIMENS UNDER CONDITION OF MAXIMUM INSTANTANEOUS TORSIONAL STRESS AND STRAIN

thus those expressions are also independent of reorientation of the principal stress and strain directions.

The dynamic shear modulus under torsional loading conditions may be calculated by the following expression:

$$G = \frac{\Delta \tau_{\theta z}}{\Delta \gamma_{\theta z}} \tag{4.20}$$

Under conditions where only one modulus can be directly calculated from the data presented, or where it is desired to predict the effective Poisson's ratio under combined vertical and torsional loading conditions, the following expression may be used:

$$E = 2(1 + \mu) \cdot G$$
 (4.21)

Data Reduction

A computer program has been written which calculates values of the tangent and secant dynamic compression moduli, the dynamic shear modulus, and the magnitude and direction of the maximum principal stresses and strains from the raw test data as a function of the Poisson's ratio.

This computer program is called Program HC, and is included as Appendix D-2.

Example results of this testing series, as reduced from Program HC, are presented in Appendices B-4 through B-6, and complete results are summarized in Chapter 6. The complete results are available in Griffin (1980).

Damping

Introduction

In both the triaxial resonant-column and the thin-walled hollow cylinder testing series energy is dissipated during dynamic loading due to damping. Generally speaking, the greater the strain amplitude, the greater will be this energy loss. With closely monitored laboratory tests such as these, the energy loss measured is due solely to the inelastic response of the materials under test, and is sometimes referred to as hysteretic damping or friction damping. If the material behaves linearly and elastically, the energy loss and thus the damping, will be zero. As long as the damping is small, it is quite possible (and common practice) to treat the peak-to-peak stress strain response as an "equivalent-linear" modulus, and introduce damping as an energy dissipation over time.

Resonant Column Testing

In the triaxial resonant-column testing series, the damping was measured using either the amplitude decay technique or the multiplication factor approach. The first of these involves the excitation of the specimen at its resonant frequency and observing the acceleration response as a function of time. When the driving force is abruptly stopped, the sample will continue to oscillate freely, but the acceleration amplitude will decay with time because of damping. The value of the damping may be calculated as follows:

$$\Delta = \frac{1}{n} \left(\ln \frac{a_0}{a_n} \right) \tag{4.22}$$

where Δ = amplitude decay damping

n = number of cycles, where <math>n > 1

 a_0 = acceleration amplitude of the 0 - th cycle

 $a_n = acceleration$ amplitude of the n - th cycle

This is illustrated graphically in Figure 4-9(a).

The amplitude decay damping is related to the ratio of critical damping, D, by the following expression:

$$\Delta = \frac{2 \pi D}{\sqrt{1 - D^2}} \tag{4.23}$$

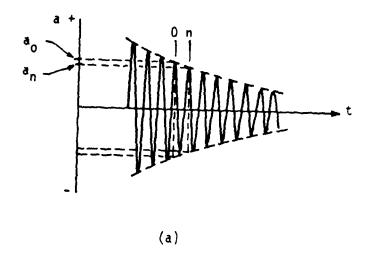
and, for small values of damping, D may be expressed in percent as follows:

$$D(%) \simeq \frac{\Delta}{2\pi} \cdot 100\% \tag{4.24}$$

The use of the multiplication factor approach requires a look at the response spectrum for the specimen as shown in Figure 4-9(b). The resonant frequency is obtained as described in Chapter 3; then frequencies and amplitudes may be determined from other points along the "bell shaped curve," as shown in Figure 4-9(b).

The sharpness of the "bell" is an indication of the energy dissipation of the sample. A perfectly linear, elastic material would respond at one precise resonant frequency, and would have a response spectrum which is just a vertical line at the resonant frequency.

The values of frequency and response shown in Figure 4-9(b) are convenient because they lead to relatively simple expressions for calculating the damping. The values of f_1 and f_2 were selected such that the acceleration response at those frequencies was $1/\sqrt{2}$ times the



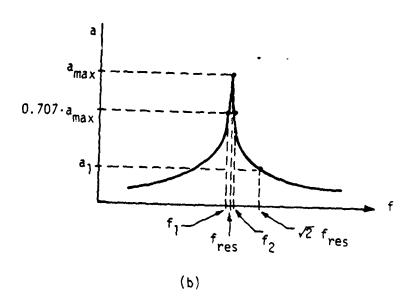


FIGURE 4-9 AMPLITUDE DECAY METHOD (a) AND MULTIPLICATION FACTOR APPROACH (b) FOR COMPUTING DAMPING IN RESONANT COLUMN TESTS

maximum acceleration response occuring at or near resonance. Using these frequencies, the damping may be expressed as follows:

$$\Delta = \frac{\pi (f_2 - f_1) a_{\text{max}}}{f_{\text{res}}}$$
 (4.25)

or

$$D(%) \simeq \frac{(f_2 - f_1) \cdot a_{max}}{2 f_{res}} \cdot 100%$$
 (4.26)

The value a_1 was selected such that the frequency at which it occurred is equal to $\sqrt{2}$ times the resonant frequency. In this case the damping may be calculated as follows:

$$\Delta = \frac{\pi}{2} \cdot \left[\frac{\Delta \sigma_{\text{amax}}}{a_{\text{max}}} \right] \cdot \left[\frac{a_1}{\Delta \sigma_{\text{a}1}} \right]$$
 (4.27)

or

$$D(%) \simeq \frac{1}{4} \left[\frac{\Delta \sigma_{\text{amax}}}{a_{\text{max}}} \right] \left[\frac{a_1}{\Delta \sigma_{\text{al}}} \right] \cdot 100\%$$
 (4.28)

where $\Delta \sigma_{amax}$ = driving stress at a response

 $\Delta\sigma_{al}$ = driving stress at a_1 response

If the driving force is held constant throughout the response spectrum, which is usually the case, Equations 4.27 and 4.28 reduce to the following:

$$\Delta = \frac{\pi a_1}{2a_{\text{max}}} \tag{4.29}$$

and

$$D = \frac{a_1}{4a_{max}} \cdot 100\%$$
 (4.30)

The practical advantage to using Equations 4.27 and 4.28 in the form presented is that the factor $\left[\frac{a_1}{\Delta\sigma_{a1}}\right]$ does not change appreciably

during testing, and normally needs to be measured only once. The damping may then be calculated by merely knowing the acceleration response and the driving stress at resonance throughout testing. The damping equations may thus be rewritten as follows:

$$\Delta = \frac{2\pi\delta}{1003} \cdot \left(\frac{\Delta\sigma_{\text{amax}}}{a_{\text{max}}}\right)$$
 (4.31)

and

$$D(%) = \delta \cdot \left(\frac{\Delta \sigma_{\text{amax}}}{a_{\text{max}}}\right)$$
 (4.32)

where

$$\delta = \begin{cases} \frac{100\%}{4} \cdot \left[\frac{a_1}{\Delta \sigma_{a1}}\right] \\ \text{damping calibration factor} \end{cases}$$
 (4.33)

Hollow Cylinder Testing

In the thin-walled hollow cylinder testing series, the energy dissipation is seen as a hysteresis loop stress-strain response, as shown in Figure 4-10. The "equivalent linear" modulus is shown as the ratio of peak-to-peak stress to peak-to-peak strain. In Figure 4-10, the following equations may be written:

$$\overline{AB} = \overline{A'B'} = \frac{\Delta\sigma}{2} \tag{4.34}$$

$$\overline{OB} = \overline{B'O} = \frac{\Delta \varepsilon}{2}$$
 (4.35)

$$E_{ps} = \frac{\overline{AB}}{\overline{OB}} = \frac{\overline{A'B'}}{\overline{B'O}}$$
 (4.36)

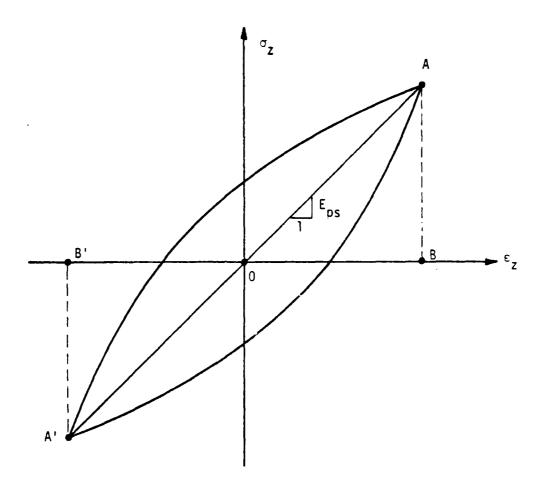
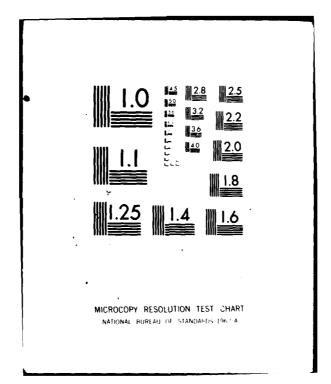


FIGURE 4-10 HYSTERETIC STRESS-STRAIN CURVE FOR DYNAMIC VERTICAL LOADING IN HOLLOW CYLINDER TEST

CALIFORNIA UNIV BERKELEY EARTHQUAKE ENGINEERING RES.—ETC F/G 8/13
INTERACTION EFFECTS OF SIMULTANEOUS TORSIONAL AND COMPRESSIONAL.—ETC(U)
DEC 79 P M GRIFFIN. W N HOUSTON
ICG/EZERC_79/34

ARO-13838.1-85

NL AD-A092 352 UNCLASSIFIED



The damping may be expressed as follows:

$$\lambda = \frac{A_{\ell}}{A_{ABO} + A_{A'B'O}}$$
 (4.37)

where

 A_{ℓ} = area of the loop

 A_{ABO} = area of the triangle ABO

 $A_{A'B'O}$ = area of the triangle A'B'O

This value of hysteretic damping is related to the ratio of critical damping by the following relationship:

$$\lambda = \frac{2 \pi D}{\sqrt{1 - D^2}} \tag{4.38}$$

or, for small values of damping,

$$D(\$) \simeq \frac{\lambda}{2\pi} \cdot 100\$ \tag{4.39}$$

Chapter 5

Low Strain Combined Cyclic Loading

Introduction

In this chapter the actual laboratory test results reduced from the triaxial resonant column series will be presented. Shown in Figure 5-1 is a photograph of the resonant column testing device with a specimen in place. Listed in Table 5-1 is a summary of the seventeen tests which have been analyzed for this presentation.

As discussed previously, the moduli during dynamic combined loading will change both with vertical strain amplitude and with compression-shear interaction. The data will be presented in such a way that the two effects may be seen both separately and together. The results will also be presented as a function of the octahedral shearing strain.

Vertical Loading Alone

The dynamic compression modulus is shown as a function of the cyclic vertical strain under conditions of vertical loading alone in Figure 5-2. It should be noted that for all values of strain referred to herafter, the single-amplitude values are used. The cyclic stress amplitudes shown on drawings, however, are peak-to-peak values. This Figure includes data from a variety of samples at various densities and confining pressures. Superimposed upon the drawing are contour lines showing values of equal vertical normal cyclic stress, $\Delta \sigma_z$. From this Figure we can see that the range of cyclic vertical normal stress used in this testing series was from approximately 0.02 psi to 25 psi. The corresponding vertical

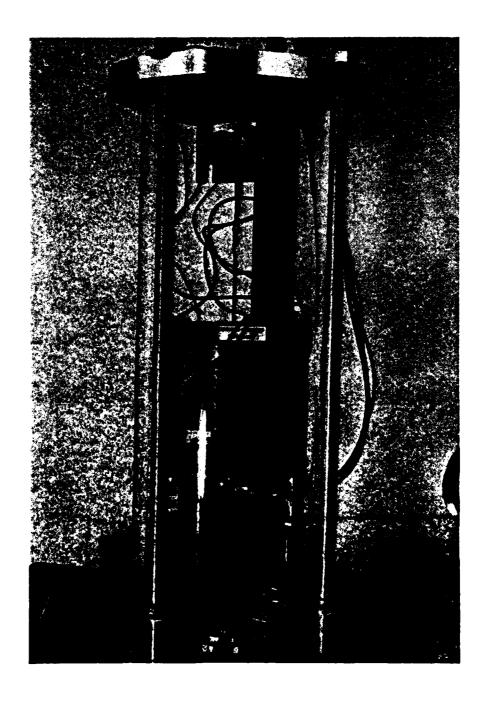


FIGURE 5-1 RESONANT COLUMN TESTING APPARATUS

TABLE 5-1
Summary of Resonant Column Tests

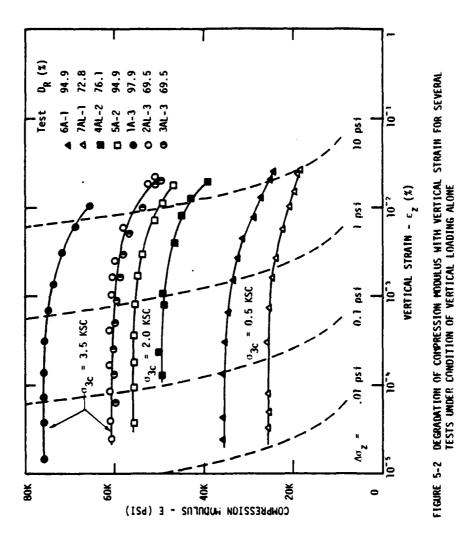
TEST NO.	Υ _d (g/cc)	D _R (%)	σ _{3c} (KSC)	V-VERTICAL MODE T-TORSIONAL C-COMBINED
1A-3	1.70	97.9	3.5	v
2AL-3	1.61	69.5	3.5	V
3AL-3	1.61	69.5	3.5	v
4AL-2	1.63	76.1	2.0	v
5A-2	1.69	94.9	2.0	v
6A-1	1.69	94.9	0.5	V
7AL-1	1.62	72.8	0.5	v
9BL-1	1.62	72.8	0.5	С
108-1	1.70	97.9	0.5	c
11BL-2	1.64	79.3	2.0	С
12B-2	1.69	94.9	2.0	С
13BL-3	1.64	79.3	3.5	С
14B-3	1.68	91.8	3.5	С
15CM-1	1.68	91.8	0.5	T
16CL-1	1.64	79.3	0.5	T
17CL-3	1.64	79.3	3.5	T
18C-3	1.67	88.8	3.5	T

 γ_d = dry density of soil

D_R = relative density

 σ_{3c} = lateral confining pressure

`



strain amplitudes were in the range of from approximately 10⁻⁵% to 3x10⁻²%. It is interesting to note the steepness of the stress contours, indicating that relatively small variations in strain amplitude will yield relatively large variations in moduli.

To eliminate variations in the modulus-strain curves due to sample peculiarities, density variations, and small measurement errors, it is desirable to "normalize" these curves by plotting a "relative modulus" in place of the absolute value of modulus. This technique allows a direct evaluation of the relative variation of modulus with strain amplitude.

In Figure 5-3 six tests are normalized to the value of the modulus at a single amplitude vertical strain of 5X10⁻³%. This value was selected to allow for comparison of these test results with those of the higher strain, thin-walled hollow cylinder testing series which will be presented in Chapter 6.

From Figure 5-3 it can be seen that the greatest rate of change of modulus with strain amplitude occurs at the lowest confining pressures. Although there is some variation in this rate as a function of density, this variation is small when compared with the influence of the confining pressure on the rate of variation.

Torsional Loading Alone

The results of four tests in which pure torsional loading was applied to samples are presented in Figure 5-4. As with Figure 5-2, the modulus (in this case the Shear Modulus) is seen to change as a function of the strain amplitude. In these tests the range of the cyclic torsional shear stress was from approximately .01 psi to 15 psi, very similar to

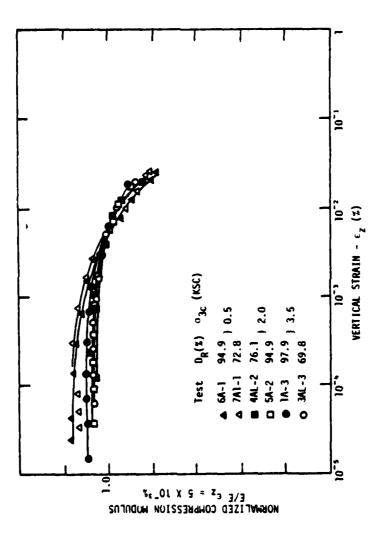


FIGURE 5-3 DEGRADATION OF NORMALIZED COMPRESSION MONULUS WITH VERTICAL STRAIN FOR SEVERAL TESTS UNDER CONDITION OF VERTICAL LOADING ALONE

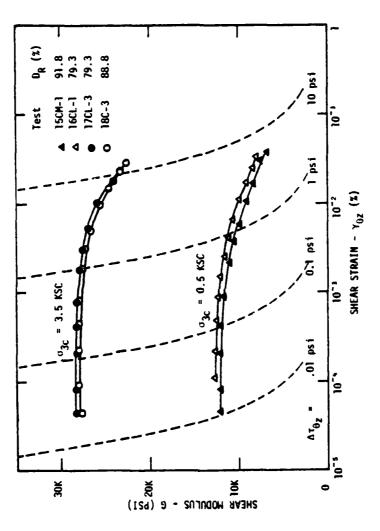


FIGURE 5-4 DEGRADATION OF SHEAR PROBLUS WITH SHEAR STRAIN FOR SEVERAL TESTS UNDER CONDITION OF SHEAR LOADING ALONE

the load range in vertical compression. The corresponding torsional shear strain amplitudes were in the range of from approximately 4×10^{-5} % to 4×10^{-2} %.

The shear modulus-shear strain curves were normalized to the modulus at a shear strain of 5×10^{-3} and presented in Figure 5-5. In this Figure it can be seen that the relative variation of shear modulus with strain amplitude is somewhat greater than the variation of compression modulus shown in Figure 5-3 over a similar strain range.

Combined Loading

Under conditions of simultaneous vertical and torsional dynamic loading, the modulus will vary both as a function of the vertical strain amplitude and as a function of the compression-shear interaction effects. As indicated in Table 5-1, six tests performed under simultaneous combined loading conditions have been presented and analyzed. These tests represented two different densities and three different confining pressures.

Interaction Effects

The effects of this interaction on two of these tests are shown in Figure 5-6. In this Figure modulus-strain amplitude curves are plotted for the two test specimens, with contours of equal shear strain amplitude superimposed on the drawing. The curves labeled $\gamma_{\theta z} < 10^{-4}$ % are those corresponding to the "virgin" modulus-strain curve in which no measurable compression-shear interaction effects are present. These are the solid lines in this Figure.

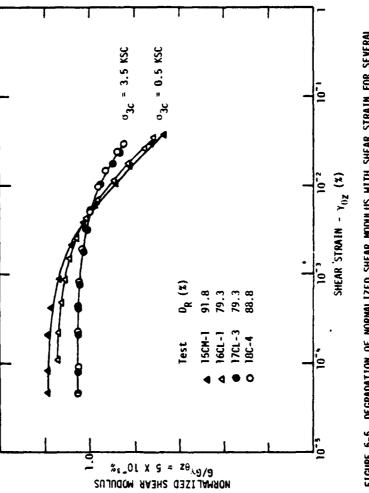
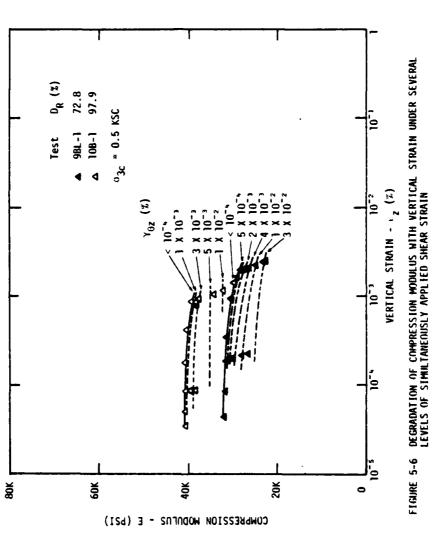


FIGURE 5-5 DEGRADATION OF NORMALIZED SHEAR MODULUS WITH SHEAR STRAIN FOR SEVERAL TESTS UNDER CONDITION OF SHEAR LOADING ALONE



Strain Ratio Effects

These interaction effects can be isolated from the other factors influencing the modulus value by plotting a normalized modulus vs. the ratio of average shear strain, $\gamma_{\theta z}$, (see Chapter 2 for definition) to the vertical normal strain, ε_z . The modulus should be normalized to the modulus at that vertical strain amplitude on the "virgin" modulus-strain curve, $E_{<<\gamma_{A_z}}$.

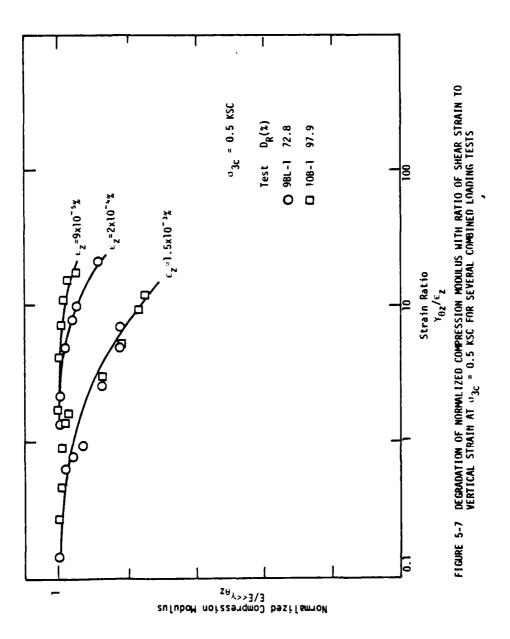
These normalized plots were prepared for all of the combined loading tests in this series, and are presented in Figures 5-7, 5-8, and 5-9, for confining pressures of 0.5, 2.0, and 3.5 KSC, respectively. These normalized curves are combined and summarized for those tests with a relative density of 95% in Figure 5-10; and for all combined loading tests in Figure 5-11.

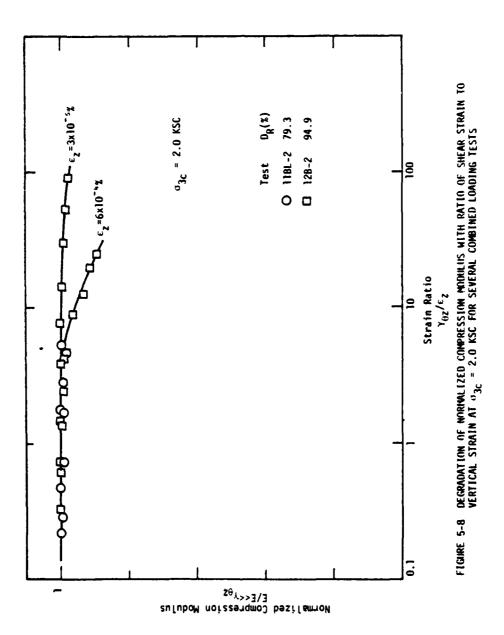
Octahedral Shearing Strain Effects

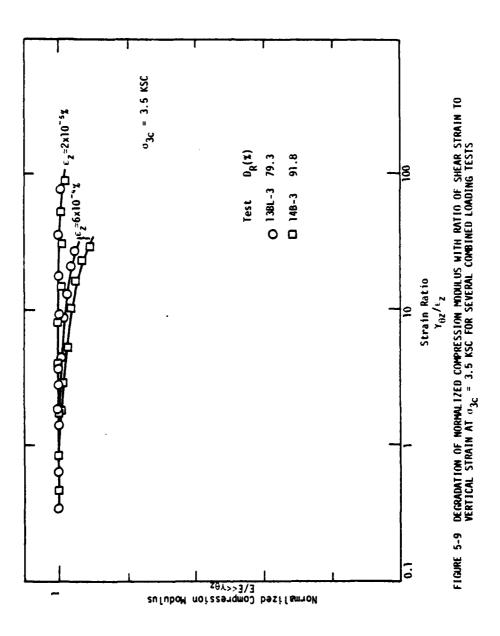
As discussed in Chapter 2, the octahedral shearing strain, $\gamma_{\rm OCT}$, is a very useful tool in evaluating the degradation of modulus with strain amplitude. For these testing series a degradation of modulus has been observed both with increasing dynamic axial strain, $\Delta \varepsilon_z$, and with increasing dynamic shear strain, $\Delta \gamma_{\theta z}$. The octahedral shearing strain combines both of these types of straining, and should be very useful in studying the combined degradation effects.

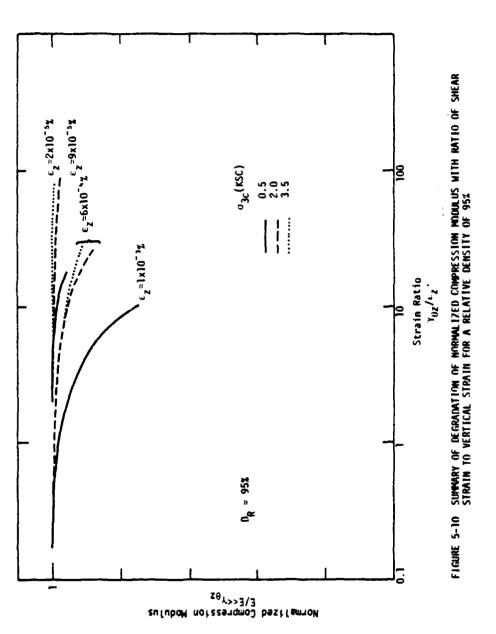
It should be noted that if the material is homogeneous and isotropic, then the degradation of modulus with octahedral shearing strain, γ_{OCT} , will be independent of how the strain was developed; i.e., from axial or torsional straining.

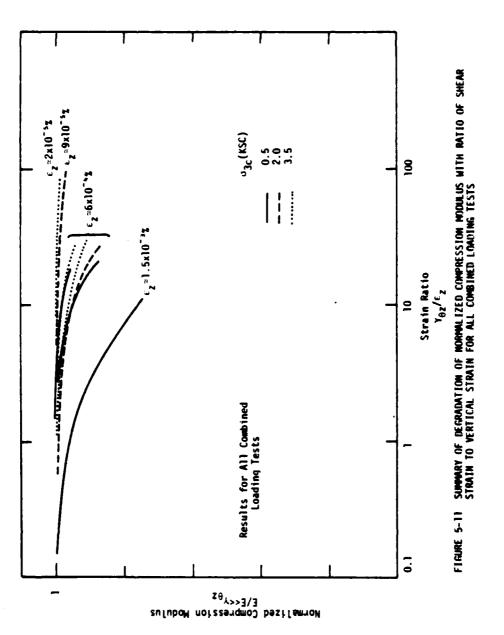
In Figure 5-12 are shown the "virgin" normalized compression moduli,

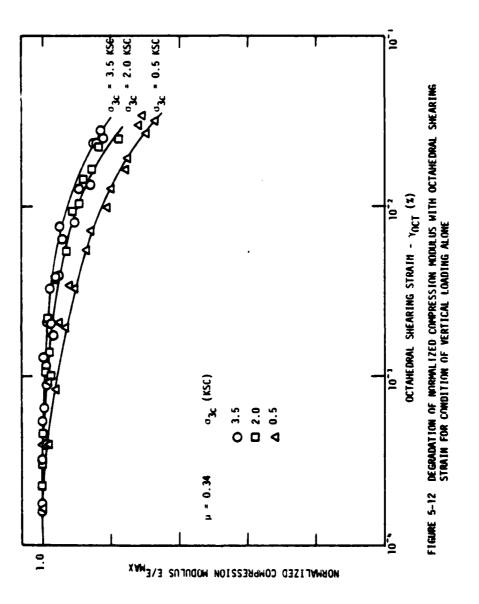










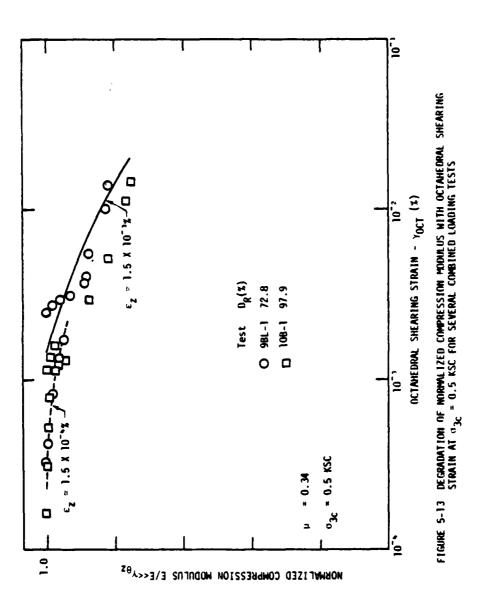


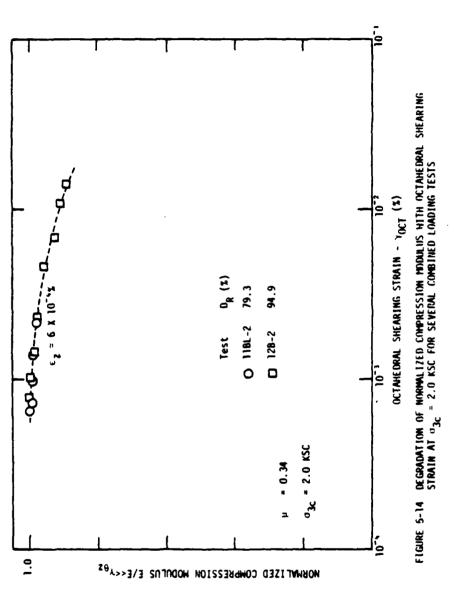
 E/E_{MAX} , plotted vs. γ_{OCT} . Because these curves were developed by applying vertical loading alone to the soil specimens, the octahedral shearing strain in that Figure is developed purely from vertical, radial, and tangential normal straining. These normal strains correspond to the principal strains in this case.

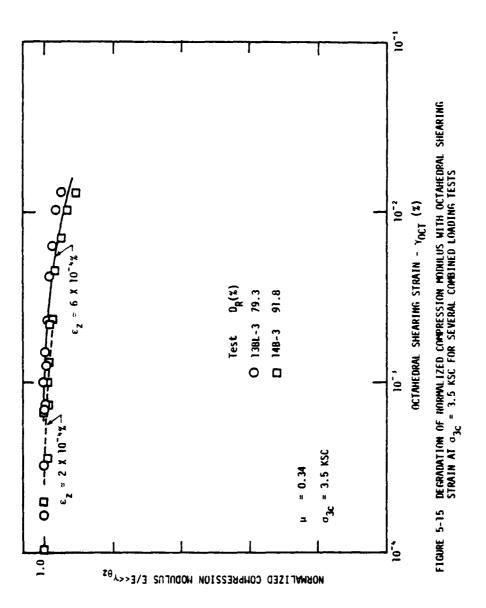
In the combined loading case, shear strains introduced in the θ_Z "plane" result in a reorientation of the principal strain directions; thus, the octahedral shearing strain in this case is developed from the combination of vertical, radial, and tangential normal strains, plus θ_Z "plane" shear strains.

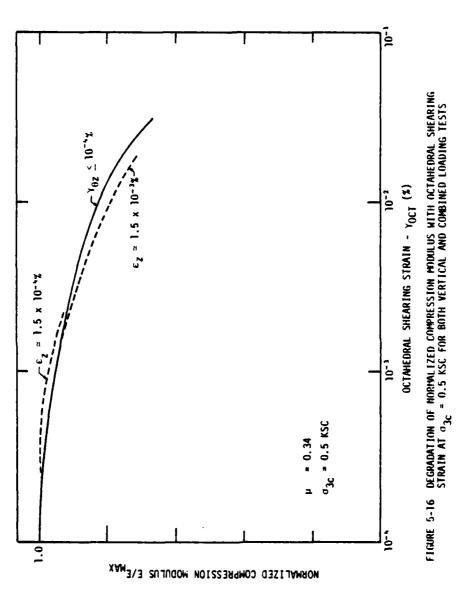
The variation of compression moduli with the virgin compression moduli, as a function of octahedral shearing strain is shown in Figures 5-13, 5-14, and 5-15 for confining pressures of 0.5, 2.0, and 3.5 KSC, respectively. These curves are combined with the virgin curves from Figure 5-12 to determine the effects of shear-compression interaction, and are presented in Figures 5-16, 5-17, and 5-18 for the same three confining pressures. For the dashed curves the increased in $\gamma_{\rm OCT}$ arise from increases in $\gamma_{\rm OCT}$ with $\varepsilon_{\rm z}$ held constant as shown. For the solid curves, the increases in $\gamma_{\rm OCT}$ arise from increases in $\varepsilon_{\rm z}$ with $\gamma_{\rm \theta z}$ held as small as possible, typically less than 10^{-4} % as shown. The intersection of the dashed curves and the solid curve occurs when the contribution of $\gamma_{\rm \theta z}$ to $\gamma_{\rm OCT}$ is small compared to the contribution of $\varepsilon_{\rm z}$ for these tests. Therefore the value of $\gamma_{\rm OCT}$ at which the intersection occurs can be computed directly from $\varepsilon_{\rm z}$ for practical purposes.

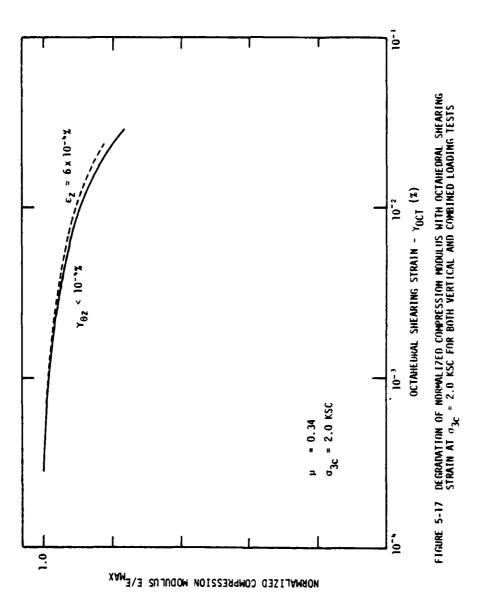
In these Figures it is apparent that the two types of curves are essentially coincident, and thus any shear-compression interaction effect which is not accounted for by the use of the octahedral shearing strain



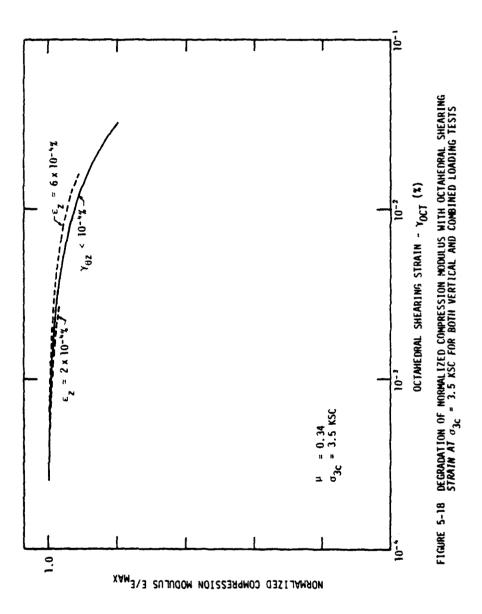








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is negligible.

The resultant best-fit normalized moduli-octahedral shearing strain curves are presented in Figure 5-19.

Conclusions

The various test results were presented in different ways to evaluate the relative compression-shear interaction under combined loading conditions.

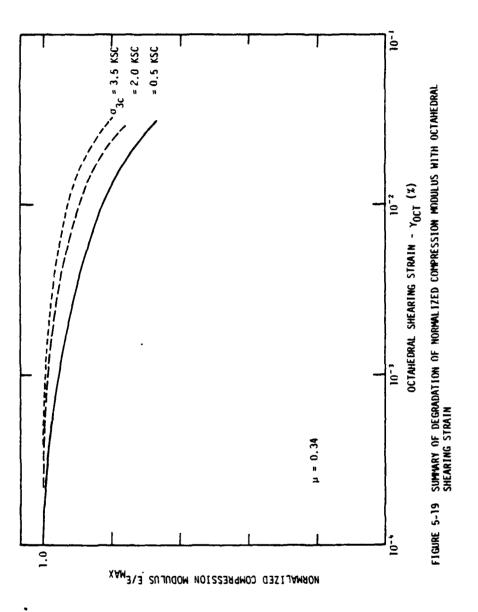
Strain Ratio Effects

From the results of the presentation of the relative normalized compression moduli plotted against the ratio of average shear strain, $\gamma_{\theta Z}$, to vertical normal strain, ε_{Z} , in Figure 5-11, it is apparent that there exists a "threshold" strain ratio below which negligible interaction effects are observed. It appears that the minimum threshold strain ratio in this testing program was approximately 1, where the average shear strain equaled the vertical normal strain. This means that when the shear strain, $\gamma_{\theta Z}$, was numerically smaller than the vertical normal strain the degradation of modulus due to $\gamma_{\theta Z}$ was negligible.

Also apparent is the fact that the moduli-strain ratio curve and the threshold strain ratio depend primarily upon the vertical normal strain amplitude. The effects of confining pressure and relative density upon these curves appears to be very small.

Octahedral Shearing Strain Effects

From the results of the presentation of normalized compression



moduli plotted against the octahedral shearing strain in Figure 5-19 it appears that the compression-shear interaction is directly predictable or "expressible" by use of the octahedral shearing strain.

The maximum octahedral shearing strain developed in this testing series was approximately $3 \times 10^{-2} \%$.

Chapter 6

High Strain Combined Cyclic Loading

Introduction

In this chapter the results of the thin-walled hollow cylinder testing program will be presented. A photograph of the testing apparatus is shown in Figure 6-1. A summary of the twenty-one tests which make up this testing series is presented in Table 6-1.

As with the low strain triaxial resonant column testing program, the data is presented in such a manner that changes in the dynamic compression modulus due to vertical strain amplitude and due to combined loading interaction may be evaluated separately and independently of one another. Additionally, the vertical compression stress-strain response under conditions of very high shear loading will be presented and discussed.

Vertical Loading Alone

In Figure 6-2 the dynamic compression modulus is shown as a function of the vertical normal strain for a value of the Poisson's ratio of 0.34. The curves shown are "virgin" curves, with no simultaneously applied shear loading. The value selected for the Poisson's ratio is near the center of the normal range of from approximately 0.2 to 0.5. Variation of the Poisson's ratio within that range will have little effect upon these curves, since the calculated value of the compression modulus is not very sensitive to the Poisson's ratio.

Superimposed upon Figure 6-2 are contours of equal values of cyclic vertical normal stress, $\Delta\sigma_z$. The range of $\Delta\sigma_z$ in this testing series was

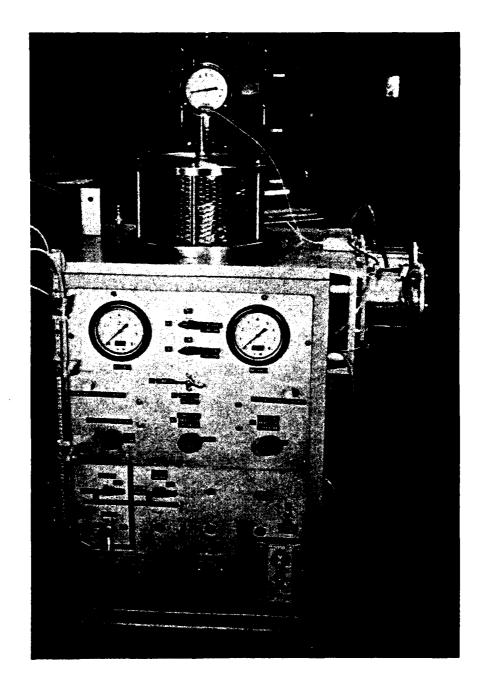


FIGURE 6-1 HOLLOW CYLINDER TESTING APPARATUS AND CONTROL PANEL

TABLE 6-1
Summary of Hollow Cylinder Tests

TEST NO.	Υ _d (g/cc)	D _R (%)	σ _{3c} (KSC)	MODE V-VERTICAL C-COMBINED	
B10-1	1.68	91.8	0.5	С	
B11-1	1.67	88.8	0.5	С	
B12-2	1.69	94.9	2.0	С	
B13-3	1.69	94.9	3.5	С	
AB15-2	1.71	100.0	2.0	V&C	
B16-2	1.69	94.9	2.0	С	
B17-3	1.69	94.9	3.5	С	
B18-1	1.70	97.9	0.5	С	
B19-2	1.68	91.8	2.0	С	
B20-3	1.67	88.8	3.5	С	
B21-1	1.67	88.8	0.5	С	
B22L-1	1.58	59.3	0.5	С	
B23L-2	1.58	59.3	2.0	С	
B24L-3	1.55	48.7	3.5	С	
A25L-1	1.59	62.7	0.5	v	
A26L-2	1.56	52.3	2.0	v	
A27L-3	1.55	48.7	3.5	v	
A28-3	1.69	94.9	3.5	v	
B29-3	1.68	91.8	3.5	С	
B30L-3	1.58	59.3	3.5	С	
B31-1	1.68	91.8	0.5	С	

 γ_d = dry density of soil

D_R = relative density

 σ_{3c} = lateral confining pressure

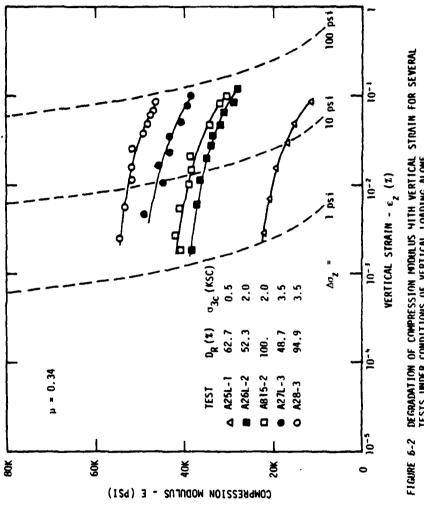


FIGURE 6-2 DEGRADATION OF COMPRESSION MODULUS WITH VERTICAL STRAIN FOR SEVERAL TESTS UNDER CONDITIONS OF VERTICAL LOADING ALONE

from approximately 1 psi to 100 psi. For low confining pressure tests particularly, these values of $\Delta\sigma_{\rm Z}$ led to significant principal stress ratios during cyclic loading. These principal stress ratios are summarized in Table 6-2. Not all of the values in Table 6-2 were actually reached in the test specimens. Figure 6-2 shows the combinations of $\sigma_{\rm 3c}$ and $\Delta\sigma_{\rm Z}$ actually attained during testing.

Vertical strain amplitudes achieved during this testing series were in the range of from approximately 2X10⁻³% to 2X10⁻¹%.

The compression moduli shown in Figure 6-2 were normalized against the moduli at a vertical normal strain of 5×10^{-3} %, as shown in Figure 6-3. As with the low strain resonant column testing program, the variation of relative compression modulus with strain amplitude appears to be primarily a function of the confining pressure, with a lower confining pressure corresponding to a greater rate of degradation.

Combined Loading

As with the low strain resonant column testing program, cyclic loading tests under conditions of simultaneous vertical and torsional loading were performed on a number of hollow cylinder samples. The specimens tested represented two different densities and three different confining pressures.

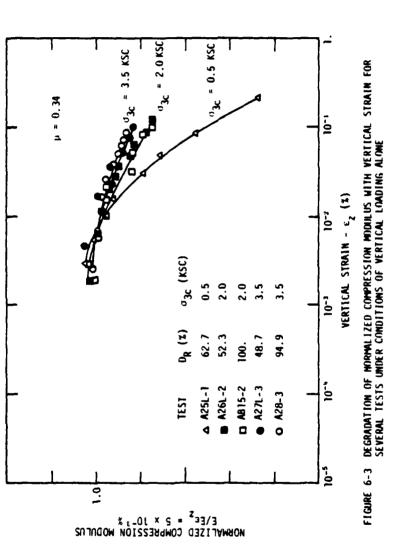
Strain Measurements

In this testing series the value of the relative rotational displacement of the top of the specimen (and thus the shear strain) was measured directly with an LVDT mounted on the side of the specimen. Although the

TABLE 6-2 ${\rm MAXIMUM\ PRINCIPAL\ STRESS\ RATIOS\ DURING\ CYCLIC\ LOADING\ -\ (\sigma_1/\sigma_3)_{max}}.$

$\Delta \sigma_{\mathbf{z}}$ (psi) (KSC)	0.5	2.0	3.5
0. (Consolidation)	1.85	1.85	1.85
20.	3.21	2.19	2.04
40.	4.57	2.53	2.24
60.	5.93	2.87	2.43
80.	7.29	3.21	2.63
100.	8.65	3.55	2.82
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relatively larger measurements were consistent, the relatively smaller shear strain measurements were somewhat erratic. This was found to be primarily due to the LVDT core friction, which was significant at very small displacements, causing the LVDT to "hang-up" and under-measure the true shear strain. This difficulty was not overcome by great care in setting up the LVDT to avoid core contact, because consolidation and dynamic straining quickly re-created the core friction problem.

The problem was further complicated by the fact that the calculated shear modulus values were extremely sensitive to the very small shear strains. A relatively minor variation in the measured shear strain could result in a large error in the calculated modulus, and would result in unreasonable predicted values for Poisson's ratio.

It was concluded that a more accurate estimate of the actual shear strain could be obtained using the measured stresses and vertical strain together with the elastic theory formulations developed in Chapters 2, 3, and 4.

The measured values of strain and stress provided a redundancy of data from which a Poisson's ratio could be computed. Because the shear strain is relatively insensitive to the value of the Poisson's ratio, the computed values of the Poisson's ratio are extremely sensitive to small errors in shear strain. By back-calculating values of the shear strain for various values of Poisson's ratio, a much more accurate evaluation of the shear strain may be made.

The measured values of the vertical and shear stress were obtained with calibrated differential pressure transducers attached to the loading cylinders, and are relatively accurate and consistent in determining those stress values. The vertical strain was measured with a proximeter

device, which accurately measures displacements without physical contact, avoiding the core friction problem. These data measurement devices are discussed in more detail in Appendix A-2.

When values of the shear strain were calculated from the measured stresses and vertical strain values, and for various values of Poisson's ratio, as described above, they were found to be in the same relative range as the measured values, but were generally slightly larger.

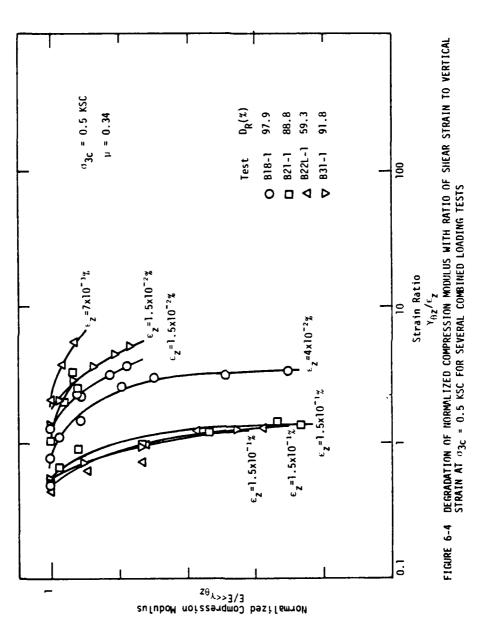
In the example of test results, presented in Appendices B-4 through B-6, both the measured and the calculated values of the shear strain are presented. The calculated values of the shear strain were used in developing the values of the shear modulus, G, and the strain ratios and octahedral shearing strains used in this chapter. The complete test results are reported elsewhere (Griffin, 1980).

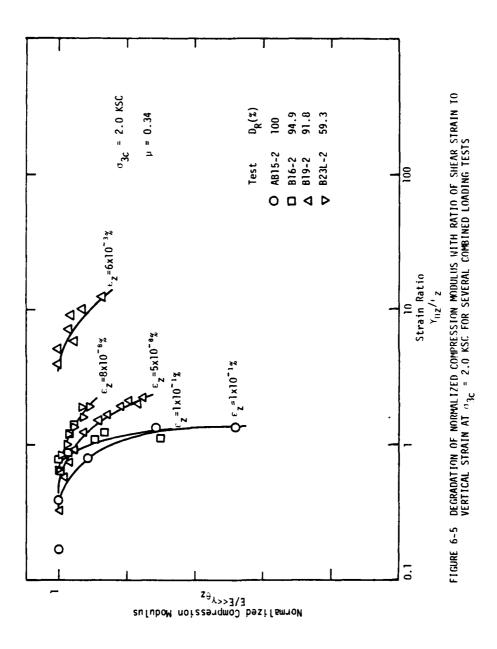
Strain Ratio Effects

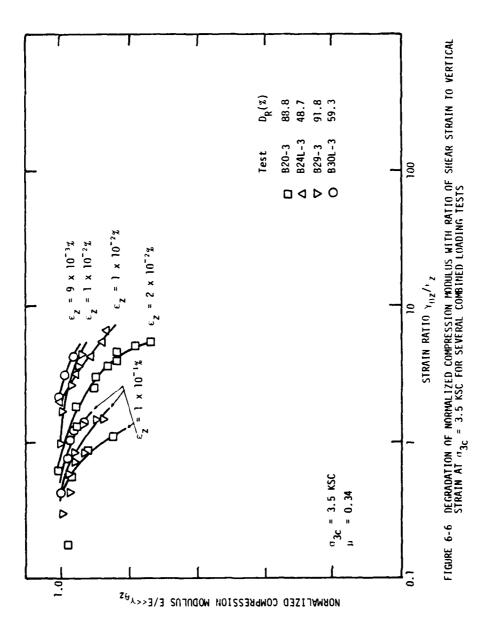
Normalized plots showing the degradation from the "virgin" modulus as a function of the ratio of shear strain to vertical normal strain were prepared for three different confining pressures. The resulting plots are shown in Figures 6-4, 6-5, and 6-6 for lateral confining pressures of 0.5, 2.0, and 3.5 KSC, respectively.

The plotted modulus reduction curves shown in Figures 6-4 through 6-6 are combined and summarized in Figure 6-7 for two values of vertical normal strain: 10^{-1} % and 10^{-2} %.

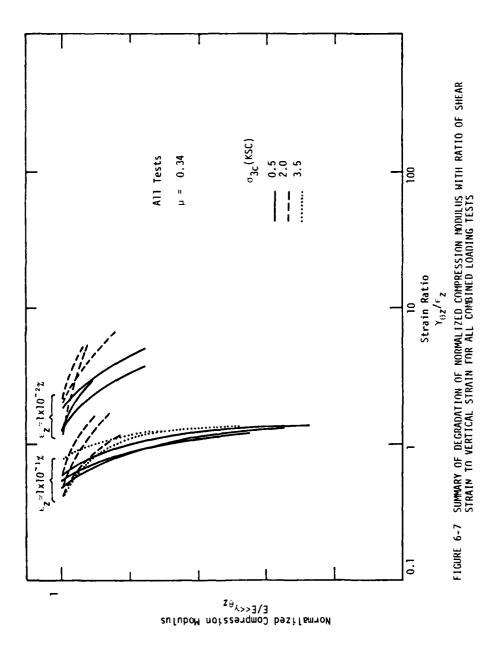
The effect of varying the Poisson's ratio is to vary the strain ratio slightly. This effect is shown in Figure 6-8.

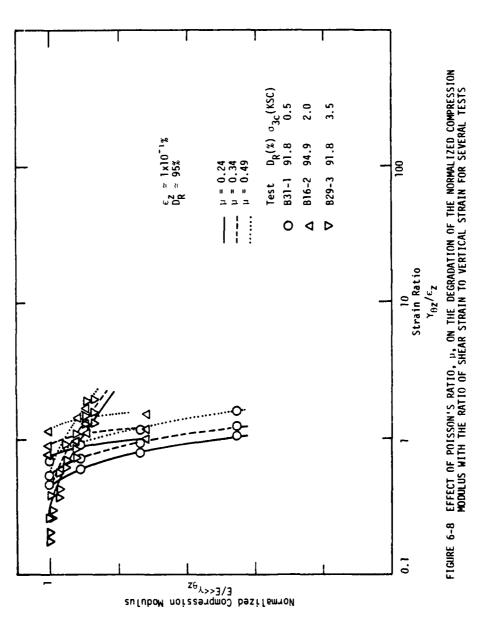






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Octahedral Shearing Strain Effects

As with the presentation in Chapter 5, the usefulness of the octahedral shearing strain in predicting compression-shear interaction was evaluated.

In Figure 6-9, the "virgin" normalized compression moduli curves are shown plotted against the octahedral shearing strain. For the data in these curves the principal strain directions are coincident with the vertical, radial, and tangential directions, and do not rotate during loading.

In the combined loading tests, the principal strain axes in the θz "plane" rotate continuously during loading, and the octahedral shearing strain arises from the combination of normal and shear strains in the vertical, radial, and tangential directions. The variation of the normalized compression modulus with octahedral shearing strain under combined loading conditions for a vertical normal strain amplitude, ε_z , of 0.1%, are presented in Figures 6-10 through 6-12 for lateral confining pressures of 0.5, 2.0, and 3.5 KSC, respectively. These data are combined for comparison purposes in Figure 6-13.

It is apparent from the data presented in Figure 6-12 that the modulus degradation curve for a vertical normal strain of 0.1% is influenced primarily by the octahedral shearing strain amplitude, and is essentially independent of the confining pressure. It is therefore possible to draw one best-fit curve through the data as shown in Figure 6-13.

Similarly, a best-fit curve was drawn through the data for a vertical normal strain of approximately 2×10^{-2} % as shown in Figure 6-14.

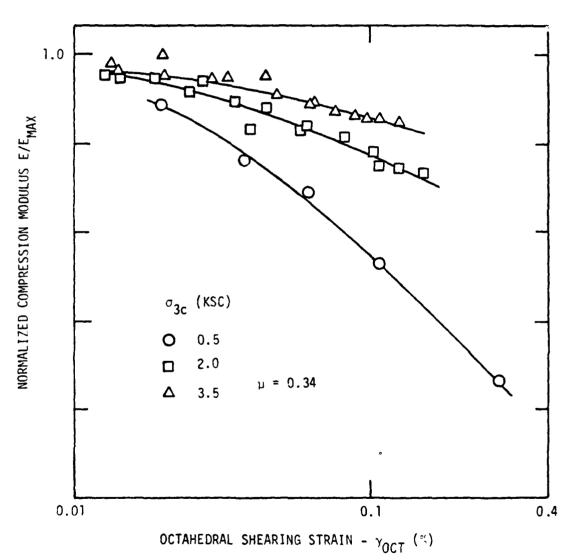


FIGURE 6-9 DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN FOR CONDITION OF VERTICAL LOADING ALONE

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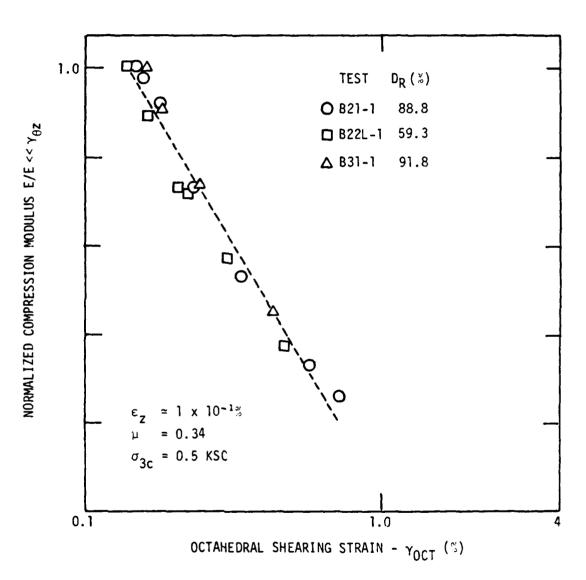


FIGURE 6-10 DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN AT σ_{3c} = 0.5 KSC FOR SEVERAL COMBINED LOADING TESTS WITH A CONSTANT VERTICAL STRAIN OF $10^{-1}\%$

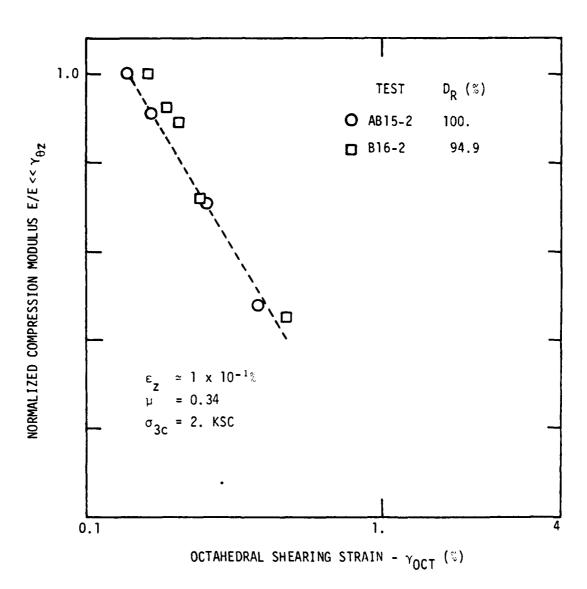


FIGURE 6-11 DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN AT σ_{3c} = 2.0 KSC FOR SEVERAL COMBINED LOADING TESTS WITH A CONSTANT VERTICAL STRAIN OF $10^{-1}\%$

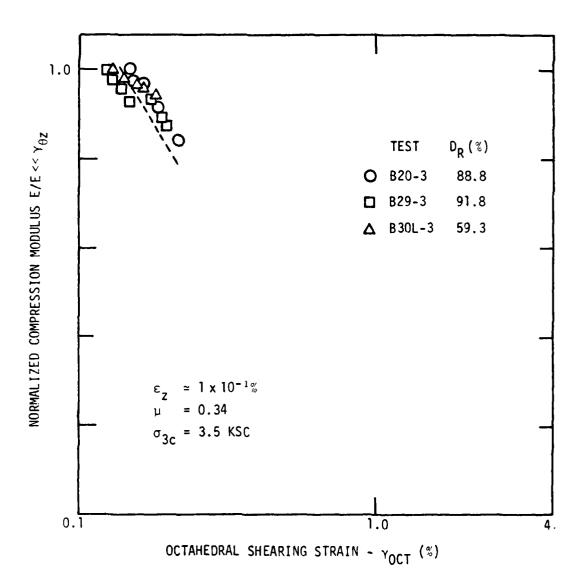


FIGURE 6-12 DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN AT σ_{3c} = 3.5 KSC FOR SEVERAL COMBINED LOADING TESTS WITH A CONSTANT VERTICAL STRAIN OF $10^{-1}\%$

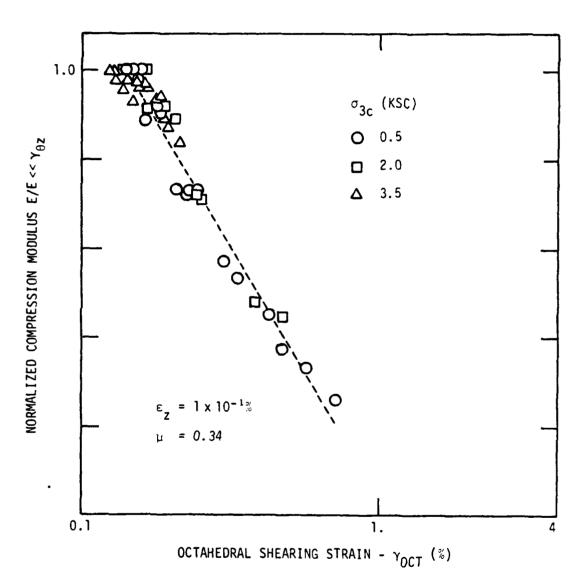


FIGURE 6-13 SUMMARY OF DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN FOR ALL COMBINED LOADING TESTS AT A CONSTANT VERTICAL STRAIN OF 10⁻¹%

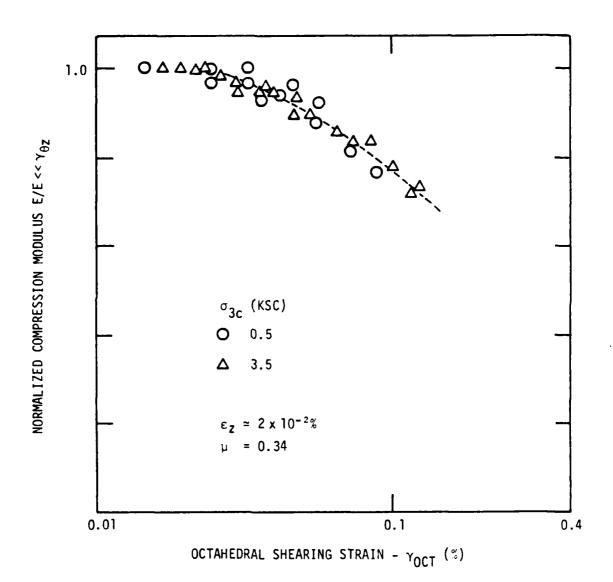


FIGURE 6-14 SUMMARY OF DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN FOR ALL COMBINED LOADING TESTS AT A CONSTANT VERTICAL STRAIN OF 2X10⁻²%

These relative moduli-octahedral shearing strain curves were then combined with the virgin curves presented in Figure 6-9 to obtain the curves presented in Figures 6-15 through 6-17 using the procedure described for Figures 5-16 through 5-18. These plots were then combined for comparison purposes in Figure 6-18.

Unlike the results of the low strain resonant column presentation in Figures 5-16, 5-17, and 5-18, the curves summarized in Figure 6-18 show a distinct variation depending on how $\gamma_{\rm OCT}$ was developed; e.i., from vertical compression or from shear straining. Of course part of the variation shown in Figure 6-18 is due to variations in confining pressure, which was illustrated in Figures 6-15 through 6-17. For low confining pressures, the modulus appears to degrade more rapidly as a result of dynamic normal vertical straining than shear straining. For higher confining pressures, the reverse appears true. Superimposed upon Figure 6-18 is an interpolated curve where it appears that the degradation of modulus is independent of the source of $\gamma_{\rm OCT}$. This curve has been labeled the "no interaction line," and appears at a lateral confining pressure of approximately 1.0 KSC.

To further summarize the results of this testing series, a set of best-fit curves have been presented in Figure 6-19 to show the degradation of normalized compression modulus with $\gamma_{\rm OCT}$ irrespective of the strain path. It is noted that the data varies from these curves to a greater degree than the comparable curves in Figure 5-19.

High Shear Strain Effects

Several combined loading tests were performed under conditions of

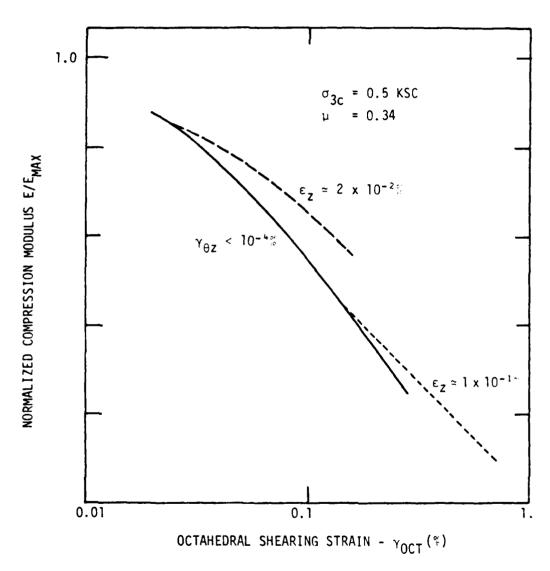


FIGURE 6-15 DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN AT σ_{3c} = 0.5 KSC FOR BOTH VERTICAL AND COMBINED LOADING TESTS

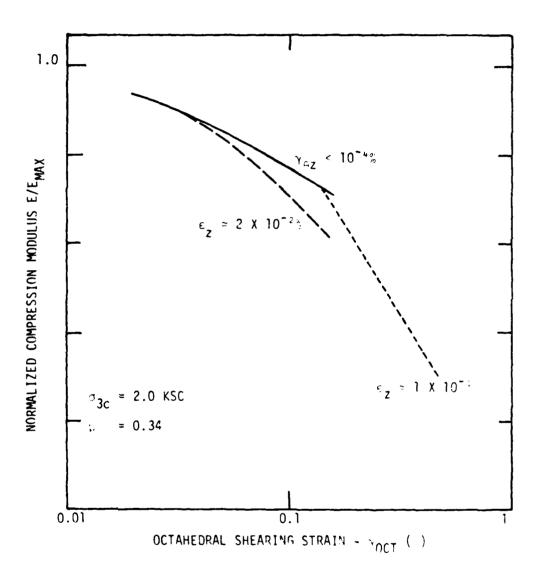


FIGURE 6-16 DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN AT 13c = 2.0 KSC FOR BOTH VERTICAL AND COMBINED LOADING TESTS

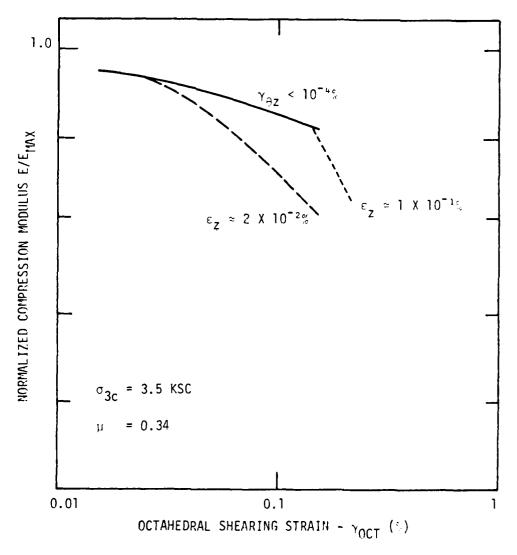


FIGURE 6-17 DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN AT σ_{3c} = 3.5 kSC FOR BOTH VERTICAL AND COMBINED LOADING TESTS

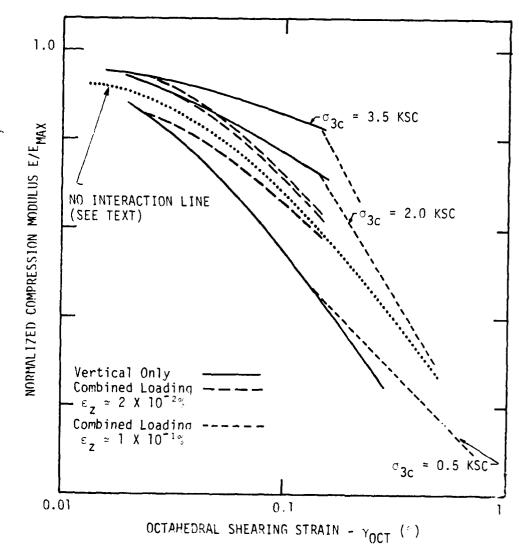


FIGURE 6-18 DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN FOR ALL VERTICAL AND COMBINED LOADING TESTS

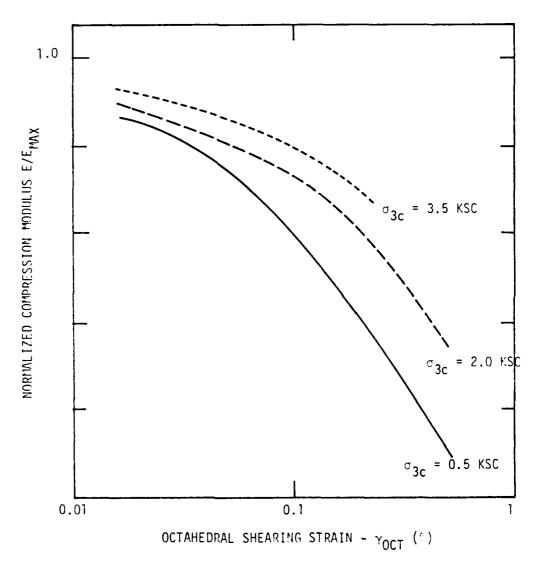


FIGURE 6-19 SUMMARY OF DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH OCTAHEDRAL SHEARING STRAIN

relatively very high shear strains, but it was very difficult, if not impossible, to determine accurate modulus values from these test results because of the shape of the hysteretical stress-strain plots. Several typical plots are shown in Figure 6-20.

The odd shapes of these hysteretic stress-strain plots provide a clue to the behavior of soils under these loading conditions. It is noted, for example, that several plots closely approximate a "Figure-8" shape, having two distinct loops. A Lissajous-figure analysis of a standing "Figure-8" would indicate an out-of-phase relationship, with the vertical axis at half the frequency of the horizontal axis. In the case of the hollow cylinder specimens it indicates that the soil sample is displaying two cycles of vertical strain for each cycle of vertical stress.

If we consider the case of a uniformly graded granular material with a relatively dense packing, the matrix of grains might look something like that shown in Figure 6-21(a). As large shear stresses are applied to the material in one direction, one grain would tend to "ride-over" another, resulting in a vertical displacement. As the shear stress reversed its direction in time, the grain would return essentially to its original packing position, then "ride-up" again upon another grain as shown in Figure 6-21(b). This effect would evidence as two cycles of vertical straining for each shear stress cycle, as shown in Figure 6-21(c).

In the case of relatively very high shear stress loading, the vertical straining resulting from this process might be large in comparison with the vertical strain induced by vertical loading. If this is the case, then a "Figure-8" stress-strain plot will result, as was observed in several tests. The other observed odd-shaped plots may be explained

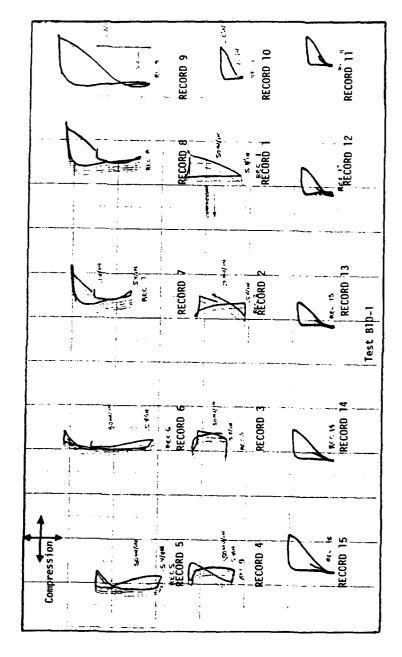


FIGURE 6-20 TYPICAL STRESS VS. STRAIN CURVES IN VERTICAL COMPRESSION UNDER CONDITION OF SIMULTANEOUS VERY HIGH RELATIVE SHEAR STRAIN

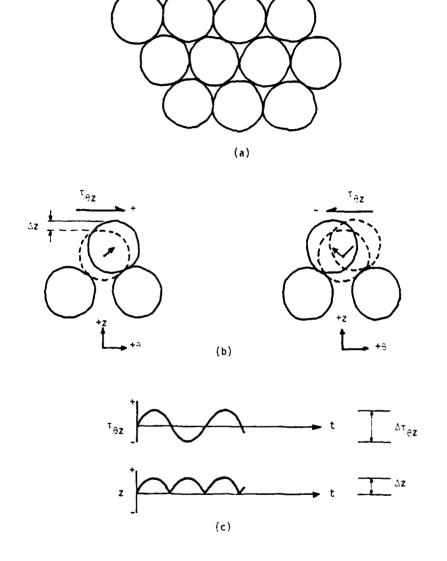


FIGURE 6-21 MATRIX OF A UNIFORMLY GRADED GRANULAR MATERIAL (a)
AND DETAIL OF VERTICAL NORMAL STRAIN RESULTING FROM
APPLICATION OF SHEAR STRESS (b) INCLUDING TIME HISTORIES (c)

by similar reasoning, considering phase effects and various combinations of vertical strain induced by vertical loading and shear loading.

It should be noted at this point that the consideration of a shear stress producing a normal strain is a coupling effect and is thus contrary to the assumptions of the applicability of elastic theory and Hooke's Law to this study.

This postulated theory of behavior is introduced only to aid in the understanding of soil behavior under conditions of very high shear straining, where inelastic, non-linear effects are apparent.

Conclusions

As in Chapter 5, the test results from this testing series were presented in different ways. The degradation of modulus was presented both as a function of the strain ratio, $\gamma_{\theta z}/\varepsilon_z$, and as a function of the octahedral shearing strain, γ_{OCT} . High shear strain effects were also explored.

Strain Ratio Effects

The result of the first presentation, summarized in Figure 6-6, indicates that there exists a relatively consistent threshold strain ratio below which interaction effects are negligible. In the case of the high vertical normal strain tests, where $\varepsilon_{\rm Z} \simeq 0.1$ %, the threshold strain ratio fell to as low as 0.5, indicating that shear strains below 0.05% would have negligible effect in further degradation of modulus.

As with the low strain resonant column testing series, the normalized modulus curves were found to be relatively independent of relative density

within the range tested but were moderately dependent upon confining pressure. The magnitude of the interaction effects appears to be greater at lower confining pressures. This effect may be explained by the fact that the principal stress ratios, $(\sigma_1/\sigma_3)_{\text{MAX}}$, during cyclic loading increase very rapidly at lower confining pressures as shown in Table 6-2; so that the soil material more closely approaches a failure condition in these tests. If the incremental modulus degrades more rapidly with higher stress increments, which was observed in the hysteretic stress-strain plots for these tests, then greater straining (and greater interaction) is to be expected.

Octahedral Shearing Strain Effects

The second presentation was of the degradation of modulus with the octahedral shearing strain, $\gamma_{\rm OCT}$, summarized in Figures 6-18 and 6-19. Unlike the low strain resonant column testing series results, there was some variation in the degradation curves depending upon how the $\gamma_{\rm OCT}$ was developed.

This variation is proportional to the interaction effect not accounted for by using γ_{OCT} . However, it was possible to select best-fit curves of normalized modulus vs. γ_{OCT} (Figure 6-19) for which the variations from the best-fit curves is quite acceptably small, except where γ_{OCT} becomes very large. When γ_{OCT} becomes very large and the confining pressure is relatively low, the assumptions on which the various derivations were made become invalid as discussed earlier.

Thus the accuracy of the normalized curves presented in Figure 6-19 decreases progressively with increasing $\gamma_{\rm OCT}$, as can be judged by com-

paring them with the curves of Figure 6-18 from which they were derived.

The highest octahedral shearing strain used in this testing series was approximately 0.5%.

High Shear Strain Effects

An analysis of soil behavior involving the modeling of uniformly graded granular soil as a collection of discrete particles was presented to help explain the observed soil response to high cyclic shear strains imposed simultaneously with relatively low amplitude vertical loading. The test data presented in Figure 6-20 indicated inelastic, non-linear response of the soil material; specifically, significant vertical normal strains were observed to result from the application of high horizontal shear stresses. When such coupling response is significant, elastic theory and Hooke's Law would not apply.

Chapter 7

Comparison of Laboratory Test Results

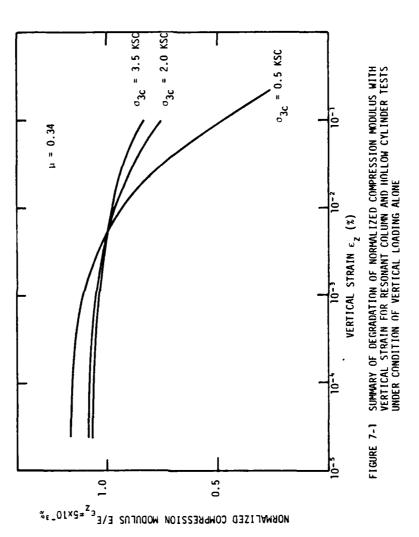
Introduction

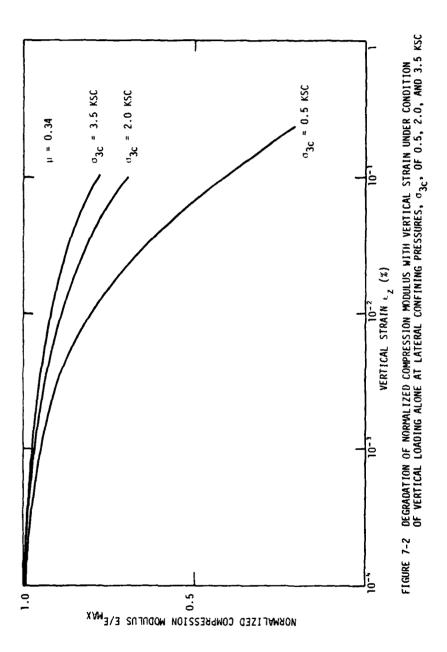
In both the triaxial resonant column testing series and the thin-walled hollow cylinder testing series, the effects of combined compression and shear loading were explored. The two testing series were conducted differently, producing different stress histories and different ranges of strain amplitudes. It is the purpose of this chapter to further develop the data presented from these two testing series in Chapters 5 and 6, and to develop a set of curves showing the loading effects over the full range of strains observed during testing.

Vertical Loading Alone

The compression modulus was plotted against the vertical strain under conditions of vertical compression loading alone in Figures 5-2 and 6-2. These figures were then normalized to the value of compression modulus at a vertical strain of $\varepsilon_z = 5 \times 10^{-3} \%$, with the resulting curves shown in Figures 5-3 and 6-3. The value of ε_z selected for normalization was chosen to maximumize the amount of overlap in the data and to provide a simple means of combining the two sets of results.

The data from Figures 5-3 and 6-3 were combined and presented in Figure 7-1. These curves were then replotted in Figure 7-2 normalized to the maximum compression modulus at a vertical strain of $\epsilon_2 = 10^{-5}$ %.





Mean Confining Stress

As indicated in these figures, the combined results were plotted for different values of lateral confining pressure, σ_{3c} . During the triaxial resonant column testing series, samples were consolidated in such a way that normal stresses in every direction were equal to σ_{3c} ; so that the mean confining stress during consolidation, σ_{m} , may be written:

$$\sigma_{\rm m} = \frac{\sigma_{\rm 1c} + \sigma_{\rm 2c} + \sigma_{\rm 3c}}{3} = \sigma_{\rm 3c} \tag{7.1}$$

With the thin-walled hollow cylinder test specimens, however, the principal stresses are not all equal during consolidation, but are as follows:

$$\sigma_{1c} = 1.85 \cdot \sigma_{3c} \quad , \tag{7.2}$$

$$\sigma_{2c} = \mu \cdot (\sigma_{1c} + \sigma_{3c}), \qquad (7.3)$$

and

$$\sigma_{3c} = \sigma_{3c}. \tag{7.4}$$

The mean confining stress during consolidation becomes:

$$\sigma_{\rm m} = \frac{\sigma_{\rm 1c} + \sigma_{\rm 2c} + \sigma_{\rm 3c}}{3} = \frac{(1 + \mu) \cdot 2.85 \cdot \sigma_{\rm 3c}}{3}$$
 (7.5)

And where μ = 0.34, σ_m may be written:

$$\sigma_{\rm m} = 1.273 \cdot \sigma_{\rm 3c} \tag{7.6}$$

Because the degradation of modulus with vertical strain for the two testing series may be better represented as a function of the mean confining stress than the lateral confining stress alone, it is desirable to replot the data from Figure 6-3 for values of $\sigma_{\rm m}$ of 0.5, 2.0, and 3.5 KSC.

In order to accomplish this replot, the variation of the normalized compression modulus was first plotted against the mean confining pressure, $\sigma_{\rm m}$, for a constant range of strain. This plot is shown in Figure 7-3, with the vertical strain range of from $\varepsilon_{\rm z}=5{\rm X}10^{-3}{\rm s}$ to $\varepsilon_{\rm z}=10^{-1}{\rm s}$. The circled data points in this figure come directly from the data in Figure 6-3, where $\sigma_{\rm m}$ is related to $\sigma_{\rm 3C}$ by Equation 7.6. The triangles represent interpolated data points for $\sigma_{\rm m}$ values of 0.5, 2.0, and 3.5 KSC.

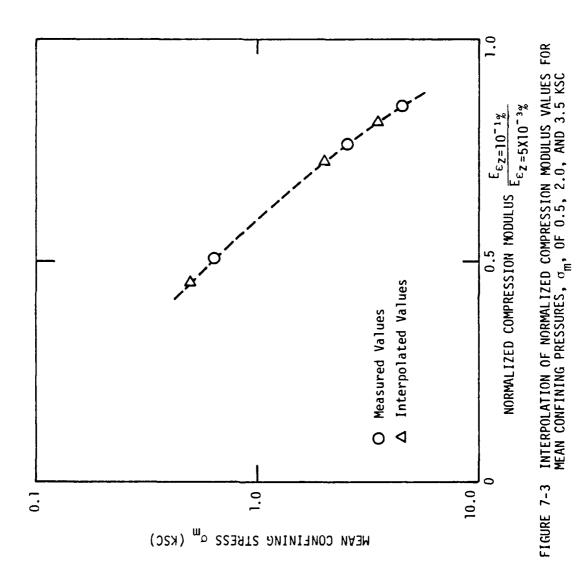
With the data represented by the triangles in Figure 7-3, it was possible to replot Figure 6-3 to reflect the desired values of $\sigma_{\rm m}$. This plot is shown in Figure 7-4.

From Figures 7-4 and 5-2 it was possible to replot the normalized compression modulus-vertical strain curves in Figure 7-2 in terms of the mean confining stress, $\sigma_{\rm m}$. This new plot is shown in Figure 7-5. It is noteworthy that the difference between this figure and Figure 7-2 is small. There is a somewhat more rapid degradation of modulus with increasing strain for the $\sigma_{\rm m}$ curves than for the comparable $\sigma_{\rm 3c}$ curves.

Octahedral Shearing Strain

In Chapter 5 it was noted that the normalized compression modulus degraded with the octohedral shearing strain relatively independently of how that strain was developed: e.i., whether from vertical loading alone or from combined vertical and torsional loading. It was concluded therefore, that for the resonant column data, compression-shear interaction could be completely accounted for by using the octahedral shearing strains.

In Chapter 6 some compression-shear interaction effects (for hollow cylinder specimens) were noted which were not completely accounted for by



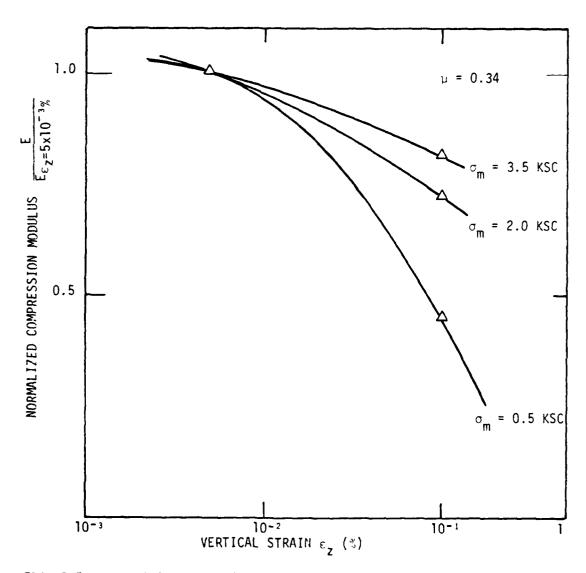
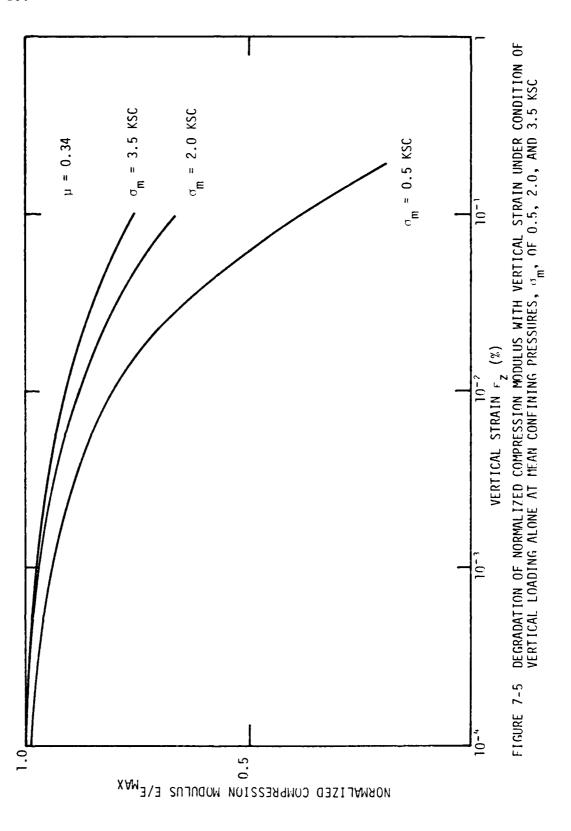


FIGURE 7-4 DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH VERTICAL STRAIN FOR HOLLOW CYLINDER TESTS UNDER CONDITION OF VERTICAL LOADING ALONE FOR MEAN CONFINING PRESSURE, $\sigma_{\rm m}$, OF 0.5, 2.0, AND 3.5 KSC



the use of the octahedral strain. Nevertheless, it was possible to develop a family of "best fit" octahedral strain curves which represent an excellent approximation of the variation of normalized compression modulus with octahedral shearing strain under a wide range of combined loading conditions.

The octahedral shearing strain curves were presented in Figures 5-19 and 6-19 for the two testing series. Using the technique described above to develop a series of curves for values of $\sigma_{\rm m}$ of 0.5, 2.0, and 3.5 KSC, these two figures were combined as shown in Figure 7-6. This figure may be used with reasonable accuracy to predict the degradation of modulus with octahedral shearing strain for Monterey No. 0 sand or a similar sand.

Strain Ratio Effects

Another way of evaluating the compression-shear interaction effects upon the degradation of modulus with strain was discussed in Chapters 5 and 6, and was referred to as the strain ratio effect. Simply stated, the method involved the use of the strain ratio degradation curves presented in Figures 5-10 and 6-7, where the degree of degradation was shown as a function of the ratio of shear strain to vertical normal strain for specific values of vertical normal strain. To predict the degradation of compression modulus under any straining condition, one would first determine the degradation resulting from the vertical straining alone; then that degradation would be further reduced by the reduction factor determined from the strain ratio figures.

The initial degradation of modulus resulting from vertical straining alone may now be obtained directly from Figure 7-5. To provide a summary

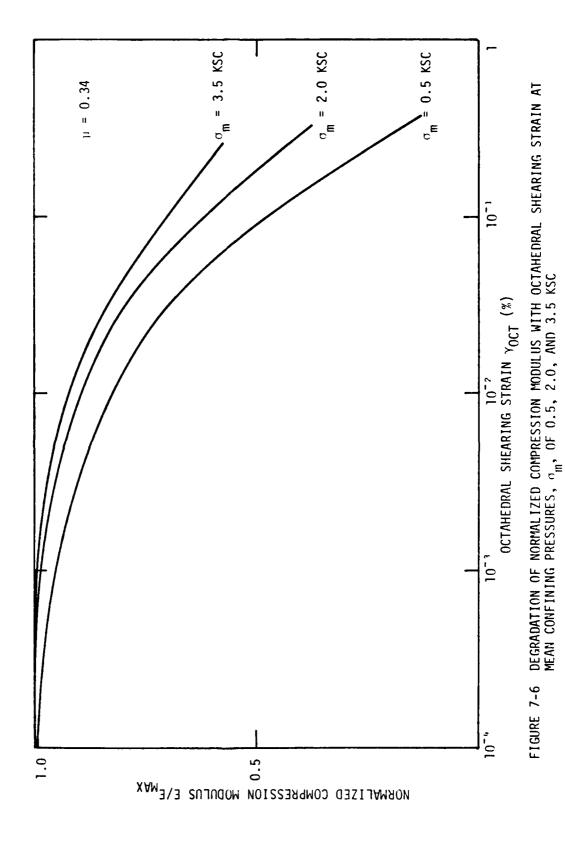


figure for obtaining the additional degradation resulting from the existence of simultaneous shear straining, a set of strain ratio curves were plotted for values of vertical strain of 10^{-1} , 10^{-2} , 10^{-3} , and 10^{-4} %. These curves appear in Figure 7-7.

Conclusions

The presentations made in Chapters 5 and 6 regarding the degradation of modulus with strain under combined compression-shear loading conditions were further developed to provide data curves over the entire strain range observed during the two testing series. These curves may be used to predict degradation of modulus under conditions of simultaneous compression-shear loading with reasonable accuracy.

Octahedral Shearing Strain Effects

The degree of degradation of modulus may be predicted by calculating the octahedral shear strain from the complete strain state, and interpolating from the curves in Figure 7-6. This degradation may then be applied to an estimate of the maximum, low-strain modulus to determine the modulus corresponding to a given state of strain. The maximum modulus value may be obtained either from a laboratory or field test, such as a seismic survey to obtain P-wave velocity, or from published data showing typical values of low-strain compression moduli at various mean confining stresses. The values of low-strain moduli observed in these testing series are shown in Figures 5-2 and 6-2, and are available in more detail elsewhere (Griffin, 1980).

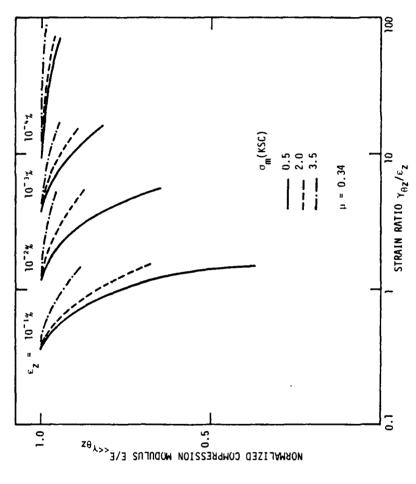


FIGURE 7-7 SUMMARY OF DEGRADATION OF NORMALIZED COMPRESSION MODULUS WITH RATIO OF SHEAR STRAIN TO NORMAL STRAIN

Strain Ratio Effects

The degree of degradation of modulus may also be estimated if the vertical normal strain and the horizontal-vertical shear strain are known. The degradation resulting from vertical straining alone is first determined from Figure 7-5, then the additional degradation resulting from shear straining is obtained by interpolating from the curves in Figure 7-7. Because of the difficulty in making the logarithmic interpolations in Figure 7-7 the octahedral shearing strain technique described earlier appears more promising for estimating compression modulus values for analysis purposes.

Chapter 8

Large Scale Model Studies

Introduction

It is the purpose of this chapter to present the observed soil response for a series of large scale dynamic model studies on slopes of sand. Because the significant response is primarily due to yielding within the soil slope, which is beyond the elastic range of loading for the soil material, conventional elastic response analysis techniques are of little value. An attempt will be made to predict the point at which yielding begins using an analysis technique proposed by Seed and Goodman (1964).

In earlier chapters, compression—shear interaction effects were studied under a variety of combined loading conditions on laboratory samples of sand. Interaction effects were presented in such a manner as to show their influence upon the dynamic moduli of the sands under various conditions of density, confining pressure, and strain amplitude. One major advantage of this form of presentation was that there exist a large number of elastic analysis techniques which employ the dynamic moduli to predict soil response, thus allowing for relatively simple application of these research findings in conventional elastic analysis methods.

In order to evaluate any interaction effects occurring beyond the elastic loading range of the soils, it was necessary to perform a soil loading test in which yielding occurs with simultaneous shear-compression dynamic excitation, and for which an accurate analysis technique has been developed. These slope model studies appeared to meet those criteria.

Test Set-up

The model studies were conducted on the 20 ft x 20 ft shaking table located at the Earthquake Simulator Laboratory at the Richmond Field Station, Earthquake Engineering Research Center, University of California, Berkeley, California. This table can move in one horizontal direction and the vertical direction. It was designed to exactly reproduce dynamic accelerations, velocities, and displacements, within certain limits, from analog records stored on either magnetic tape or disk. A detailed description of this shaking table has been provided by Rea (1972) and by Rea and Penzien (1972).

The cross-section of a typical soil slope specimen is shown in Figure 8-1. In this figure, four accelerometers and four DC linear variable displacement transformers (labeled DCDT) are shown. In addition to these, the table itself is heavily instrumented with accelerometers and displacement transducers.

Test Box

The inside dimensions of the test box in which the soil slope specimens are formed are approximately 84.2 in x 42.5 in x 21.7 in, with a triangular spacer in one corner approximately 15.8 in on each side, placed at an angle of approximately 45° with the horizontal. The test box was bolted securely to the shaking table, and a layer of high-strength "hydrostone" cement mortar was placed between the box and the table, and in the base of the box to assure rigid, solid contact between the two.

The test box is shown in Figure 8-2(a), and the bracing behind the

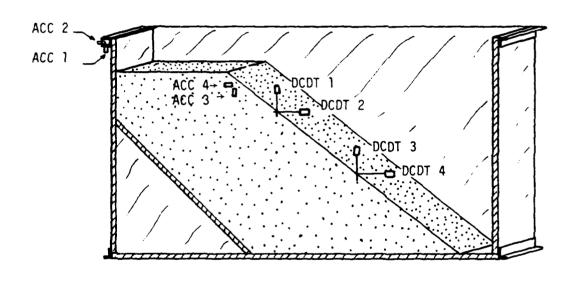


FIGURE 8-1 CUT-AWAY VIEW OF TYPICAL SLOPE SPECIMEN SHOWING LOCATION OF INSTRUMENTATION

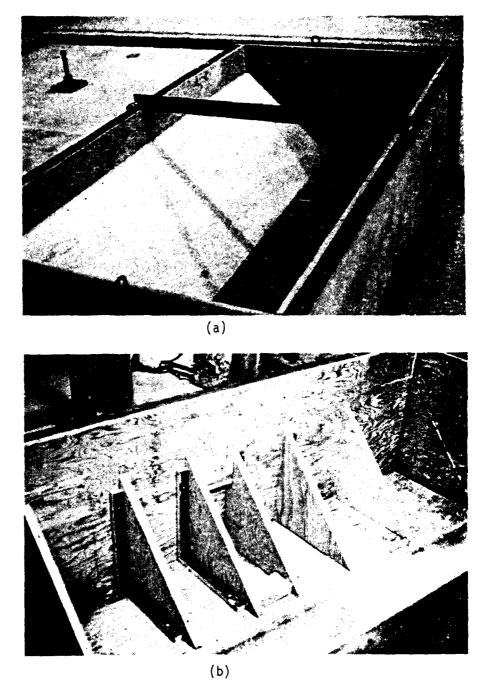


FIGURE 8-2 DETAIL OF TEST BOX (a) AND BRACING BEHIND SPACER BOARD (b)

spacer board is shown in Figure 8-2(b). The bracing was installed to insure a stiff response of the spacer board during loading, so that its deflection during loading would be very small when compared with the displacements of the soil slope.

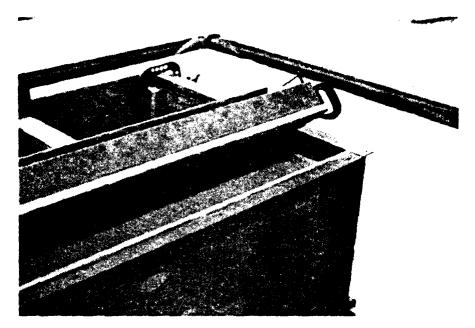
Before the test box was used for this testing series, a thin film of high-strength epoxy was sprayed upon the back, spacer board, and bottom, and a thin layer of the test sand placed upon those surfaces.

The sand used in this testing series was the same Monterey No. 0 sand used in the hollow cylinder and resonant column testing series.

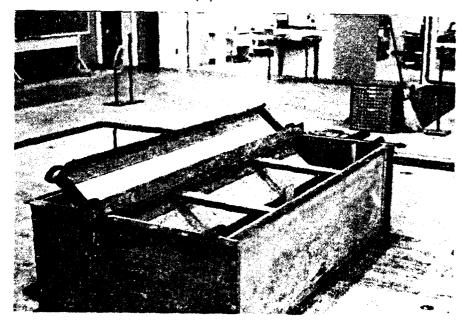
Forming Test Specimens

Slope specimens were formed by placing sand into a "mold" formed within the test box, and vibrating the sand, one lift at a time, until the desired slope height was obtained. Vibration was done by the shaking table. The forming "mold" assembly is shown in Figures 8-3(a) and (b).

A typical completed soil specimen is shown in Figures 8-4(a) and (b), with instrumentation in place. A closeup of the placement of the DCDT's on the soil slope surface is shown in Figure 8-5(a). As placed, the core of the DCDT's have a very thin (.01 in diameter), stiff 2 in length of piano wire protruding from them. This piano wire is coated with a very thin film of epoxy, and carefully inserted into the soil slope surface. The DCDT's are then zeroed by adjusting their position relative to their cores in the mounting bracket. Once the epoxy film hardens, the sand immediately surrounding the piano wire protrusion is cemented to the wire, but its fabric remains relatively undisturbed. A closeup of this wire, epoxy, sand cementation is shown in Figure 8-5(b).

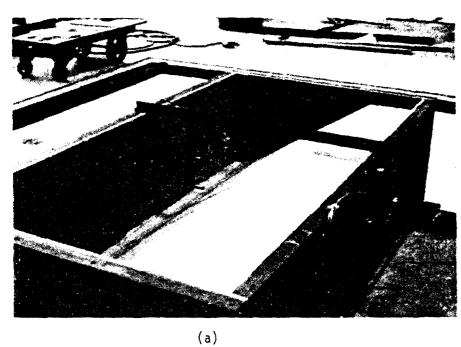


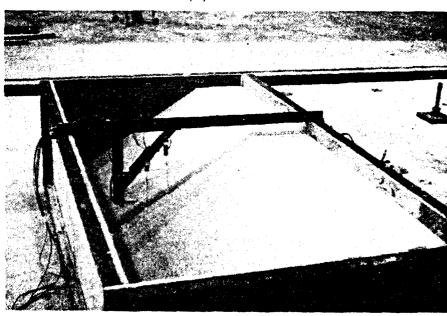
(a)



(b)

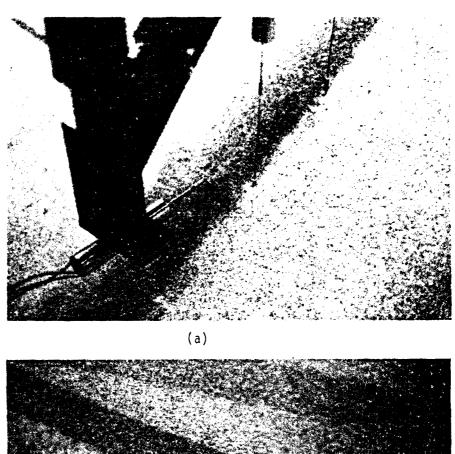
FIGURE 8-3 TWO VIEWS OF FORMING MOLD ASSEMBLY USED WHEN CONSTRUCTING SLOPE SPECIMENS





(b)

FIGURE 8-4 TWO VIEWS OF A COMPLETED, INSTRUMENTED SLOPE SPECIMEN READY TO BE TESTED





(b)

FIGURE 8-5 DETAIL OF PLACEMENT OF DCDT CORE PROBES ON THE SURFACE OF A SLOPE SPECIMEN (a), AND OF SAND-EPOXY BOND ON WIRE PERTUSION

Summary of Tests

The tests performed in this testing series are summarized in Table 8-1.

Soil Strength Characteristics

Because the yielding which occurs along the outer surface of the soil slopes is of primary importance to this study, it is important to know accurately the shear strength characteristics of the outermost soil slope materials.

The soil slopes were formed by vibrating soils within a forming mold, which was later removed. It is highly probable that passive pressures and other stress concentrations developed during sample formation resulted in non-uniform distribution of densities within the slopes. It is therefore necessary to determine the shear strength characteristics of the outermost soil materials by methods other than conventional shear-strength vs density correlations.

Static Slope Failure

In Test No. 9, shown in Table 8-1, a marginal static slope failure occurred soon after the slope was formed. This observed failure, which was a shallow sloughing failure, provides a probable upper bound for the angle of internal friction, ϕ , of:

$$\phi_{\text{MAX}} \simeq \alpha_{\text{failure}} = 37.6^{\circ}$$
 (8-1)

TABLE 8-1
Summary of Soil Slope Model Tests

Test No.	Test Code No.	α(°)	D _R (%)	a _v /a _h	ω(°)	Record	Mode (Horizontal or Combined)
1 2 3 4 4B 5 5A 6	291075.1 301075.1 311075.1 101175.3 120576.1 041175.1 100576.1	33.6 33.6 33.6 33.6 35.3 35.3	80 80 95 95 95 95 95 95	.67 .67 .67 .67 .67	33.7 33.7 33.7 33.7 33.7 33.7	A A A A A A A	н с с н н с с н
7 8 7x 8x 8y 8z 9	061176.1 101175.2 140576.1 170576.1 190576.1 250576.1 021175.*	35.3 35.3 35.3 35.3 35.3 35.3 37.6	80 80 80 80 80 80 80 95	.67 .67 .75 .60 (Stat:	33.7 33.7 36.9 31.0 ic Slope	A B B B B Failure)	н С С

where α is the angle of the slope,

 D_R is the relative density of the soil,

 a_{y} is the peak vertical acceleration,

 $\boldsymbol{a}_{\hat{h}}$ is the peak horizontal acceleration, and

 ω is the angle of application of the resultant acceleration.

Direct Shear Tests

During the testing series, two groups of special direct-shear tests were performed on the outermost soils of the slopes after they had been formed and were ready for testing. These direct-shear tests were performed by placing a lightweight block, specially prepared with a thin layer of sand epoxied to it, upon the surface of the slope. The block was then pulled upslope with a string until yielding occurred. As shown in Figure 8-6, if the weight of the block and the string tension are known, the normal and shear stresses on the failure surface at yielding are easily calculated.

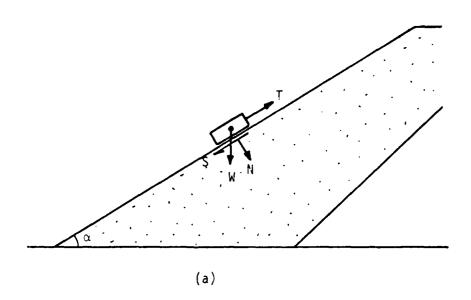
In Tables 8-2 and 8-3 the calculated values of N and S and the normal and shear stresses on the failure surface at yielding are shown for average relative densities of 95% and 80% respectively. From these tables, the shear strength plots shown in Figures 8-7 and 8-8 have been developed. From these figures it appears that the shear strength characteristics of the outermost slope materials are approximately the same even when the average soil densities are not, and are defined by the following equation:

$$S = s_{i} \cdot l + N \cdot tan\phi$$
 (8.2)

where s_i is the shear strength intercept, ℓ is the length, and φ is the angle of internal friction. Equation 8.2 may be rewritten as follows:

$$S = 0.029 \cdot l + N \cdot tan(35.4^{\circ})$$
 (8.3)

It is interesting to note at this point that the value of s_i is very small, and that the values of s_i and φ are much smaller than those used by Seed and Goodman in their studies involving Monterey sand. There are



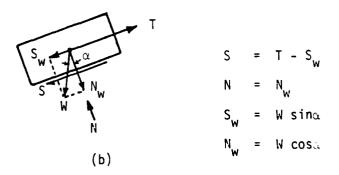


FIGURE 8-6 DIAGRAM OF FORCES ACTING ON THE SLIDING BLOCK USED IN SURFACE DIRECT SHEAR TESTS PERFORMED ON SLOPE SPECIMENS

TABLE 8-2 N and S for $D_R \simeq 95\%$

					 	
			W cosa	W sin α		į
			A	A		ļ
Test	α(°)	W(1b.)	N(psi)	-Sw(psi)	+ T(psi)	= S(psi)
						
060576.0	33.5	0.1552	.0032	.0021	.0040	.0019
		0.5847	.012	.008	.0164	.0084
		1.2339	.0254	.0168	.0333	.0165
		0.3849	.0079	.0052	.0098	.0046
		0.1552	.0032	.0021	.0047	.0026
		0.5847	.012	.008	.0166	.0086
		0.1552	.0032	.0021	.0041	.002
		0.5847	.012	.008	.0151	.0071
100576.1	35.3	0.1552	.0031	.0022	.0047	.0025
		0.3849	.0078	.0055	.0121	.0066
		0.5847	.0118	.0083	.018	.0097
		0.8144	.0164	.0116	.0249	.0133
		0.1552	.0031	.0022	.0046	.0024
		0.3849	.0078	.0055	.0112	.0057
		0.5847	.0118	.0083	.0179	.0096
		0.8144	.0164	.0116	.0257	.0141
		0.1552	.0031	.0022	.0044	.0022
		0.3849	.0078	.0055	.0105	.005
		0.5847	.0118	.0083	.0163	.008
		0.8144	.0164	.0116	.0238	.0122
		0.1552	.0031	.0022	.0043	.0021
		0.3849	.0078	.0055	.0106	.0051
		0.5847	.0118	.0083	.0156	.0073
		0.8144	.0164	.0116	.0216	.01
		0.3849	.0078	.0055	.011	.0055
		0.8144	.0164	.0116	.0238	.0122
110576.1	35.7	0.1552	.0031	.0022	.0047	.0025
		1.2339	.0247	.0178	.0339	.0161
		0.1552	.0031	.0022	.0043	.0021
		0.5847	.0117	.0084	.017	.0086
		1.2339	.0247	.0178	.0339	.0161
		2.7893	.0559	.0402	.0741	.0339
		1.2339	.0247	.0178	.0336	.0158
			•			
						

 α = angle of slope A = cross-sectional area of block

W = weight

T = tension in string

TABLE 8-3 $N \text{ and S for } D_{R} \cong 80 \text{ } \$$

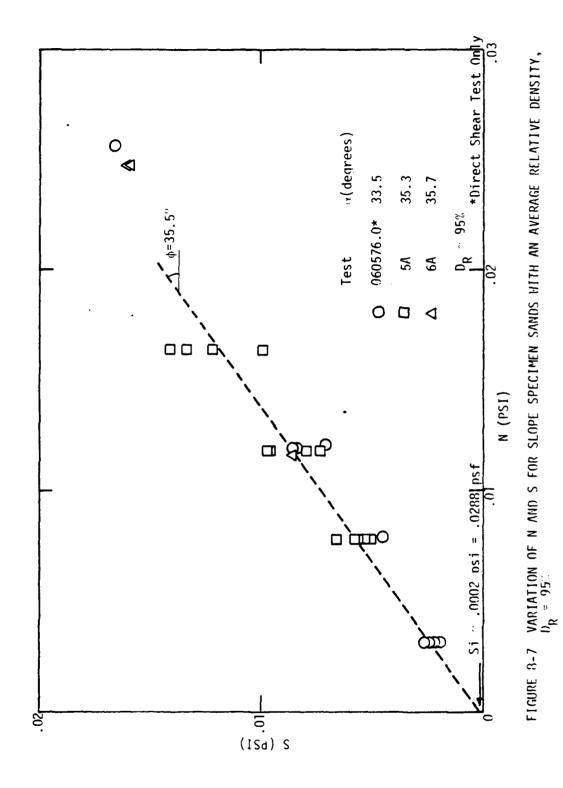
			W cosa	W sinα A		
Test	α(°)	W(lb.)	N (psi)	-S _w (psi)	+ T(psi)	= S(psi)
140576.1	35.2	.1552	.0031	.0022	.0044	.0022
		.5847	.0118	.0083	.0164	.0081
		.8045	.0162	.0115	.0226	.0111
1		1.234	.0249	.0176	.0345	.0169
ł		2.789	.0563	.0397	.0774	.0377
		.5847	.0118	.0083	.0162	.0079
		1.234	.0249	.0176	.0351	.0175
		2.789	.0563	.0397	.0765	.0368
}		.1522	.0031	.0022	.0044	.0022
		.5847	.0118	.0083	.0173	.0090
		.8045	.0162	.0115	.0234	.0119
		1.234	.0249	.0176	.0360	.0184
		2.789	.0563	.0397	.0785	.0388
170576.1	35.4	.1552	.0031	.0022	.0045	.0023
		.5847	.0118	.0083	.0163	.0080
(1.234	.0248	.0177	.0347	.0170
		2.789	.0561	.0399	.0761	.0362
		.8045	.0162	.0115	.0226	.0111
		.5847	.0118	.0083	.0166	.0083
		1.234	.0248	.0177	.0348	.0171
		2.789	.0561	.0399	.0812	.0413
		.1552	.0031	.0022	.0044	.0022
		.5847	.0118	.0083	.0170	.0087
		.8045 1.234	.0162	.0115	.0241	.0126
		2.789	.0248	.0177	.0351 .0797	.0174
		.5847	.0561 .0118	.0083	.0170	.0398
		1.234	.0248	.0177	.0170	.0181
				}		1
190576.1	35.3	.1552	.0031	.0022	.0045	.0023
		.5847	.0118	.0083	.0163	.0080
		.8045	.0162	.0115	.0239	.0124
		1.234	.0249	.0176	.0352	.0176
		2.789	.0562	.0398	.0812	.0414
		1.234	.0249	.0176	.0345	.0169
		2.789	.0562	.0398	.0788	.0390
		.1552	.0031	.0022	.0045	.0023
		.5847 .8045	.0118 .0162	.0083	.0164 .0222	.0081 .0107
		1.234	.0162	.0115	.0222	.0107
		2.789	.0562	.0398	.0344	.0354
		3.439	.0693	.0398	.0732	.0419
		J. 7JJ	.0055		.0310	.0417
				<u> </u>		

continued

TABLE 8-3 continued

N and S for $D_R \approx 80\%$

Test	α(°)	W(lb.)	$\frac{\text{W } \cos\alpha}{\text{A}}$ N (psi)	$\frac{\text{W sin}\alpha}{\text{A}}$ $-S_{\text{W}}(\text{psi})$	+ T(psi)	= S(psi)
250576.1	35.4	.1552 .5847 1.234 2.789 .1552 .5847 1.234 .1552 .5847 1.234 2.789	.0031 .0118 .0248 .0561 .0031 .0118 .0248 .0031 .0118 .0248	.0022 .0083 .0177 .0399 .0022 .0083 .0177 .0022 .0083 .0177	.0045 .0170 .0370 .0811 .0049 .0170 .0360 .0045 .0173 .0345	.0023 .0087 .0193 .0412 .0027 .0087 .0183 .0023 .0090 .0168



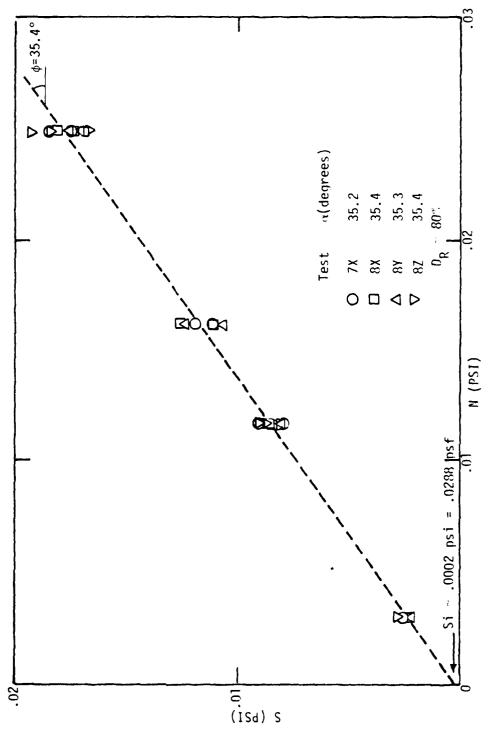


FIGURE 8-8 -VARIATION OF N AND S FOR SLOPE SPECIMEN SANDS WITH AN AVERAGE RELATIVE DENSITY, $D_{\rm R} = 80\%$

several reasons for this difference, one of which is the fact that different gradations of Monterey sand were used in the Seed and Goodman study than were used in this testing program. A gradation analysis of the Monterey No. 20 sand used by Seed and Goodman is shown alongside that for the Monterey No. 0 sand used in this study in Figure 8-9.

The low value of the shear strength intercept, s_i , is expected both because of the low mean grain size of the sand, and because the outer slope material is probably at lower density than the average slope material density because of arching and stress concentration effects during the sample formation process. The effect of mean grain size on the value of s_i is shown in Figure 8-10, as developed by Seed and Goodman. The mean grain size of Monterey No. 0 sand (.015 millimeters) would correspond to a very low value for s_i .

The larger degree of uniformity of the gradation analysis curve for Monterey No. 0 sand would support a lower friction angle, ϕ , for that sand.

Shaking Table Tests

As described earlier, slope specimens were subjected to sinusoidal dynamic loading in either the horizontal direction only, or in combined horizontal and vertical excitation. Combined horizontal and vertical loading was accomplished in phase, as illustrated in Figure 8-11. The resultant excitation which the samples experience is as shown in Figure 8-11(c), inclined at an angle ω with the horizontal plane. The two loading conditions are further illustrated in Figure 8-12. The horizontal component of loading was held constant for the different testing series, so that the resultant acceleration amplitude in the combined loading

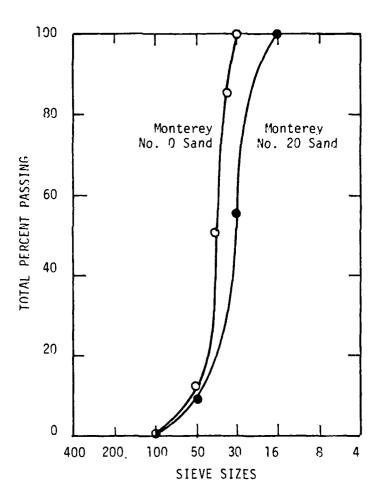


FIGURE 8-9 COMPARISON OF GRADATION ANALYSES FOR MONTEREY NO. 20 AND MONTEREY NO. 0 SANDS

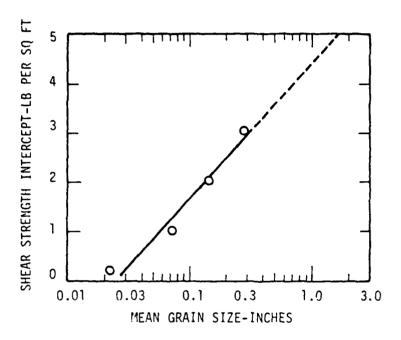


FIGURE 8-10 RELATIONSHIP BETWEEN SHEAR STRENGTH INTERCEPT AND MEAN GRAIN SIZE FOR UNIFORMLY GRADED SANDS

(AFTER SEED AND GOODMAN)

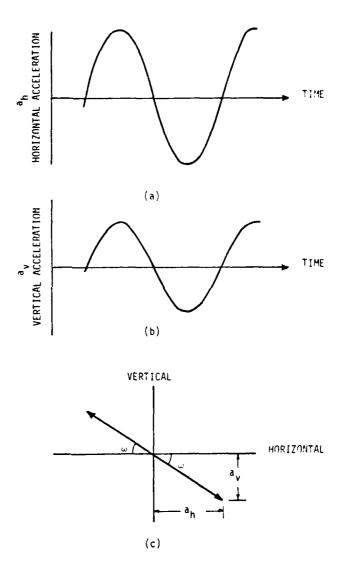
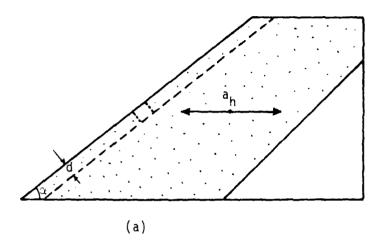


FIGURE 8-11 ILLUSTRATION OF IN-PHASE COMBINED HORIZONTAL AND VERTICAL LOADING USED IN SLOPE TESTS



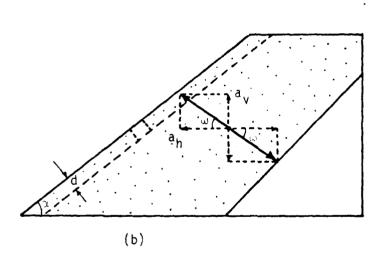
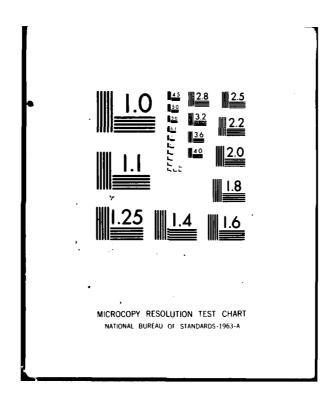


FIGURE 8-12 GRAPHICAL ILLUSTRATION OF APPLICATION OF HORIZONTAL ACCELERATION ALONE (a) AND COMBINED HORIZONTAL AND VERTICAL ACCELERATION (b) ON SLOPE SPECIMENS

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INTERACTION EFFECTS OF SIMULTANEOUS TORSIONAL AND COMPRESSIONAL.—ETC(U)
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tests was necessarily higher than in the horizontal loading tests.

Test Records

The two basic testing records utilized in this testing series were records A and B as shown in Figure 8-13. Record A consists of four acceleration amplitude steps of ten cycles each increasing in equal increments to a maximum acceleration amplitude of 0.35 g. Record B contains fifteen acceleration amplitude steps of five cycles each increasing in equal increments to a maximum acceleration amplitude of 0.175 g.

Analysis

In accordance with the analysis technique proposed by Seed and Goodman (1964), the sliding surface at which limiting equilibrium exists when the critical yield acceleration is applied is approximated as shown with the dashed line in Figure 8-12, at a depth d below the surface of the slope. The forces acting upon an element of soil along this sliding surface are illustrated in Figure 8-14(a). To satisfy equilibrium, $F_{\rm u}$ must equal $F_{\rm d}$ in this figure, and the remaining forces may be evaluated as follows:

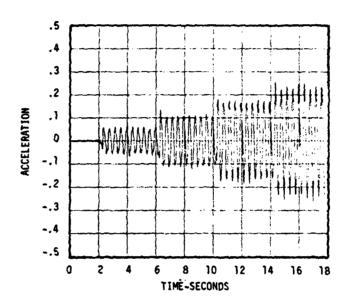
$$N = W \cdot \cos \alpha - ky \cdot W \cdot \sin (\alpha + \omega), \text{ and}$$
 (8.4)

$$S = W \cdot \sin\alpha + ky \cdot W \cdot \cos(\alpha + \omega)$$
 (8.5)

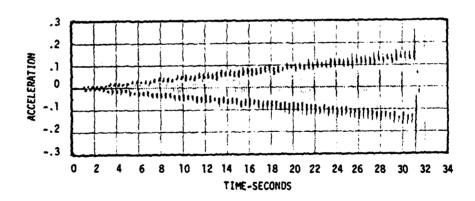
But, from Equation 8.2 we know that:

$$S = S_{i} \cdot l + N \cdot tan\phi$$
 (8.2)

Equations 8.4 and 8.5 may now be written:

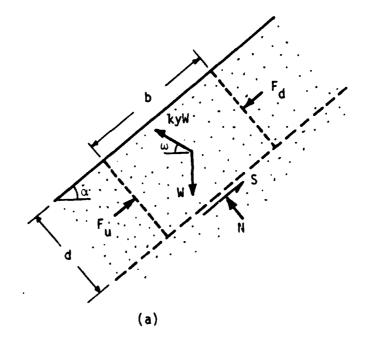


(a) RECORD A



(b) RECORD B

FIGURE 8-13 EXAMPLES OF TWO ACCELERATION RECORDS USED IN SLOPE TESTS



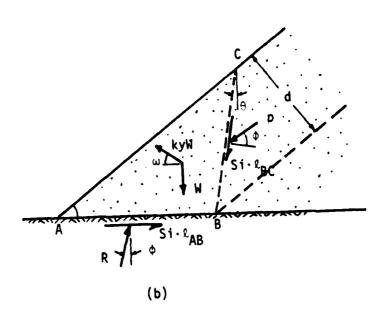


FIGURE 8-14 DIAGRAM OF FORCES ALONG SLIDING SURFACE AT LIMITING EQUILIBRIUM FOR SLOPE SPECIMENS

$$S = s_i \cdot b + [W \cdot \cos\alpha - ky \cdot W \cdot \sin(\alpha + \omega)] \cdot \tan\phi$$

$$= W \cdot \sin\alpha + [ky \cdot W \cdot \cos(\alpha + \omega)]$$
 (8.6)

Collecting terms and writing:

$$W = b \cdot d \cdot \gamma_d \tag{8.7}$$

it is found that:

$$ky = \frac{\frac{s_i}{d \cdot \gamma_d} + \cos\alpha \cdot \tan\phi - \sin\alpha}{\cos(\alpha + \omega) + \sin(\alpha + \omega) \cdot \tan\phi}$$
(8.8)

or:
$$ky = \frac{\sin(\phi - \alpha)}{\cos(\phi - \alpha - \omega)} + \frac{\sin(\alpha + \omega) \cdot \tan(\alpha + \omega)}{d \cdot \gamma_d \cdot [\cos(\alpha + \omega) + \sin(\alpha + \omega) \cdot \tan(\alpha + \omega)]}$$
(8.9)

For the slope of finite length used in this testing series, a passive wedge will form at the toe of the slope as shown in Figure 8-14(b). The primary effect of this passive wedge is to resist sliding not only by the shear resistance along the base of the sliding mass, S, but also by the force, P, at the toe of the slope.

Summing the forces perpendicular to the resultant force, R, gives the following expression:

$$[P \cdot \cos(2\phi - \theta)] + [ky \cdot W \cdot \cos(\phi - \omega)] =$$

$$W \cdot \sin\phi + [s_i \cdot l_{AB} \cdot \cos\phi] + [s_i \cdot l_{BC} \cdot \sin(\phi - \theta)] \qquad (8.10)$$

Noting that,

$$\ell_{AB} = \frac{d}{\sin \alpha}$$
 , and (8.11)

$$\ell_{BC} = \frac{d}{\cos(\alpha + \theta)} , \qquad (8.12)$$

and solving for P leads to the following:

$$P = \frac{W \cdot [\sin \phi - ky \cdot \cos(\phi - \omega)] + s_i \cdot d \cdot \left[\frac{\cos \phi}{\sin \alpha} + \frac{\sin(\phi - \theta)}{\cos(\alpha + \theta)} \right]}{\cos(2\phi - \theta)}$$
(8.13)

Writing
$$W = \frac{\gamma_d \cdot d \cdot \ell_{AC}}{2}$$
 (8.14)

where
$$l_{AC} = d \cdot [\tan \alpha + \tan (\alpha + \theta)]$$
 (8.15)

and combining and rewriting Equation 8.13, the following is obtained:

$$P = \frac{d^{2} \cdot \gamma_{d}}{2\cos(2\phi - \theta)} \cdot \left[\left\{ \tan\alpha + \tan(\alpha + \theta) \right\} \cdot \left\{ \sin\phi - ky \cdot \cos(\phi - \omega) \right\} + \frac{2s}{d \cdot \gamma_{d}} \cdot \cos\phi \cdot \left\{ \frac{1}{\sin\alpha} + \frac{\sin(\phi - \theta)}{\cos(\alpha + \theta) \cdot \cos\phi} \right\} \right]$$
(8.16)

As pointed out by Seed and Goodman, it is simply necessary to substitute the appropriate values of α , s_i , ϕ and γ , and a close approximation for ky, to obtain a relationship for P in terms of d and θ . By means of several trials, the values of θ giving the minimum value of P for different values of d can then be determined.

The influence of the force, P, can then be represented approximately by an equivalent shear resistance, s_{μ} , as follows:

$$s_{e} = P/L \tag{8.17}$$

where L is the length of the slope. This is analogous to increasing the shear strength intercept, s_i , to a value of $s_i + s_e$, so that the corresponding value of ky is given by:

$$ky = \frac{\sin(\phi - \alpha)}{\cos(\phi - \alpha - \omega)} + \frac{\sin^2(\cos(\alpha + \omega) + \sin(\alpha + \omega) \cdot \tan\phi)}{d \cdot \gamma_d \cdot [\cos(\alpha + \omega) + \sin(\alpha + \omega) \cdot \tan\phi]}$$
(8.18)

By means of several trials, the value of d corresponding to the minimum ky may be determined.

The equations presented by Seed and Goodman differ from those above only in that their presentation dealt with the condition of horizontal accelerations only, whereas these equations were developed for combined vertical and horizontal accelerations. The vertical component of acceleration appears in Equations 8.16 and 8.18 in the form of the angle ω .

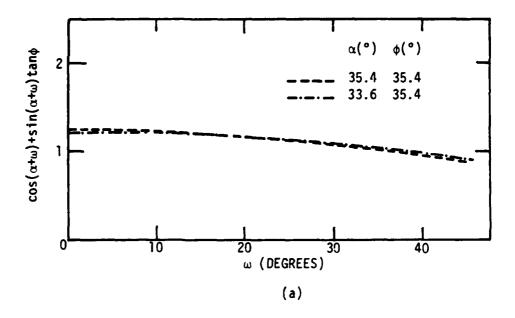
The influence of the angle, ω , may be seen in its effect upon the value of ky in Equation 8.18. The influence of the angle, ω , upon the first term of that equation, and upon the denominator of the second term is illustrated graphically in Figure 8-15. It is apparent from this figure that ky does not vary greatly as a function of the angle, ω , for the soil and slope characteristics of the current testing program.

For these slope specimens, ky at a value of ω = 33.7° will be approximately 1.2 times the value at ω = 0. This fact is of value to this analysis, as it allows the use of the simplified analysis technique formulated by Seed and Goodman to predict yield accelerations for the current testing program. This eliminates the need to perform the trials and iterations as described earlier.

Simplified Analysis

In the simplified analysis proposed by Seed and Goodman, the yield acceleration is calculated as:

$$ky = tan(\phi - \alpha + \phi_{SL})$$
 (8.19)



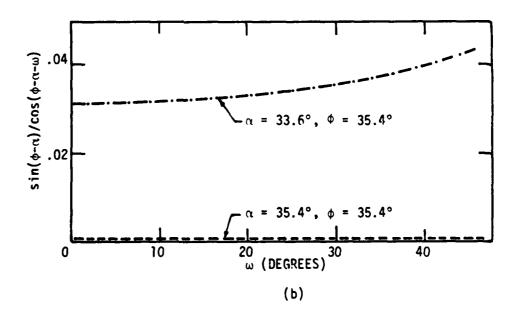


FIGURE 8-15 VARIATION OF COMPONENTS OF EQUATION 8.18 WITH THE ANGLE, ω , OF APPLICATION OF YIELD ACCELERATION, ky

where $\phi_{\rm SL}$ is a function only of the shear strength intercept, $s_{\rm i}$, and the slope length, ℓ . The values suggested for $\phi_{\rm SL}$ are presented in Figure 8-16. For the current testing program, $s_{\rm i}$ is approximately equal to 0.029 psf (from Figures 8-7 and 8-8), and the slopes are approximately 3 ft long.

As an example of how this simplified procedure may be used, consider the case of a slope with α = 33.6° and ω = 36.9°. From Figure 8-16, with $s_i \simeq 0.03$ and $\ell \simeq 3$ ft, $\phi_{SL} \simeq 0.8$ °. Now, from Equation 8.19,

$$ky = tan(35.4^{\circ} - 33.6^{\circ} + 0.8^{\circ})$$

$$= tan(2.6^{\circ}) = 0.045$$
(8.20)

This value of ky is that for $\omega=0$. To obtain the value of ky when $\omega=36.9^{\circ}$, reference is made to Figure 8-15. The value of the first term in Equation 8.18, illustrated in Figure 8-15(b), is approximately 1.22 times the value when $\omega=0$. Similarly, the value of the second term in Equation 8.18, whose denominator is illustrated in Figure 8-15(a), is also approximately 1.22 times the value when $\omega=0$. The value of ky from Equation 8-30 must therefore be multiplied by 1.22 to obtain the value when $\omega=36.9^{\circ}$. This new value is as follows:

$$ky_{(\omega=36.9^{\circ})} = (.045) \cdot (1.22) = .055$$
 (8.21)

Predicted Yield Accelerations

This computation was performed for the various test conditions encountered during this testing series, and a summary of the predicted yield acceleration values are presented in Table 8-4.

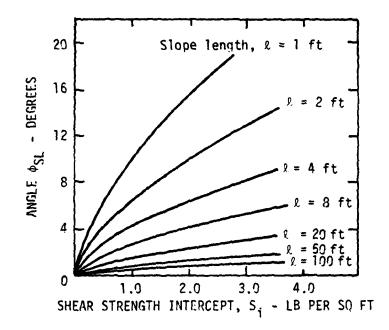


FIGURE 8-16 RELATIONSHIP BETWEEN SLOPE LENGTH, SHEAR STRENGTH INTERCEPT, AND ANGLE ϕ_{SL} (AFTER SEED AND GOODMAN)

and the second s

TABLE 8-4 Predicted Values of Yield Acceleration, ky, for Various Test Conditions

	Slope A		
Angle ω	33.6°	35.4°	
0°	.045	.016	
33.7°	.054	.019	
36.9°	.055	.020	(TEST 8)
31°	.052	.018	(TEST 82

Y)

Z)

Test Results

As discussed earlier, test results consisted of digitized records of accelerations and displacements measured during the course of dynamic loading. For each test, four accelerometers and four DCDT's measured the response of the slope specimen and test box to the base excitation, and numerous accelerometers mounted upon the table measured the average horizontal and vertical table acceleration during loading. These digitized records are stored on magnetic tape, and are easily processed with the aid of a computer.

Presented in Figures 8-17 and 8-18 are several typical acceleration time histories which have been plotted from these digitized test results. In Figure 8-17 are several A records showing both table and top of slope accelerations in both the horizontal and vertical directions. Note how closely the top of slope acceleration records mirror the average table acceleration records. Similarly, Figure 8-18 includes several B records showing table and top of slope accelerograms for horizontal and vertical loading. Some high-frequency noise appears to have been "picked-up" in the top of slope response shown in these results.

The plotted time histories shown in these and the following figures were drawn by the CALCOMP plotter at the University of California Computing Center, Berkeley. A complete set of CALCOMP plots of results of this testing program are on file along with the digitized records at the Geotechnical Engineering Office at Berkeley.

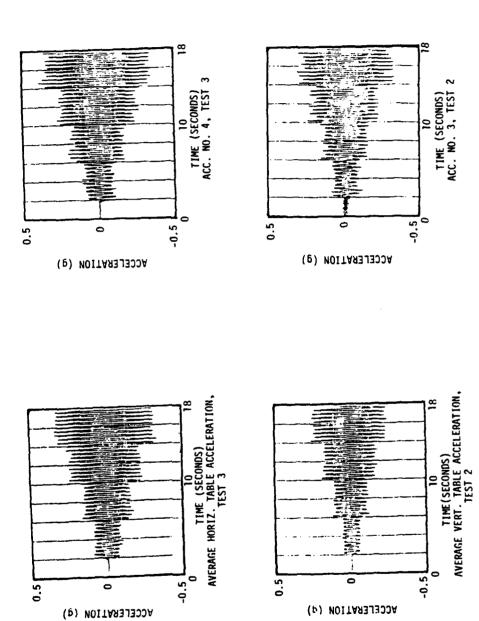


FIGURE 8-17 ACCELERATION TEST RECORDS FOR TESTS 2 AND 3

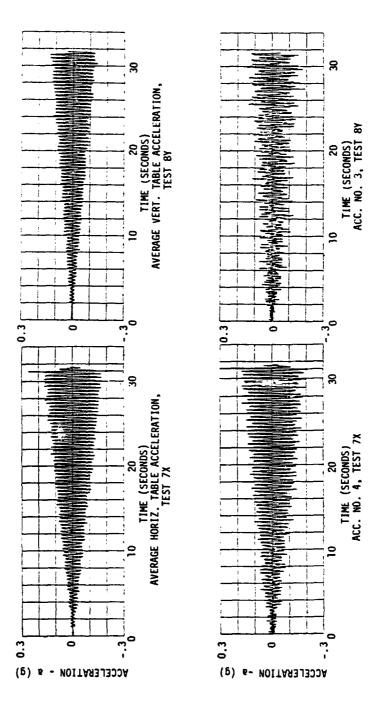


FIGURE 8-18 ACCELERATION TEST RECORDS FOR TESTS 7X AND 8Y

Data Reduction

To obtain estimated values of the yield acceleration from the acceleration and displacement records, it is necessary to carefully study the displacement records to determine the time in the record when yielding first began.

Figures 8-19 and 8-20 include several typical acceleration and displacement records for tests 1 and 4 respectively. In Figure 8-19 yielding appears to begin at approximately 2 seconds into DCDT record No. 2, near the top of the slope, and some time later in DCDT record No. 3, near the bottom of the slope. At 2 seconds, the average horizontal table acceleration is changing from 0 g to approximately .085 g, and accelerometer No. 4 at the top of the slope is changing from .015 g to approximately 0.1 g. Although the .015 g acceleration before 2 seconds is essentially all high frequency noise, it is being applied upon the sample and is therefore a lower bound on the yield acceleration. Because yielding was first observed at 2 seconds, the yield acceleration for this slope must fall somewhere between these acceleration values.

In Figure 8-20, yielding may be seen at 2 seconds into DCDT records 1 and 4, at both the top and bottom of the slope. At this time the lower bound values of the yield acceleration are 0 g and .018 g, and the upper bound values are .106 g and .118 g for the table acceleration and top of slope acceleration respectively. These accelerations are obtained by noting the vertical (a_v) and horizontal (a_h) peak acceleration values and, because they are in phase, calculating the resultant acceleration, a', as follows:

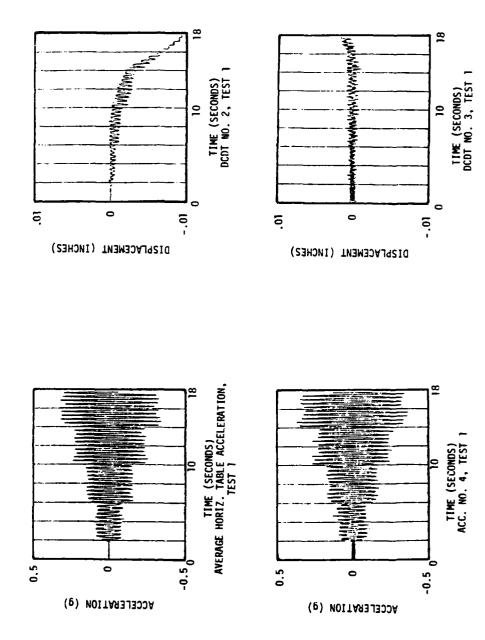


FIGURE 8-19 ACCELERATION AND DCDT TEST RECORDS FOR TEST NO. 1

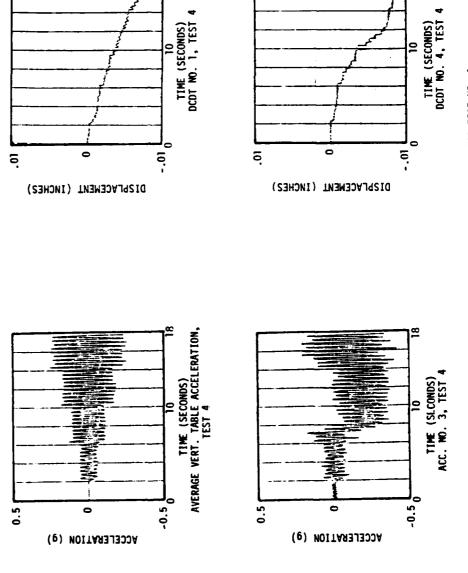


FIGURE 8-20 ACCELERATION AND DCDT TEST RECORDS FOR TEST NO. 4

$$a' = \sqrt{a_h^2 + a_v^2}$$
 (8.22)

This value of a' is then the lower or upper bound on ky for the slope.

In the example shown in Figure 8-20, the peak vertical table acceleration shown after 2 seconds is approximately 0.056 g. The peak horizontal table acceleration occurring at the same time, which is not shown in the figure, is approximately 0.09 g. The resultant acceleration is:

$$a' = \sqrt{(.056)^2 + (.09)^2} = 0.106$$
 (8.23)

This acceleration is an upper bound on the yield acceleration measured from the bottom of the slope.

It is interesting to note in Figure 8-20 that acceleration record No. 3 shows a marked shifting of center acceleration beginning at approximately seven seconds into the record. This occurred because the soil around this accelerometer began to move significantly at this time and the "plane" of the accelerometer rotated. The accelerometer was originally oriented vertically, so that the center axis was at a value of 1 g. If the center axis moves by .2 g, as it appears to at eight seconds, then the new center axis becomes 0.8 g. The accelerometer has been reoriented by the angle θ from the vertical, where θ is defined by the following expression

$$\theta = \arccos(0.8) = 36.9^{\circ}$$
 (8.24)

Yield Accelerations

The computations described above for determining the upper and lower bounds on the yield acceleration, ky, were performed for all tests in this testing series. The results are tabulated and compared with the

predicted values of ky in Table 8-5. It is noteworthy that the predicted values of ky generally fall within the bounds presented in that table. It should also be noted that for all but two of the tests, yielding was observed within 2 to 3 seconds into the record, as the first increment of acceleration was applied.

The observed values of ky show two things:

- There is often a significant difference between the amplitude of the accelerations at the bottom of the slope (the table) and the top of the slope, due primarily to amplification of the accelerations; and
- 2. There is usually a significant difference between the observed upper bound and observed lower bound, due primarily to the fact that the increments by which the acceleration amplitudes were increased were relatively large: typically larger than ky itself.
 This latter effect was minimized somewhat in the B record tests because a smaller increment of increased acceleration amplitude was used.

Due to these facts and the general technical difficulties of making accurate measurements of accelerations, it is difficult to obtain highly precise values of the observed ky values. However, the comparisons shown in Table 8-5 serve at least to support the conclusion that no gross errors were made in computing the predicted ky values. Furthermore, as a corollary, it may be reasonable to conclude that ky can be computed as accurately as it can be directly measured.

One of the objectives of this research was to investigate the validity of direct vectorial superposition of the horizontal and vertical acceleration in arriving at the yield acceleration. Because of the

TABLE 8-5 Summary of Yield Accelerations

			ky Lower	ky Lower Bound (9)	Predicted ky (g)	ky Upper	ky Upper Bound (g)
Test No.	Time* (sec)	(sec)	тор	Bottom		Top	Bottom
1	2		.015	0	.045	٦.	.085
2	2		.038	0	.054	911.	.101
٣	3		0	0	.045	560.	*00
4	2		.018	0	.054	.118	106
4B	9		.129	٦.	.054	.222	.198
ĸ	2		0	0	910.	.085	.085
5 A	7		• 05	0	910.	٦.	60.
9	2		.016	.014	.019	.122	.103
6A	1		.052	0	610.	.148	104
7	2		0	0	.016	560.	.085
6 0	2		.018	0	610.	.127	.097
7X	8		90.	.035	910.	.07	.047
8X	2		.037	0	610.	.058	.019
Ж	2		.023	0	.020	.055	.021
Z8	2		.043	С	.018	950.	.018

*"Time" refers to the time in a test record where yielding was first observed.

difficulty in pinpointing precise values of the observed ky values, the validity of this superposition cannot be verified conclusively. However, the experimental evidence is at least consistent with the validity of this superposition.

It should also be noted that the calculated (predicted) value of ky is quite sensitive to the near-surface ϕ value, which is not easily measured with great accuracy. In view of this fact, it appears in retrospect that any error introduced by assuming validity of vectorial addition of the accelerations is much smaller than the error introduced by using imprecise ϕ values in the computations. Furthermore, it appears that any error incurred by direct vectorial superposition is smaller than that which can be detected using an experimental program similar to that described herein.

In view of these conclusions it is recommended that direct superposition of simultaneous horizontal and vertical accelerations be made as outlined in this chapter if practical cases should arise where yield accelerations must be predicted.

Qualitative Observations

In addition to the quantitative tabulations presented in Table 8-5, it is of value to study the test records and qualitatively evaluate the performance of the slopes during dynamic loading.

The four DCDT test records for Test No. 3 are presented in Figure 8-21. Referring to the positioning of these instruments upon the slope which was shown in Figure 8-1; it is seen that DCDT's 1 and 2 are located in the upper portion of the slope, and DCDT's 3 and 4 are in the

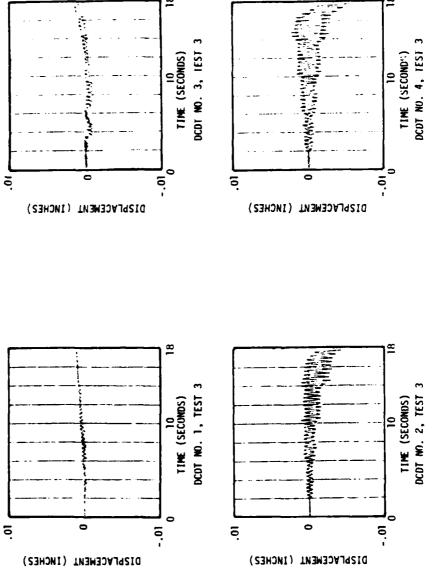


FIGURE 8-21 DCDT TEST RECORDS FOR TEST NO. 3

lower half. DCDT's 1 and 3 are vertically oriented, and DCDT's 2 and 4 are horizontal.

The two horizontal DCDT's reflect the acceleration record for Test

No. 3 which was shown in Figure 8-17. Although yielding may be seen from
the very beginning of these records, large permanent displacements are not
seen until late in the record. This is because yielding may occur during
only a small portion of each cycle of loading, and displacements do not
accumulate to significant values until the larger accelerations occur.

Shown in Figures 8-22 through 8-25 are the DCDT test records for the four B record tests: 7X, 8X, 8Y, and 8Z. As was shown in Table 8-1, the only differences between these tests was in the value of the vertical component of acceleration. The vertical acceleration was zero for Test 7X, was 0.67 of the horizontal for Test 8X, etc. Because the horizontal component remained unchanged, the resultant acceleration amplitude steps were different for the different tests.

The DCDT test records for Test 7X shown in Figure 8-22 show considerably less permanent displacement than the other three tests, as would be expected. The DCDT test records for Tests 8X, 8Y, and 8Z show remarkable similarity, both in their shapes and in the amplitudes of the displacements. Theoretically, the displacements should be slightly greater for Test 8Y, and less for Test 8Z, since the vertical accelerations are slightly higher and lower for those tests. It is difficult to distinguish significant variation for these records, however.

Additional Observations

It should be emphasized that the placement of DCDT's on the face of

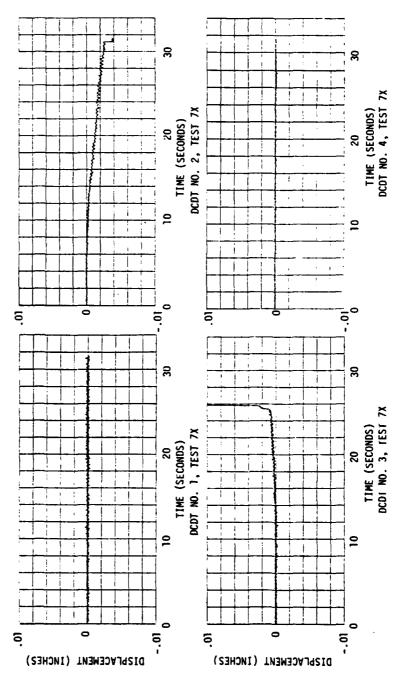
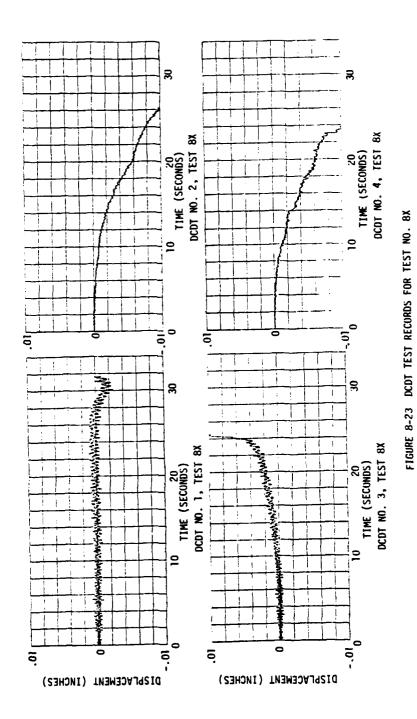
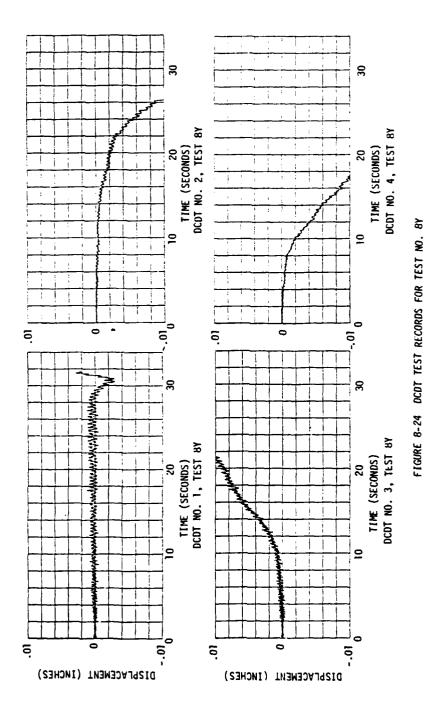


FIGURE 8-22 DCDT FEST RECORDS FOR TEST NO. 7X





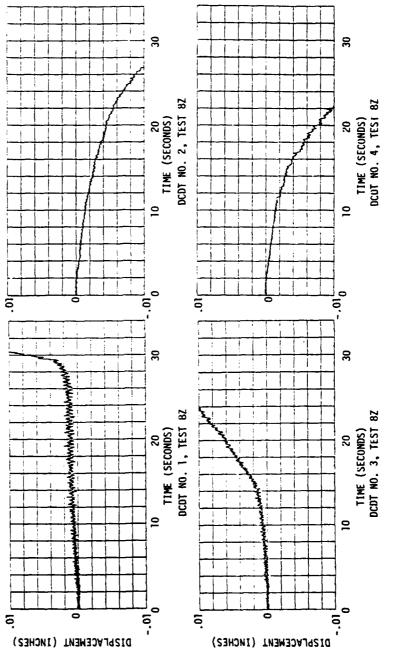


FIGURE 8-25 DCDT TEST RECORDS FOR TEST NO. 8Z

the test slopes was intended to assist in determining the point at which yielding began, and not to accurately measure displacements or strains. Because of the manner in which the DCDT core probes were carefully inserted into the soil, minimizing the disturbance to the soil structure, they are only capable of accurately measuring the very smallest displacements. As the sand grains become mobilized, they quickly flow around the core probes, thus preventing accurate measurements.

As an additional tool in this testing series, several tests were filmed with a high-speed data camera on 16 millimeter film. These films were synchronized to the base acceleration record and contained a minimum of 40 frames per cycle of loading. Thin horizontal lines were highlighted along the sides of those slopes which were photographed to provide a clear view of any large soil displacements and sand flows during dynamic loading. Additionally, a triangular "window" was cut out of one side of the test box to provide a better wide-angle view of the slope samples. These films are on file along with the other test results at the Geotechnical Engineering Office at the University of California, Berkeley.

Analysis of Real Slopes

The analyses of real slopes of cohesionless materials in the field, both natural and man-made, are somewhat more complex than for the carefully controlled laboratory test slopes of standard sand analyzed in this study. As discussed earlier, the predicted yield accelerations of slopes are extremely sensitive to the shear strength characteristics of the outermost slope material. The in-situ outer surface shear strengths may vary significantly in real slopes, and will be very difficult to accurately

measure. Nevertheless, it is believed that the analysis techniques described herein may be used with appropriate precautions and careful judgment to predict the yielding response of real slopes to real earthquakes.

When selecting the shear strength characteristics of the outer slope materials for analysis of a real slope, care should be taken to use the in-situ strengths. The shear strength intercept, $\mathbf{s_i}$, will tend to be higher in-situ, particularly for older slopes or those with surface vegetation. The primary effect of a higher value of $\mathbf{s_i}$ is to move the critical sliding surface down into the slope, thus increasing the yield acceleration.

For some slopes, special direct shear tests, similar to those performed in this study, may be performed. In other slopes some other special field test may be devised to ascertain the in-situ shear strength characteristics. In either case, careful judgment should be exercised in selecting the values of s_i and ϕ from the field tests to insure that the lowest measured values (and thus the most critical zones) are given maximum consideration in selecting weighted average values for analysis purposes.

Displacements

Seed and Goodman (1964) have also proposed a procedure for estimating total permanent displacements in slopes from yield acceleration values and dynamic acceleration records. This procedure is based on suggestions by Newmark (1963). It has been utilized and further refined by Makdisi and Seed (1977) and others. The procedure, simply stated, provides for the double integration of dynamic accelerations in excess of the yield accel-

eration to obtain values of dynamic displacement. Because displacement usually occurs during only one half of each dynamic cycle of loading, total residual displacement may be estimated by summing the incremental dynamic displacements accumulated for a particular earthquake record.

When this method is applied to real slopes, special precautions should be taken to consider the possibility that the predicted yield acceleration may decrease with progressively increasing displacements. Because the calculated yield acceleration is highly sensitive to the value used for the shear strength intercept, s_i ; the fact that the true value of s_i may change with straining should be taken into account. The variation of s_i with strain may also be evaluated from a field testing program.

It should also be noted that for the tests herein the horizontal and vertical accelerations were in phase and at the same frequency. Thus " ω " was constant for a given test. By contrast, in real slopes during an actual earthquake, the horizontal and vertical accelerations would be expected to have different and variable frequencies. Thus a numerical procedure using a computer program must be used to perform the double integration of the accelerations in excess of the yield acceleration. This integration must be performed over very small time steps, and the yield acceleration must be recomputed for each time step, as a function of ω .

The double integration process should be repeated for various relative "starting points" for the horizontal and vertical acceleration records in order to obtain the range of residual displacement values which corresponds to these records.

Conclusions

A large scale slope model testing program was developed to explore the effects of combined vertical and horizontal dynamic loading on the yield acceleration of sand slopes. The equipment, sample formation and instrumentation used was discussed in the first part of this chapter and in Appendix A-3.

The strength characteristics of the Monterey No. 0 sand were evaluated with special direct shear tests performed on the test slopes. The results of this evaluation are summarized in Figures 8-7 and 8-8.

The yield accelerations for the various dynamic test conditions were predicted by using an extension of an analysis method originally proposed by Seed and Goodman (1964). Equations 8.16 and 8.18 were derived to include the angle, ω , which permits a vertical component of acceleration to be included in the analysis. By modifying the simplified technique proposed by Goodman and Seed, a simplified technique which includes vertical as well as horizontal accelerations was developed. From those equations the yield accelerations in Table 8-4 were predicted.

Using a procedure described in this chapter, the range of yield accelerations observed during the various model tests were determined, and were compared with the predicted values in Table 8-5. It was difficult to draw definitive conclusions concerning these comparisons both because of the variation in acceleration amplitude from top to bottom of the slopes, and because of the relatively large increments by which the table accelerations were increased.

It was concluded from this study that, because of these limitations and the general technical difficulties of making accurate measurements of

accelerations, it is difficult to obtain highly precise values of observed yield accelerations. Nevertheless, it appears that the yield acceleration may be predicted by the method outlined in this chapter, at least as accurately as it may be directly measured. The results summarized in Table 8-5 appear to support the conclusion that the predicted values of yield acceleration are reasonable estimates of the true values.

It is recommended that direct superposition of simultaneous horizontal and vertical accelerations be made, as outlined in this chapter, if practical cases should arise where yield accelerations must be predicted.

Chapter 9

Summary and Conclusions

Introduction

The objective of this research was to study the interaction effects of combined compression and shear loading on the response of sands to dynamic loading. Various experimental studies were performed to evaluate the response of Monterey No. 0 sand to various types of combined compression and shear cyclic loading. The behavior observed in these tests was then used to evaluate postulated theories of soil response under these combined loading conditions. The results of these studies and the main conclusions are summarized in this Chapter.

Effects of Combined Dynamic Loading

During the course of this research significant effects were observed during combined compression and shear cyclic loading. The primary observed effect of combined loading was the more rapid degradation of modulus with strain than would otherwise occur.

Because the effect of primary practical interest is the influence of simultaneous shear straining on the degradation of the compression modulus, it was this effect that was evaluated in detail in this study.

Strain Ratio Method

Two methods were developed and presented for calculating the degra-

dation of compression modulus with strain under combined loading conditions. The first of these, called the Strain Ratio Method, requires the computation of either the instantaneous or an overall average ratio of shear strain amplitude to compression (normal) strain amplitude. By entering the sets of curves presented in Figure 7-7, the additional degradation in compression modulus due to the presence of shear strain may be determined. This method requires logarithmic interpolation among the curves, depending on the value of normal compression strain amplitude and mean confining pressure.

Octahedral Shearing Strain Method

A simpler method, called the Octahedral Shearing Strain Method, requires that either the instantaneous or an average value of octahedral shearing strain be calculated from Equation 2-42. If the principal strains are known, the simpler Equation 2-44 may be used for this calculation. Once the octahedral shearing strain is known, the total degradation in compression modulus may be determined directly from Figure 7-6, interpolating only for the mean confining stress. This total degradation in modulus includes both the component due to normal compressive straining and that due to the presence of shear strains.

Using either of these methods a reasonable estimate of the degradation of modulus with the total straining may be obtained. This is a significant improvement over the current state of engineering practice. However, analysis of Figure 7-7 shows that the degradation in modulus due to interaction is of importance only when the shear strain is significant compared to the normal strain. In other cases the use of this technique

provides no advantage over current practice.

Combined Loading Effects on Slope Stability

A series of large-scale shaking table tests were conducted upon slope models constructed of sand. These samples were subjected to purely horizontal or combined horizontal and vertical cyclic accelerations in an effort to determine the point at which yielding in the slopes began. The results of these tests were summarized in Table 8-5.

Analysis Method

A method of analysis was proposed for use in practical cases where yield accelerations for slopes under combined horizontal and vertical accelerations are needed. This method is an extension of a method originally developed by Seed and Goodman (1964) for horizontal excitation alone.

The analysis method requires the calculation of a yield acceleration, ky, from Equation 8.18 for the values of plane strain friction angle, ϕ ; slope angle, α ; angle of acceleration, ω ; shear strength intercept, s_i ; the length of the slope, L; and the density of the soil, γ_d . This value of ky is calculated for several values of the depth of sliding surface, d; and for the minimum value of the internal force, P, for those values of d. The minimum ky is determined by an iterative process, utilizing Equations 8-16 and 8-18.

A modified, simplified approach was also presented. This simplified method requires the determination of the factor, ϕ_{SL} , from Figure 8-16, and the calculation of the horizontal ky using Equation 8-19. This value is then modified to account for the acceleration angle, ω , by use of

Equation 8-18. An example of this latter process was presented using the shaking table strength characteristics and slope geometry, with the effect of variation of the angle, ω , shown in Figure 8-15.

This latter, simplified method was used for the slopes and strength characteristics of this testing program, with the results presented in Tables 8-4 and 8-5.

It was concluded from this study that, because of the limitations of the testing program and the general technical difficulties of making accurate measurements of accelerations, it is difficult to obtain highly precise values of experimentally determined yield accelerations. Nevertheless, it appears that the yield acceleration may be predicted by the analytical method outlined at least as accurately as it may be directly measured. The results of this testing program, summarized in Table 8.5, support the conclusion that the analysis method proposed provides reasonable estimates of the true yield acceleration values.

It is therefore recommended that direct superposition of simultaneous horizontal and vertical accelerations be made, as outlined in Chapter 8, should practical cases arise where yield accelerations for slopes must be predicted.

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 <u>Division</u>, ASCE, Vol. 90, No. SM6, Proc. Paper 4128, November, pp. 43-73.

Appendix A

Testing Apparatus

The following testing apparatus were utilized in this study:

- A-1 Resonant Column-Torsional Shear Apparatus
- A-2 Hollow Cylinder Test Apparatus
- A-3 Test Box for Slope Model Studies

Appendix A-1

Resonant Column-Torsional Shear Apparatus

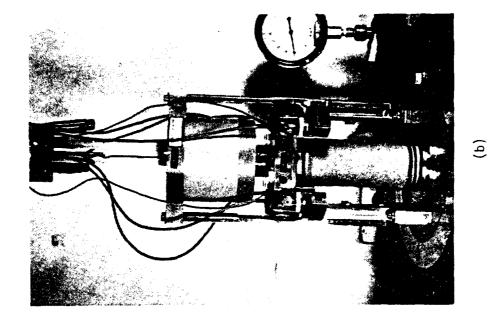
Introduction

The Resonant Column-Torsional Shear testing apparatus used in this study was developed by Professor Vincent P. Drnevich of the University of Kentucky. The apparatus is capable of exciting cylindrical specimens either vertically or torsionally, or in both modes simultaneously. The testing set-up and a close-up of a sample prepared for testing are shown in Figure Al-1.

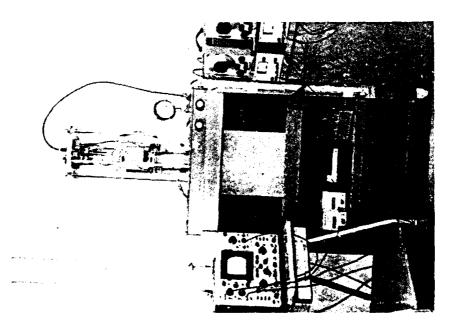
As seen in Figure Al-1(b), vertical excitation is obtained by applying an A.C. voltage across the large vertical coil mounted atop the specimen. The application of this voltage creates a sinusoidally varying magnetic field which interacts with the field of the large permanent magnet mounted to the counterbalanced framework, resulting in a sinusoidally varying vertical force being applied upon the specimen.

By varying the frequency and amplitude of the A.C. voltage, the cyclic stress applied upon the specimen can be controlled. A vertically oriented accelerometer is mounted within the cap assembly of the specimen, which measures the acceleration response to the applied cyclic stress.

Similarly, torsional stress is applied to the specimen by applying an A.C. voltage across the four torsionally oriented coils which are wired in series and mounted upon the counterbalanced framework alongside the specimen. The resulting electromagnetic field reacts with the magnetic field of the four torsionally oriented permanent magnets affixed to the cap assembly of the specimen, creating a cyclic torsional stress.







(a)

A torsionally oriented accelerometer is mounted in the cap assembly for measuring the acceleration response to the applied torsional cyclic stress. Because the two loading systems are mechanically independent, it is possible to simultaneously excite the specimen vertically and torsionally without mechanical interaction of the loading.

Test Set-up

The complete test set-up is shown in Figure Al-1(a). Just below the testing apparatus on the first shelf of the cabinet is a 1 ft cube concrete block. This block acts as a counter balance for the applied cyclic loads, and is securely bolted to the steel framed cabinet. The base of the testing apparatus, including the framework alongside the specimen and the large vertical permanent magnet are securely attached to this counterbalancing concrete block.

The testing apparatus is completely enclosed within an airtight lucite cell. This enclosure allows for the application of cell confining pressures up to approximately 7 KSC. Additionally, the specimen may be confined with intercell vacuum of less than 1 KSC. Although the equipment is designed to allow both sample saturation and the use of water or other fluid for applying the external cell confining pressure, these options were not utilized in this testing series.

The A.C. voltages used to excite the specimens in both the vertical and torsional directions were produced by a low-frequency sine wave generator and power amplifier combination. The two generator/amplifier combinations are shown in the right side of Figure Al-1(a), above the utility cabinet. The sine wave generators used in this testing series

were Hewlett-Packard Model 202C, and the power amplifiers were Hewlett-Packard Model 6824A.

The output of these power amplifiers is wired to a control box which allows the operator to quickly switch the voltage to the coils when desired. This control box, which is located on the right hand side of the second shelf below the testing apparatus in Figure Al-1(a), also provides a sampling point for precise measurement of the amplitude and frequency of the voltages applied to the coils.

Located just to the left of the control box in the center of the second shelf is a Fluke Model 1900a frequency counter used for precise frequency measurements. The applied A.C. voltage is displayed on the vertical plates of an oscilloscope; in this study a Tectronics Model 565 Dual Beam Oscilloscope was used, and is shown on the left side in Figure Al-1(a). A dual beam oscilloscope was helpful in this testing program because it allowed for simultaneous monitoring of the vertical and torsional response during combined loading.

The two accelerometers mounted in the cap assembly are Colombia electrolytic devices, Model 200-1-H, and require a charge amplifier or cathode follower to condition their outputs before they can be read by conventional high impedance measuring equipment. Two charge amplifiers were used to supply an acceleration response signal to the oscilloscope. These charge amplifiers, which are shown on the left side of the second shelf in Figure Al-1(a), are Colombia Model 4102.

The output of the charge amplifiers are applied to the horizontal plates of the oscilloscope to produce Lissajous figures with the voltages applied to the driving coils. The acceleration amplitude may also be

measured using the oscilloscope display.

Cylindrically shaped samples are formed with a rubber membrane and conventional forming molds. The vertical weight of the cap assembly is offset with a constant-force spring shown in the top of Figure Al-1(b).

Determination of the strain amplitudes and damping factors from the raw test data is described in Chapter 4; and the determination of dynamic compression and shear moduli is discussed in Appendix C-3. Several typical data sheets are shown in Figures Al-2 through Al-4, and the test results from this raw data are presented in Appendices Bl through B3.

UNIVERSITY OF CALIFORNIA RESONANT COLUMN - TORSIONAL SHEAR DATA SHEET

Date _5/1/79 Operator P SKIFFIN Test No. 1A-1,2,3 Page 1 of 1 345.00 1015.57 Wb1 Wb2 Spill _ @ 226 97 210. 40 Wal Wa2 Ws = W1 + ...2 - SpillWl 188 . 60 134.10 Ws = 922.70 W2 Diameter = 2.795 Confining = 3. Length = 5.398 Rec # .lode LorT Coil Volt Rec # Mode Accel Volt Accel Volt Reson Coil Reson Volt Frec $\sigma_{\mathbf{x}} \cdot \sigma$ 1A-1 27 52 324.4 20 205 4462 (48 hrs) 3.3 82 331 9 395 1 (a) 40 446.6 OCF 16% L τΦΦ 64 469 4 5 80 760 446.4 1(6) L 3.2 58 1700 445.3 3375 4 200 tiguten haviluare 3 2 63 327.9 Ū 1020 629 7 25.5% 1(0) 1000 Os. = 2.00 14-2 41 6(a) 445.6 2.8 200 188⊅ 406.7 L OCF 148% D L 100 59 575.4 7 400 3770 443.5 පිමුණ 1(4) 7.2 K 439.5 43 3.2 406.6 14-3 o+ 3.5 9 L 16.25K 434 4 3.2 4436 CCF 25.2% D 100 10 4 K 103 6273 31 K 424.7 8 K 414.5 49.5 K 1(0) 33 37 444.1 11 82 446.1 7.8

FIGURE A1-2 DATA SHEET FOR RESONANT COLUMN TEST NO. 1A-3

UNIVERSITY OF CALIFORNIA RESONANT COLUMN - TORSIONAL SMEAR DATA SHEET

Date <u>6/25/79</u> Operator <u>PM 6</u>	Test No. 128-2	_ Page <u>1</u> of <u>3</u>
Wb1 <u>1039.98</u>	WD2 815.90	Spill 0.64
Wal 286.82	Wa2 <u>618.32</u>	Ws = M1 + M2 - Spill
WI 753.16	W2 197.58	Ws = 950.10
Diameter = 2.785"	Length = 5.62	Confining = 2.0

Frec #	∷ode LorT	Coil Volt	Accel Volt	Reson Freq	Rec #	liode LorT	Coil Volt	Accel 7olt	Reson Freq	CF
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1	L	4	31	298.9	1	L	4	61	392.3	
2	Τ	4	36	107.4	2	Ĺ	8	120	391.5	
9	L	8.4	120	391.2	D	L	Too	65	\$\$4.8	17%
70	7	20	176	146.4	2a	L	8	120	391.6	
11	Ĺ	8.2	120	391.2	3	Т	4	30	146,4	
12	τ	40	345	146.3	4	τ	రి	64	146.3 207.0	
13	اد	8.2	150	391.2	D	Τ	1000	21	207.0	2.25%
14	τ	දිග	6 50	145.9	4a_	Т	8	65	146.3	
15	L	8.4	150	390.3 307.0	S	Ĺ	8	120	3920	
16	τ	200	1.37K	144.8	6	٣	4	33	146.4	
ון		8.6	150	369.3	٦	د	8.4	750	39 i.e	
18	τ	400	2.35	143.5	æ	Т	8.	\$ &	146.3	

FIGURE A1-3(a) DATA SHEET FOR RESONANT COLUMN TEST NO. 12B-2

UNIVERSITY OF CALIFORNIA RESONANT COLUMN - TORSIONAL SHEAR DATA SHEET

	te _ 6/3										
q 0	erator .	PM	6	Test	701	Page 2 of 3					
Wb1				Wb2			Spill				
Wal				Wa2		_	Ms = M1 + M2 - Spill				
W1				W2			Ws :	=			
Diameter =				Length	. =			Cining :	z		
Rec #		Coil 7olt	Accel Volt	Reson Freq	Rec 💤	Hode LorT	Coil Volt	Accel Volt	Reson Prec	ж	
١٩	١	ی) <u>(</u> 6	120	388.5	31	L	200	1 2.6k	38 6.8		
20	τ	800	3 9k	141.5	32	Ŧ	ික	1.3K	141.9		
21	ι	9.6	120	386.7	33_	L	512	2.6K	386.1	<u> </u>	
22	7	2 K	7.2K	137.2	34	T	400	2.25K	140.5		
23 25	20 10	TORS .	200	389.₫	35	L	225	26K	384.8		
24	Ĺ	20	305	389.7	36	٦	800	4K	1389		
25	L	40	620	389.2	37	L	250	2.6K	363 4		
26	L	80	1.23K	388 6	38	7	2K	8K	135.6 136.1		
27	Ĺ	200	2. 53 K	387.0	39	L	310	2.6k	379.1		
ze	τ	40	320	143.3	40	т	4 K	11.5K	134.0		
29	Ĺ	200	2.6K	387.0	41	L	300	2.6 K	312.9		
36	т	පිග	590	143.0	42	τ	8 K	17K	126 3		

FIGURE A1-3(b) DATA SHEET FOR RESONANT COLUMN TEST NO. 12B-2 (continued)

UNIVERSITY OF CALIFORNIA RESONANT COLUMN - TORSIONAL SHEAR DATA SHEET

	perator		A6 Test To. 126-2 Page 3 of _						г <u>з</u>		
ı	/bl	·		Wb2 Wa2			Spill				
V	n		W2				Ws				
	Mameter	=		Lengti	. =		Con Pre	fining ssure	=		
Hec :	ode LorT	Coil Yolt	Accel Volt	Reson Freq	Rec i	Mode LorT	Coil Volt	Accel Volt	Reson Frec	ŒF	
43	Ĺ	300	2.GK	348.૬							
44	т	11K	21 K	125.9							
45	L	300	2.6K	366.3							
46	K TOKS	265	2.6K	38 2 .0							

FIGURE A1-3(c) DATA SHEET FOR RESONANT COLUMN TEST NO. 12B-2 (continued)

UNIVERSITY OF CALIFORNIA RESONANT COLUNI: - TORSTONAL SHEAR DATA SHEET

		DATA SHEET											
			71-5/77										
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			727 60		W2	512	,4⊘			1.64			
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Ì	7		400	3 K	158.2	1	٣	4	56	104.7			
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	70		4 K	13K	150.4	1		4	26_	142.5			
l	11	7	8K	19K	144.2	Ð	т	1000	22	201.5	5.5%		
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ł	12	 		23%						181.0			
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						6	Т	200	1.8K	151.0			
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FIGURE A1-4 DATA SHEET FOR RESONANT COLUMN TEST NO. 18C-3

Appendix A-2

Hollow Cylinder Test Apparatus

The thin-walled hollow cylinder testing apparatus used in this testing program was a modification of a static loading device originally developed by Professor Poul V. Lade of the University of California at Los Angeles and Professor J. M. Duncan of the University of California at Berkeley (Lade, 1972). A front view of the testing apparatus showing the control panel is shown in Figure A2-1, and a diagram of the main loading chain is shown in Figure A2-2. The various numbered components from the load chain diagram are listed in Table A2-1.

This testing apparatus is capable of cyclic, stress-controlled loading in both the vertical and torsional directions, both separately and simultaneously. When vertical and torsional loading are applied simultaneously, both cyclic excitations are at the same frequency, but may be out of phase. The apparatus is capable of operating at any frequency between approximately 0.2 Hertz and 20 Hertz. All tests in this testing series were performed at a frequency of approximately 0.3 Hertz.

This apparatus is capable of loading hollow cylinder specimens cyclically at a peak-to-peak stress of up to 100 psi vertically and torsionally, simultaneously.

Loading System

Simply stated, dynamic load is applied to the test specimen as follows:

- A variable speed motor, controlled on the front panel, operates at the desired frequency of loading (i.e. at 0.3 Hertz).
- 2. Two offset cam assemblies which are attached to the shaft of

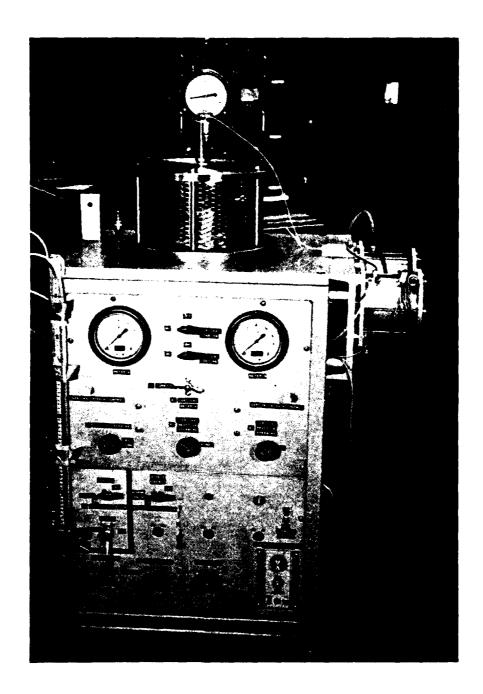


FIGURE A2-1 HOLLOW CYLINDER TESTING APPARATUS AND CONTROL PANEL

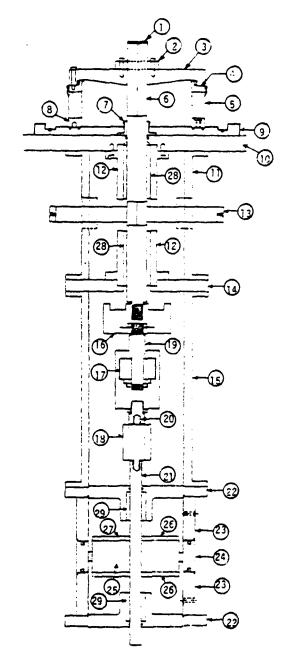


FIGURE A2-2 DETAIL OF LOAD CHAIN OF HOLLOW CYLINDER TESTING APPARATUS

TABLE A2-1

Components of Hollow Cylinder Testing Apparatus

- 1. Proximeter Target
- 2. Center Shaft Sample Top Clamping Assembly
- 3. Sample Top Piece
- 4. Sample Cap
- 5. Sample In Rubber Membrane
- 6. Center Shaft
- 7. Bellofram Rubber Cell Seal
- 8. Sample Base
- 9. Sample Bottom Plate
- 10. Top of Cabinet
- 11. Upper Standard Framework
- 12. Center Shaft Bearing Holders (2)
- 13. Torsional Moment Arms
- 14. Center Standard Plate
- 15. Lower Standard Framework
- 16. Vertical Load Cell
- 17. Thrust-Tension Joint
- 18. Double Ball Joint
- 19. Spacer Shaft
- 20. Spacer Shaft
- 21. Center Shaft of Double Acting Cylinder (Thompson Ball Shaft)
- 22. Top and Bottom Plates of Double Acting Cylinder (2)
- 23. Top and Bottom Walls of Double Acting Cylinder (2)

- 24. Center Wall of Double Acting Cylinder
- 25. Piston of Double Acting Cylinder
- 26. Piston Face Plates of Double Acting Cylinder (2)
- 27. Bellofram Rubber Membranes of Double Acting Cylinder (2)
- 28. Rotolin Center Shaft Bearings (2)
- 29. Thompson Ball Bearings (2)

the variable speed motor cause two pistons to traverse into and out of two small pressure regulators in a cyclic manner, at the same frequency as the motor.

- 3. The output of the two small pressure regulators are fed to the control of two large pneumatic volume booster relays which are fed with 100 psi air pressure and 1/2-inch supply lines.
- 4. The output of these pneumatic volume booster relays are fed into two large reservoir cylinders which contain an air-oil interface and a supply of low-viscosity loading oil.
- 5. The output of one of the two large cylinders is then fed directly to the lower half of the vertical double acting cylinder, while the output of the other cylinder is fed directly to one of two torsionally balanced torsional loading cylinders.
- 6. The cyclic pressure in the lower half of the vertical double acting cylinder reacts with the constant pressure in the upper half of that cylinder, resulting in a sinusoidally varying force which is transmitted through the center shaft to the sample top piece, and thus to the sample.
- 7. The cyclic pressure in one of the two torsional loading cylinders reacts with the constant pressure in the other cylinder, resulting in a torsional, sinusoidally varying stress which is transmitted through the torsional moment arms and the center shaft to the sample top piece, and thus to the sample.

A photograph of the left side of the testing apparatus showing the variable speed motor, offset cams, small pressure regulators, and the large pneumatic volume booster relays is shown in Figure A2-3. The speed

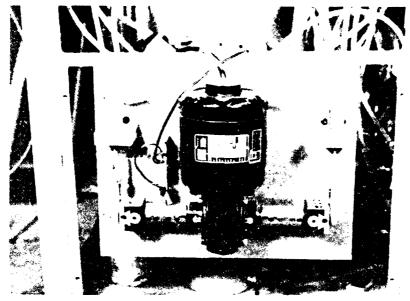




FIGURE A2-3 VIEW OF LEFT SIDE OF HOLLOW CYLINDER TESTING APPARATUS (a) AND CLOSE-UP OF DYNAMIC LOAD CONTROL SYSTEM (b)

(P)

control for the variable speed motor is visible in the lower right corner of the front control panel in Figure A2-1.

Careful examination of Figure A2-3(a) shows that the two pneumatic volume booster relays, which are mounted on the cross-brace in the center of the photograph are vented at the control port at their tops by muffled, pin type vent values. This is done because the small pressure regulators are not very efficient at venting air pressure when their regulated pressure is dropped, and they need additional externally provided venting to keep their time constants low during that portion of the loading cycle.

There are several details of interest in Figure A2-3(b). Close examination of the shaft couplers used between the center shaft of the variable speed motor and the offset cams reveals a small micro-switch mounted on the upper coupling. This microswitch was installed to provide a simple means of insuring that cyclic loading begins and ends at the same place in the loading cycle for each test. The switch is normally adjusted so that loading begins and ends at the balance point at the center of the sine wave of loading.

The phase lag between vertical and torsional loading is easily adjusted by adjusting the relative rotation of the offset cams with the shaft couplings. The offset cams are shown precisely in phase in the photograph.

The balance pressure at the center of the sine wave of loading is adjusted by positioning of the shaft between the offset cam and the piston of the small pressure regulator. Lengthening the shaft will result in a higher balance pressure. Care must be used when this adjustment is made that the selected balance pressure will allow sufficient "range" of peak-

to-peak pressure to accommodate the range of load anticipated in each test.

A photograph showing the rear of the testing apparatus is provided in Figure A2-4(a); and a photograph showing the right side and the two torsional loading cylinders is presented as Figure A2-4(b). A 3/4-inch pressure hose may be seen at the rear of the test cabinet. This hose provides air pressure for the loading system in this apparatus.

The two large reservoir cylinders, containing the air-oil interface, are in the bottom of the cabinet, out of the view of the photographs. Each cylinder is approximately seven inches in diameter and approximately fourteen inches high. It should be noted at this time that the use of air-oil cylinders and oil in the loading cylinders was provided in this testing apparatus so that it could also be used for strain-controlled static testing. If the apparatus was intended for use only for stress-controlled testing, it would have been quite possible (and desirable) to feed the output of the pneumatic volume booster relays directly into the loading cylinders, using air only.

The vertical double acting cylinder is visible in the center of Figure A2-4(a) near the bottom. A small differential pressure transducer is visible at the top of this cylinder. An identical differential pressure transducer is visible on the under-side of the right torsional loading cylinder in Figure A2-4(b). It is these two transducers which are calibrated to provide a measure of the vertical and torsional stress applied upon the specimens during testing.

Torsional strain is measured with a small LVDT mounted with long hose-clamps directly upon the sample (not visible in photographs). The vertical strain is measured with a proximeter device, which may be seen

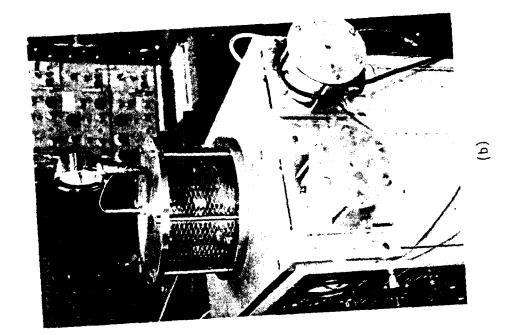
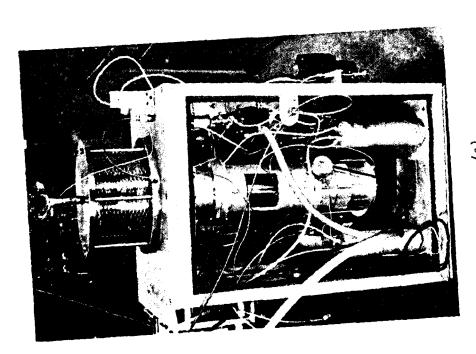


FIGURE A2-4 VIEW OF REAR (a) AND RIGHT SIDE OF HOLLOW CYLINDERS (b) TESTING APPARATUS SHOWING TORSIONAL LOADING CYLINDERS (b)



on the top of the outer pressure cell in Figure A2-4(b).

External cell pressure is measured with the pressure gage mounted directly behind the proximeter on the top of the pressure cell.

The levels of constant pressure are maintained in the upper half of the vertical double acting cylinder and in one of the torsional loading cylinders by means of pressure regulators controlled on the front control panel. The ensure that constant pressure is maintained during loading, two reservoir cylinders are provided in series with the pressure lines. These cylinders are visible at the bottom of the cabinet in Figure A2-4(a) on the two sides of the cabinet. It should be noted that two pneumatic volume booster relays would have served equally well in this position.

The vertical load cell shown in Figure A2-2 is used primarily for calibration purposes, for adjustment purposes and to estimate the magnitude of mechanical friction developed in the load chain. The thrust-tension joint and the double ball joint, which are also shown in that figure, are designed to de-couple the vertical and torsional loading applied to the center shaft. The double ball joint corrects any small misalignment between the vertical double acting cylinder and the center shaft, and the thrust-tension joint allows transfer of vertical normal stresses to the center shaft, but rotates freely, preventing any torsional stresses from transmitting to the vertical cylinder.

The two torsional loading cylinders transfer their load to the torsional moment arms by means of 1/4-inch cables 18-inches long. Resultant lateral forces from these cables are balanced out by the two large Rotolin bearings through which the center shaft passes.

Recording of vertical stress and strain during this testing program

was done on a Hewlett-Packard Model 7046A xy-recorder. Torsional stress and strain and vertical load cell stress were recorded on a Sanborn Model 150 strip chart recorder. Examples of several tests results are presented in Figure A2-5.

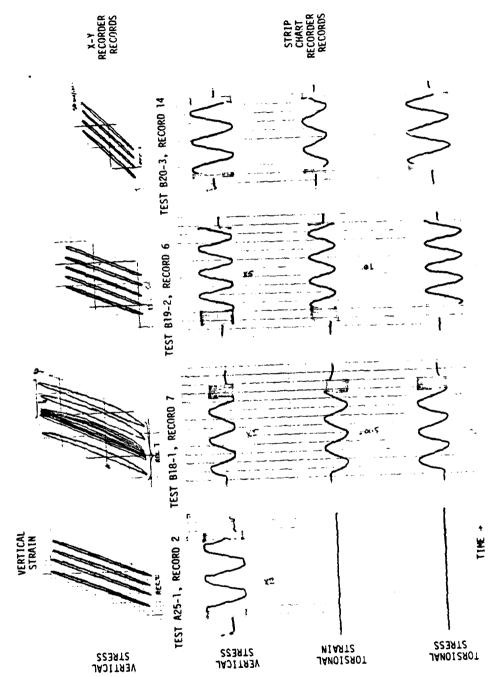


FIGURE A2-5 TEST RECORDS FOR 4 TYPICAL HOLLOW CYLINDER TESTS

APPENDIX A3

Test Box for Slope Model Studies

The slope model studies were performed in a wooden test box affixed to the 20 ft x 20 ft shaking table at the Earthquake Simulator Laboratory, Earthquake Engineering Research Center, University of California, Berkeley, California. The shaking table is capable of reproducing with excellent accuracy dynamic accelerations, velocities, or displacements stored on magnetic tape or disk. The test box was designed both to contain the sand test slopes and to efficiently transfer the dynamic table motions to the slope models.

A schematic diagram of a typical slope specimen is shown in Figure A3-1. The inner dimensions of the test box, as illustrated in this figure, are approximately 84.2 in long x 42.5 in wide x 21.7 in high. A spacer board has been added as shown in the figure to reduce the total volume of sand needed for the construction of a test specimen. This spacer board, which is positioned at an angle of approximately 45° with the horizontal, is approximately 15.8 in on a side. Also visible in Figure A3-1 is a 2 in steel angle reinforcing system on the outer edges of the test box. This reinforcing system provides rigidity to the outer walls of the test box.

A view of the test box affixed to the shaking table is shown in Figure A3-2(a). As the box was placed upon the table, a layer of high-strength "hydrostone" cement mortar was placed between the box and the table. The box was then firmly bolted down to the table by seven 1-1/2 in diameter steel bolts. Three of these bolts were tightened against a 1/4 in thick steel plate which was placed in the bottom of the box, and

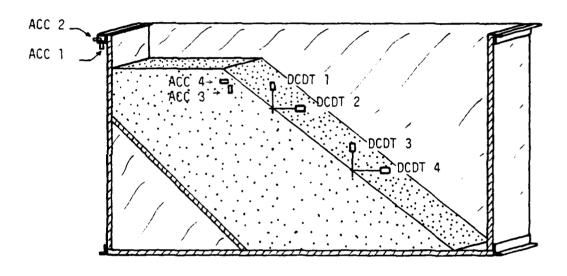


FIGURE A3-1 CUT-AWAY VIEW OF TYPICAL SLOPE SPECIMEN SHOWING LOCATION OF INSTRUMENTATION

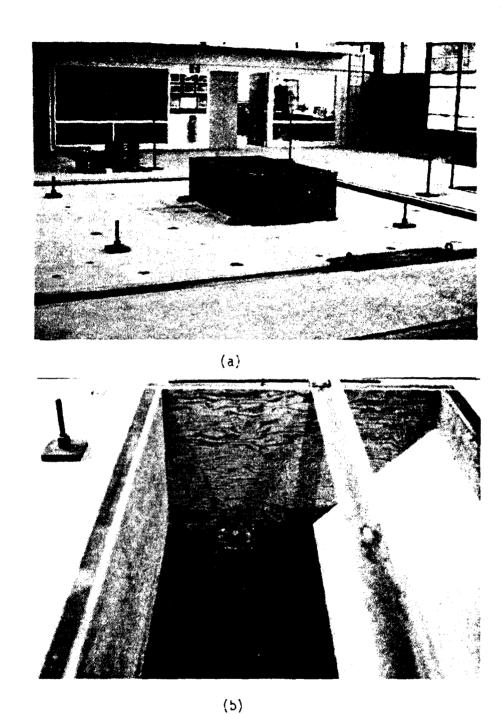


FIGURE A3-2 VIEW OF TEST BOX ON SHAKING TABLE (a) AND CLOSE-UP OF STEEL PLATE AND SPACER BOARD IN TEST BOX (b)

the remaining four bolts were tightened against the reinforcing steel angles on the outer corners of the test box. Two of these outside bolts are visible on the outer edge of the test box in Figure A3-2(a), and the steel plate and bolts in the bottom of the test box may be seen in Figure A3-2(b).

Behind the spacer board a stiff bracing system has been constructed to ensure that the deflection of the board during loading would be very small compared with the displacements of the soil slope. This bracing system consisted of six triangular 1-1/2 in thick plywood wedges, cut to exactly fit between the board and the inside corner of the test box.

Before slope specimens were constructed in the test box, a 2 in thick layer of high-strength "hydrostone" cement mortar was placed in the bottom of the box on top of the steel plate. The "hydrostone" was carefully leveled and a thin layer of the Monterey No. 0 test sand was cemented to the top of it. Also, a thin layer of the test sand was epoxyed with a high strength resin epoxy to the spacer board and the back wall of the test box. This treatment insured a level bottom and good soil-box contact between soil slope specimens and the test box.

A photograph showing the specimen forming "mold" in position within the test box is shown in Figure A3-3(a). The mold assembly consists of a 7 ft x 4 ft plywood panel, 1 in thick, which is reinforced with three longitudinal 3 in steel angles on the back side, and associated mounting hardware. The plywood panel is positioned to the desired specimen slope angle, and rigidly secured in place as shown in Figure A3-3(a). After the specimens are formed within the mold by table vibration, the plywood panel is carefully removed with an overhead crane, leaving a smooth slope

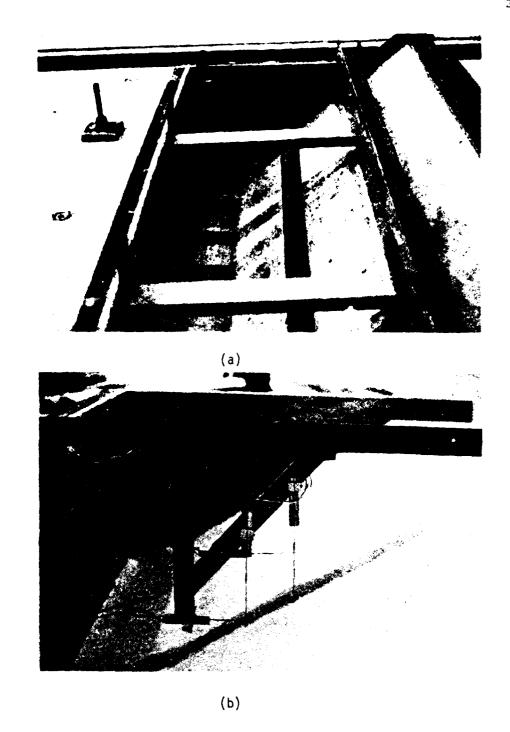


FIGURE A3-3 VIEW OF SPECIMEN FORMING MOLD IN PLACE WITHIN TEST BOX (a) AND VIEW OF DCDT'S ON MOUNTING FRAME AND FAILED SLOPE AFTER TESTING (b)

face on the specimen.

The mounting hardware which secures the plywood panel was designed to provide rigid support without distorting the smooth "plane" inner surface of the plywood. Two steel channels along the sides of the test box, covered with strips of compressible foam rubber, provide upward support at the edges for the plywood panel, while a longitudinal steel angle and two 2 in x 4 in lengths of lumber provide the securing downward force.

A 7 ft long lumber 2x4 provides the toe footing support at the bottom of the plywood panel.

Instrumentation is mounted both within the test slopes and on the test box in preparation for each test. As the slope specimens are formed, two very small accelerometers, one vertical and one horizontal, are carefully positioned within the sand near the top face of the slope. The accelerometers are mounted on 1/32 in thick perforated aluminum wafers, approximately 7/8 in on a side. This allows for relatively accurate orientation with respect to the horizontal and vertical directions. Also, two accelerometers are mounted on the back side of the test box, behind the top of the slope on the steel reinforcing angle. These two accelerometers are also oriented for the vertical and horizontal directions, and may be observed in Figure A3-1.

A cross-brace is placed across the center of the test box after the test specimen is formed, and four DCDT's are affixed to it on a mounting framework as shown in Figure A3-3(b). The slope pictured in this figure has already failed, and characteristic sloughing of the sand downslope may be observed. Note that the rod extensions of the DCDT cores are bent slightly as a result of the soil mobility. This effect could result in

inaccurate readings at significant strains, as discussed in this report.

One additional modification performed on the test box during this testing series was the cutting of a triangular shaped "window" on one side of the box. This was to permit the photographing of the slope face with a high speed data camera during testing. The opening was approximately 10 in x 10 in with a diagonal at 45° with the horizontal. This modification was necessary for low angle photography, nearly parallel with the slope.

Appendix B

Example Test Results

The following examples are included of test results from computer analysis of the raw test data:

Resonant Column Test Results

- B-l Test No. 1A-3
- B-2 Test No. 12B-2
- B-3 Test No. 18C-3

Hollow Cylinder Test Results

- B-4 Test No. A26L-2
- B-5 Test No. B29-3
- B-6 Test No. B31-1

The complete test results are available in Griffin (1980).

APPENDIX B-1 Resonant Column Test No. 1A-3

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40										
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29	44.32 66.89	51.45	\$1.45	o.	-6939E-03	665E-031	665E-03	8.	.1203E-03	.8113E-03
39		- 21 -22 -			-4939E-03	139E-03-2	15 0F -01 700E -03	- 8:	7402F-04	-AZAZE-C1
24 29 34 39	74.61	51.45	31.45	: :	-6939E-03 -6939E-03 -6939E-03 -6939E-03 -6939E-03	305 ¥-033 400E-033	053E-03 400E-03	9. 9.	.2776E-04	.8113E-03 .8443E-03 .8767E-03 .9094E-03 .9421E-03
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40	5161	2165	5143	BETA	EPSI	*052	EP\$3	EATA	EPSOC	\$440C
	PSI	PSI	PSI	DEGAE				JEGFE	ES #E#CE!	 NT 254CEM
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29	56.79	51.45	51.45	3.	. 1350E-02	1914F-711	9146-33	3.	14805-03	14415-3
29 34 30	54.79 67.30 69.24	51.45 51.45 51.45	51.45 51.45 51.45	š:	.1350E-02 .1350E-02	1914E - 33 3	914E-03	<u>.</u>	.1489E-03	17005-02
29 34 39 44 -	54.79 57.30 57.30 72.51 75.38 77.65	51.45 51.45 51.45 51.45 51.45	51.45 51.45 51.45 51.45 51.45	٥.	.1350E-02 .1350E-02 .1350E-02 .1350E-02	1914E-33-3 1914E-33-4 1263E-03-3 1934E-33-6	914E-33 566E-33 253E-31 436E-03 613E-03		.1489E-23 .1439E-24 .5398E-24 .5398E-25	15415-12 17055-02 17055-02 17035-02 18325-07 18465-02
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39 44 49	72.51 75.38 77.65 CALCULATE	31.45 51.45 0 37PESS 51 DEPS V PERCENT	51.45 51.45 51.45 51.45 51.45 0.45 0.45 0.45 0.45 0.45 0.45 0.45 0	ASTIC CO	DGANT DGANT DERCENT	DAMPT PERCENT	9	STRAT	.1489F-23 .1439E-23 .439E-26 .5398E-26 .5398E-25	11951 = -12 -1195 = -92 -1195 = -92 -1193 = -92 -1193 = -92 -11846 = -92
19	71-51 75.38 77-69 CALCULATE MJOV #51 71462.16	31.45 51.45 51.45 0 319655 5 DEPS V PERCENT .31186-02	TATE AND ELLOW PERCENT 2.65 51.65 51.65 51.65 51.65 51.65	ASTIC CO	DGANT DERCENT	DAMPT PERCENT EPS2 PERCENT	4 MUC EP\$3 P\$+CENT	STRAT	1439F-23 1439F-23 2474E-24 5398F-26 5398F-25	10015-12 1705F-02 11705F-02 1102F-07 1102F-07 1104F-02
	71-63 77-63 77-63 77-63 77-63 71-62-16 8101 951	21.45 21.45 21.45 2695 v PERCENT . J118E-02	31.45 31.45 51.45 51.45 51.45 91.45 91.45 94.45 94.45 94.45	ASTIC CO	DGANT DGANT PERCENT ES PERCENT	DAMPT PERCENT PERCENT	EPS3 PERCENT	STRAT	1439F-23 1439F-23 2474E-24 5398F-26 5398F-25	10015-12 1705F-02 11705F-02 1102F-07 1102F-07 1104F-02
14 - 24 - 24 - 24 - 24 - 24 - 24 - 24 -	01.10 75.38 77.65 CALCULATE MUOV 951 951 951 960 71.10 71.11 73.75	21,45 31,45 31,45 26,54	31.45 31.45 31.45 31.45 31.45 31.45 31.45 0AMPY PERCENT 2.65	ASTIC CO	DGAMT DGAMT PERCENT ES PERCENT All de - 02-1 Ji 165-02-1 Ji 165-02-1	DAMPT PEPCENI EPS2 PERCENT 148 BE - 93 - 74 360 C - 92 - 17	EP\$3 PRPCENT 93E-03 41E-03 10-02-02 21M-03	STRAT EATA DEGRE	10396-03 10396-04 10396-04 10396-04 10396-04 10396-03 10396-03 10396-03 10396-03	1705F-02 1705F-02 1705F-02 1103E-02 1103E-02 1104E-02
1949	71-63 77-63 77-63 77-63 77-63 71-62-16 8101 951	21.45 21.45 21.45 2695 v PERCENT . J118E-02	31.45 31.45 51.45 51.45 51.45 91.45 91.45 94.45 94.45 94.45	ASTIC CO	DGANT DGANT PERCENT ES PERCENT	DAMPT PEPCENI EPS2 PERCENT 148 BE - 93 - 74 360 C - 92 - 17	EP\$3 PRPCENT 93E-03 41E-03 10-02-02 21M-03	STRAT	1439F-23 1439F-23 2474E-24 5398F-26 5398F-25	1705F-02 1705F-02 1705F-02 1103E-02 1103E-02 1104E-02
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4 3 2 4 3 5 3 5 3 6 6 6	CALCULATE WIDV 951 71-62-1e 551 59-0e 71-10 71-10 71-10 71-70 71-10 71-70 71-7	31.05 0 37PESS 1 0 37PESS 1 0 37PESS 31 0 3100-02 31G2 31.05	TATE AND EL STATE	ASTIC CO	DGANT PEACSMI	DAMPT PERCENT PROCESS NO. 1 (1) (1) (1) (1) (1) (1) (1) (1) (1) (EPS3 PROCENT P	STRAT	10396-03 10396-04 10396-04 10396-04 10396-04 10396-03 10396-03 10396-03 10396-03	1705F-02 1705F-02 1705F-02 1103E-02 1103E-02 1104E-02
34 44 49 24 34 34 39	CALCULA TE	31.05 0 37PESS 1 0 37PESS 1 0 37PESS 31 0 3100-02 31G2 31.05	11.45 31.45 31.45 31.45 31.45 31.45 OAMPY PECENT 2.65 31.65 31.65 31.65 31.65 31.65	ASTIC CO	DGANT - SERCENT - SINGE-02	DAMPT PERCENT EPS2 PERCENT 483E-037 1041E-032 105E-031 125E-021	EP33 PRPCENT 181E-03 101E-03 101E-03 101E-03 101E-03 101E-03	STRAT EATA DEGRE G. G.	10396-03 10396-04 10396-04 10396-04 10396-04 10396-03 10396-03 10396-03 10396-03	1705F-02 1705F-02 1705F-02 1103E-02 1103E-02 1104E-02
34 44 49 24 24 34 39 49	CALCULATE WIDV 951 71-62-1e 551 59-0e 71-10 71-10 71-10 71-70 71-	31.05 0 37PESS 1 0 37PESS 1 0 37PESS 31 0 3100-02 31G2 31.05	31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45 31.45	ASTIC CO	DGAMT PERCENT ES PERCENT 31165-02	DAMPT PERCENT PERCENT PERCENT 100 10 - 0 - 0 0 0 0 0 0 0 0 0 0 0 0 0 0	9 44C EP\$3 P\$9CENT 81E-03 11E-03 11E-02 12E-02	STRAT	EPSOCES PROCES PROCE	1 0415-22 11705-22 11705-22 11905-22 11905-22 11905-02 11905-02 11905-02 11905-02 11905-02 11905-02
144 144 144 144 144 144 144 144 144 144	CALCULATE WIOV PSI 71-62-16 PSI 71-62-16 71-76-70	31.45 31.45 31.45 0.319655 51 0.6256MT 31.06-02 31.05 31.4	11.45 31	ASTIC CO	DGAMT PERCENT ES PERCENT 31165-02	DAMPT PERCENT PERCENT PERCENT 100 10 - 0 - 0 0 0 0 0 0 0 0 0 0 0 0 0 0	9 44C EP\$3 P\$9CENT 81E-03 11E-03 11E-02 12E-02	STRAT EATA DEGRE O. J. J. J. STRAT	EPSOCES PROCES PROCE	1 0415-22 11705-22 11705-22 11905-22 11905-22 11905-02 11905-02 11905-02 11905-02 11905-02 11905-02
100000000000000000000000000000000000000	CALCULATE #IOV PSI 11462-16 #IOV PSI 12462-16 ACCULATE #IOV PSI 12462-16 ACCULATE #IOV #I	31.05 0 31PE33 51 0 665 4 666 6 6 7 6 7 6 7 6 7 6 7 6 7 6 7 6 7	TATE AND EL OAMPY PERCENT 2.65 SIGJ PSI TATE AND EL OAMPY PERCENT 2.65 TATE AND EL OAMPY PERCENT 2.75	ASTIC CO	DEATT PERCENT	DAMPT PERCENT PROCENT 100 110 110 110 110 110 110 110 110 11	9 4UC EPS3 PROCENT 938-03 01E-03 01E-03 02-02 03E-02 03E-02 03E-02 03E-02	STRAT EATA DEGRE 3. 3. 3. 3. 3. 3. 3. 3. 3. 3. 3. 3. 3.	EPSOCES PROCES PROCE	10015-12 11705-12 11705-12 11932-12 11932-12 11946-02

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.29	70.55 73.13 75.70	51 -45 51 - 45 	51.45 51.45	٠	1033E-012 1033E-012 1033E-013	994E-7229	946-05	9:	.179CE-02 .1 .1446E-02 .1	2545
39	13.84	51.45 51.45	51.45 51.45 51.45	3: :	1033E-014 1033E-016 1033E-015	527E-02-44	276-32	č.	.75778-33 .1 .61306-23 .1 .63648-04 .1	1576
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APPENDIX B-2 Resonant Column Test No. 12B-2

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74	ST FFSUTS	FOR SAMPLI	e NO. 4						
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TY.	51 #F3ULTS	FOR SAMPLI UT - COAMS- - 956-10	Clam Cu-	ME IGHT	ND 39				-
TE	\$ 1/3C	UT GRANS	CIAM			part			
T T T	\$143C 45C — —	950 -10	C1AB	14.27	**	рит 31.45			
TE	STASC ASC — 2.00	950 -10	C1AM CB- 7.37	14.27 	29 DCFT				
T T T	\$1/3C 45C — 2.00 TCW GRANS 1570.A0	250.10 450.10 ACFV 2430.00	C1AM - C2- 7.37 OCFV	14.27 ACFT 2800.00	20 DCFT 5.25				
74	\$163C 45C — 2.00 TCW GRAWS 1570.60	#T	C1Am C27 7.37 OCFV 17.00	14.27 ACF7 2200.00 SHR	29 PCFT 5.25 Soute S.10	31,45			
T U	\$1/3C \$50 — 2.00 TCW GRAMS 1570.A0 VOL CC 301.00 —	27	C1AM CB- 7.37 OCFV 17.00 9841	14.27 ACF7 2200.00 SHR	29 PCFT 5.25 Soute S.10	31,45			
78	\$1/3C \$50 — 2.00 TCW GRAMS 1570.A0 VOL CC 301.00 —	27	C1AM CB- 7.37 OCFV 17.00 9841	14.27 ACF7 2200.00 SHR	29 PCFT 5.25 Soute S.10	31,45			
- Teg	\$1/3C \$50 — 2.00 TCW GRAMS 1570.A0 VOL CC 301.00 —	27	C1AM CB- 7.37 OCFV 17.00 9841	14.27 ACF7 2200.00 SHR	29 PCFT 5.25 Soute S.10	31,45			
74	\$1/3C \$50 — 2.00 TCW GRAMS 1570.A0 VOL CC 301.00 —	27	C1AM CB- 7.37 OCFV 17.00 9841	14.27 ACF7 2200.00 SHR	29 PCFT 5.25 Soute S.10	31,45			
Te e	\$1/3C \$50 — 2.00 TCW GRAMS 1570.A0 VOL CC 301.00 —	27	C1AM CB- 7.37 OCFV 17.00 9841	14.27 ACF7 2200.00 SHR	29 PCFT 5.25 Soute S.10	31,45			
Teg .	\$1/3C #\$C — — 2.00 TCW GRANS 1570.A0 VIL CC	250.10 450.10 ACFV 2430.00	7.37 7.27 7.27 7.20	ACFT 2000-00	29 PCFT 5.25 Soute S.10	31.45			

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	245.0	2400-0	3 143.0	• ••-•:					
	CH.CUL ATE	7 TRESS S		LASTIC CD	NSTANTS. DATA	SFT NO.	1	-	
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	37.41 39.40 40.87	29.40 29.40 29.40	79.43 29.48 - 29.40	4.	.1378F-043	684E-654	9976-25 4846-85 1746-05		1910 - 05 . 1474 - 04 1479 - 05 . 1741 - 04
	43.81	29.40	29,40	· ::	.1378# -045 .1378# -246 .1378# -046	754 E-056	754E-05		#3100-07 1611F-04 19305-05 174F-04 19305-05 1741F-04 1011F-05 180AF-34 5913F-04 1871F-04 91895-07 193AF-04
	CALCUL ATE	STHESS 5	FATE AND EL	LASTIC CO	NSTANTS, PATA	₹140.	7	-	
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	37.94 39.41	29.40	29.40 27.43 29.40 29.40	å:	.277 E-04+.7	ATT-057	#95F-05	9:	3912F-05 .3311F-24 2904F-05 .3440F-04
	40.85 42.35 43.82	29.40	29.40 29.40 29.40	- 4. 0.	.2723E-041 .2723E-041 .2723E-041	1985-041	062F-34 1985-04	3: :	3717-05 .3117-04 3717-05 .3117-04 7904-05 .3460-04 1997-05 .3468-04 1984-05 .3468-04 1815-06 .3875-04
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	400	7605V PERCENT 5172 PS1	PARE AND EL	#007 PSI	DGRAY DGRAY DEPCEMT 3018F-04 FS PERCENT	DAMPT PERCENT +66	MUC PRS3 PERCENT	EATA DEGREES	mincine Minch
	407 V 741 PSI 29.41 29.41 29.41	7605V PERCFMT \$152 PSI 20,40 20,40	PATE AND EN	#007 PSI	DGRAY DGRAY DEPCEMT 3018F-04 FS PERCENT	SET. NG DAMPY PERCENT OBS PERCENT OBS PERCENT OBS PERCENT OBS PERCENT	#UC FPS3 PERCENT 465E-04	EATA DEGREES AS. 500. 45.000.	2392F-0A , 749F-04 , 2302F-04
	400	PERSENT	PARE AND EL	#007 PSI	DGANT DATA	DAMPY PERCENT .66 EBS2 PERCENT -1	PPS3 PERCENT	EATA DEGREES AS. 800.	mincine Minch
	4 7 61 PSI 20.41 20.41 20.41 20.41 20.41	76 DSV PERCENT \$152 PSI 	74 30 29 30 29 30	PODT PSI	7 GANT DATA 7 GANT 30187-04 FOST PENCENT 1.4658-04 1.4658-04 1.4658-04 1.4658-04 1.4658-04 1.4658-04 1.4658-04 1.4658-04 1.4658-04 1.4658-04 1.4658-04 1.4658-04	DAMPY PERCENT .66 EBS2 PERCENT 1 1	#UC ##53 PERCENT 4636-04 4656-04 4656-04	E474 7#GRFFS 45.000. 45.000. 45.000.	2397F-GA ,747F-G4 ,2707F-G4 ,2702F-G6 ,2702F-G6
	9771 971 971 20.41 20.41 20.41 20.41 20.41	78 P S V P ER C F M T	104 AND STATE AND FLATE AN	PODT PSI 27722.20 BY TA OF GOE 45.00 45.00 45.00 45.00 45.00	DATA OGANY OGENERAT OGANY	DAMPY	#UC ##53 PERCENT 4636-04 4656-04 4656-04	E474 7#GRFFS 45.000. 45.000. 45.000.	2397F-GA ,747F-G4 ,2707F-G4 ,2702F-G6 ,2702F-G6
	4771 PSI 29-41 29-41 29-41	5172 PSI 20.40 20.40 20.40 20.40	CAMPY PERCENT	#007 PSI 27722.20 BFTA 0FGAP 45.00 45.00 45.00 45.00 45.00	PGANT PGANT PGECEMT 3018F-04 FOST PS PENCENT 1465F-04 1467F-04 1467F-04	DAMPY REPCENT .060 EBS? PERCENT ! ! ! !	FPS3 PERCENT PERCENT A655-04 4655-04 4656-04 4656-04	E474 7FGAFFS 45.000. 45.000. 47.000. 45.000.	2397F-GA ,747F-G4 ,2707F-G4 ,2702F-G6 ,2702F-G6
	4701 PSI 20-41 20-41 20-41 20-41 20-41 20-41 40-41	78.05W PERCENT	SIGS SIGS SIGS SIGS SIGS SIGS SIGS SIGS	#001 P\$1 22722,20 8f 14 0f 64 45.00 45.00 45.00 45.00 45.00 25.00 45.00	DATA DATA DATA DATA DATA DATA ABORDA FOSI FOSI FOSI FOSI FOSI FOSI FOSI FOS	DAMPY REPCENT .066 EPS2 PPECENT .11 .11 .11 .11 .12 .13 .14 .15 .15 .16 .16 .16 .16 .16 .16 .16 .16 .16 .16	### C	EATA THE CAFFS AS	2102F-04 .7107F-04 .7107F-04 .7107F-04 .7107F-04 .7207F-04
	901 951 951 20.41 20.41 20.41 20.41 20.41 20.41 20.41 20.41	78 0 5 V PERCENT	74 MOV PERCENT - 5173 PS 1 20 30 20 20 30 20 20 30 20 20 20 20 20 20 20 20 20 20 20 20 20	#001 PSI. 22722,70 85 TA 05 GBP 45.00 45.00 45.00 45.00 45.00 45.00 45.00 45.00 45.00	TOTAL TO	DAMPY REPCENT .00 EDST PERCENT .1 .1 .1 .1 .1 .1 .1 .1 .1 .1 .1 .1 .1	FPS3 FPSCHY AASE-GA AGSE-GA AGSE-GA AGSE-GA AGSE-GA AGSE-GA AGSE-GA AGSE-GA AGSE-GA	E174 PFGAFFS AS.000. 45.000. 45.000. 45.000. 45.000. 579AT	## 2103F-04 .7 107F-04 .7 107F-04 .7 107F-04 .7 107F-04 .7 107F-04 .7 207F-04
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		- 4163	***					•	*****
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	36.29 39.76 41.21 42.70	29.40 29.40 29.40 29.40 29.40	29.40 29.40 29.40	-71	.6170E-03 .6165E-03 .6160E-03 .6150E-03 .6152E-03	649E-03 25 E-03	5426-03 8 7 96 -03	.13 .1846E-83 .13 .0457E-84 .12 .0479E-84 .12 .447F-94 .11 .241%E-94	.7247E-03
	39.76	29.43 29.43	29.40	-51	6160E-03	2053E-032 2354E-032	176E-03	-12 -6439E-04	.7007E-03
	42. 70	29,40	29.40	• 39	.61526-03	2656E-032	7718-03	. 11 .241 E - 04	. 8359F - 03
	CALCULATE	STRESS ST	TATE AND E	LASTIC CO	STANTS. DAT		1		
	4CD ¥	SENCEAL DENZA	PERCENT	P001	DGAUT PE DCE NT	PERCENT	nuc	STRA T	
	*****	. 4043(- 63 -		-33433-18				A17A	
	5'11 	\$102 PS1	\$153			EPS2	Engo Rescent	EATA EPS CC	
	36 · A 2 18 · 29 39 · 74 41 · 23 42 · 73 44 · 17	29.40 29.40	29.40 29.40 29.40	1.30	.648(E-03 .6466E-03 .6436E-03 .6436E-03 .6414E-83	1 450E - 63 1 1 753E - 63 2	889E-03	-23 -1047E-33 -22 -8460E-04 -23 -4445E-04 -21 -4432E-04 -20 -2417E-04 -20 -4024E-05	·7693F-03 ·795章-03
	41.23	29.40	29.40	-62	.6436F-03	23576-032	7526-03	-21 -4432E-04	. 84 855 - 03
	44.17	29.40 29.40	29.40 29.40	:33	.6414E-03	2961E-033	33\$E-03	20 -4024E-05	.0019E-03
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					NSTANTS. DAT				
	PS 1	PERCENT	PERCENT	PSI	PERCENT	PERCENT	e.c	STRAT	
				• •					
	57840 -65 ·	6065E-03	1.17	22200.28	. 16696-02	.7598	. 3425	1.3756	
	5151	41.05	51G3	BETA DEGRE	#0 \$ ES DERCENT	DEDCENT FDS 3	ownCirt. Encs	FATA FRESC DESRFES PERCEN	GAMOC T PERCEN
	74 44	29.40 29.40 29.40	29.38 29.39 29.39 30.39 29.39		.7921F-03 .7677E-03 .7878F-03 .7740E-03 .7737E-03			.42 .1041F-03 .41 .6447E-04 .40 .6447E-04 .38 .2426E-04 .37 .4043E-09	
	34.33	29.40 29.40	29.38 29.39	2.39	.7877E-03	20626-033	8556-03 2276-03		.10246-05
	38.33 39.77 - 41.24 - 42.71 44.19	29.40	- 30, 30	1.60	.7740E-03-	2365f- 0 3~.4 2669E-03~.4	4RIF-33-		-1047E-03
		29.40	29.39	1.44	.76968-03	9728-034	6038-03	137 .4043E-05	. 10916-07
•					OGANT			STRAT	
	#CDV)EP\$V	DESCENS.		PERCENT	DAMOT PPRCENT— 	- 360S	2,4128	
	59438.39				.2946E-02				
	51G1	51 G2	5163 P51	PETA DEGREE		PERCENT	6953 0F0CENT	EATA EPSOC OFGRES PERCEN	
•	30.07 30.16 39.80 01.70	29.40 29.40 29.40 29.40	29.35 29.36 29.37 29.37 29.37	4:12	.1046F-02 .1046F-02 .1046F-02 .1036F-02	1771E-036	184E-03	.54 .6549F-04 .53 .6513E-04 .52 .4478E-04 .52 .4478E-04 .51 .4071E-05	1413F-02
	- - 1 - 2 6		29.37	- 3:11	1036 -02-	23 m E - 01 6	639E-63 663E-63		-1447F-02
	44.29	29.40	29.37	7.44	. 1 021 5-02-0	Z 4 7 Z E - 4 J 7	0 4 4 E -0 3	.51 .40712-05	.14776-07
		578255 51	TATE AND M	LASTIC CO		. SET NO. 2	•		
	CALCULATE		CAUPY PERCENT	900T	DGAMT	DAWR T PERCENT	•uc	STRAT	
		00000				.9077	.3016	4. 1999	
	#00V #51	DEPSV PERCENT 61518-03	1.34	21393.01	*23286-05				
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	₩00¥ •51		20.23 20.23 20.20 20.20 20.20	3E@#		PERCENT	8#43 OEPCENT	#454 P496 DEGREES PEOCE #65 1044 P49 #64 164 P49 #65 105 P49 #63 1651 P49	

:44	**. 32	79.40	~ 24.30 ~ 34.31	4:45	.154%-021 .157%-321	770AF-031 1614F-331	7395-22 2395-22	:33	.?**0F-04 -41616-04	:33477-03
	CALCULAT		TATE 4ND E		NSTANTS, DAT	set 40. 2	•		-	- ~
			DAMPY DESCENT							_
	95.1	DEDCEMA	DEDCE 4.	\$1.04 \$ \$4	DESCRIPT	DESCENT DESCENT	= 1¢	STRAT		
	57544.42	. ~ ? ~ 1 # ~ 3 3	- 1.44	\$9341.31	- 1120F-01	1.7346	-4174	8.9372		
٠.,	S [31	51G2	51G3 .						FPSCC	
, ,	17.47	29.40	29.75 28.85	19.44	.30 77#-0?1 .3046#-0?1 .3050#-0?1 .30 37#-0?1	510F-032	449F-37	• 77	. 1090F - 03	4444F-07
	40.24	29.10	24.43 24.97	11.42	.305CF-07	1 39E = 33 1	635F + 32	•7i	-67115-04	. 4446 -07
. 5	14.59	79.63	79.07 79.05	9.43	.3010E-02	7595-03-03	671 E-32	:70	.7514F-04 .4194F-05	• 6464F-07 • 4644F-07 • 4644F-07 • 8664F-07 • 4474F-07
	CALCULATE	O STEERS S	TATE AND E	LASTIC CC	wetamts, nata	sev wo. z				
	40'0V	DFPSV	DAMPY	MOOT	DEBCENT -	DAMPT	. AUC .	LARTZ _		
		.44025-33			.1009-01	1.7177	. 4043	17.7290		
•()	91.51 951	51G2 P51	5163	9ETA DEGREI			BEBCENA EB4J	25 as 55		
70	14.27	70.43 20.43	24.14	18.25 18.25	.4405F-07-1		920 E - 32		-1177E-01	.6P00F-07 .6E04F-07 .6E07F-07
14	42.43	29.40	24.55	13.16	. 4 14 7E - 37 1	376F-Q33	995F-02	- 73	47685-34	.6911F-62
٠.	14.	70.43	24.70	17.01	.43325-02	1146F-074	401F-07	.73		4614F-07
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		··· ·	·					·		
	3 C J 400 A	DESCRIPT	PERCENT	#007 P \$ 7	CGAMT PERCENT	PERCENT	*uc	. 57947		
	* 1472.75	.AAA 75 - 07	1.63	1 = 21 9 . 3 7	. 7449E-01	7.3240	.4984	20.0770		
4)	5161 ≈ 1	\$1.67 05.1	\$167 251	RF TA GE Œ FF	# DEQ(FNT	F05.7 PERCENT	BEBCEHT EDE3	OF GOFE		
4	15.21	29.43	- 24-99 - 27.76 27.47	26-33	.643RE-02-41	5956-03-4 978F-03-4	4338-02	-ZS.	-1152E-43	10-35-01
0	3 J. 67 31 . 69 62 . 97	79.40	27.00	71.00	.6007F-07-07	760F-016	468F-37	.73	. 4975 - 04	.130 TF -21
•	44.29	29.40	27.41	16.65	.646 2E -023	1925F-334 1257E-436	304F-07 517F-07		. 24 SQE - CA	.104 W-01
	CALF H.AT		 TATE AND E							
	5994	DEBSV DERCENT	7440V 258C5NT	#(*) * 8 \$ 1	DGAMT	7 4007 860(ENT	— ⊎ ∪¢	STRAT	•	
	956 51856,54	.677WF-03			*34541-61	2.5464	.5150	19.4076		
•	4141-	- 6143 PS 1	- (143 PS 1	- MEIA .	PPSI -	EPS2	FORT -	- FATA . DEGREE	E DE CC	GMDC IT PERCENT
	~ 40.17	79,43 29,40 29,40	26.05	20.12	. FEZ75-021	617E-03 A	3198-02		11695-03	-14005-01
•	~ 41.32 42.51	29.43	20 - 3 2 - 24 - 65 26 - 54	24.62 22.77	. PEZTE-02-1 -86116-02-1 . ETGSE-02-1 . ETGSE-02-1 . ETGSE-02-1	2918-034	143F-07	- 76	.7189F-G4	.1430F-01
•	•5.21 •6.33	20,40 24-44	37,39		. *763 * -072 4347E -083	130 H - 334	3456-05 -		-2499F-04	-14615-01
	CALCULATE	 PD ST#F 95 5	 TATE AND E	LASTIC CC	45T4NTS. 74TA	SPT NO. 2	•			
	470 Y	DERCENT	74 0 0 V	ernt est	NGAMT DEDCENT	DAMOT PERCENT	wc .	STRAT		-
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4 1)	9 7 61	* 152	4163	8514	F0 51	F057	* 053	-474	E=500	GAMOC
	4141	P\$1			E DESCENT	. DEG CEPAT.	DECEMI	. DE GREE	L. BERCES	II PERCENT
•	36.92	29.40	79.40		-61965-031			٥	10747-83	, 7744E-03
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14	14.76 41.23 -2.7)	29.40 29.40 29.40	29.40 29.40 29.40 29.40	0.	.6196F-03	-,2416E-07 -,2724E-07 -,3076E-03		3.	.4544F-34	.9120F-33
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APPENDIX B-3

Resonant Column Test No. 18C-3

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	รู้ สดด รูลิพตเ	F 40. 7		- - ·		
\$143C #56	S FOR SAMEL	F NO. 7	PE [GHT			
5 143C	5 FOR SAMBLE	7 40 7		NB 12		
\$143C #\$C 3.40 Tre GR444	9 FOR SAME	7+10 Of AN 7+10 OCFV		NB 12	Put 31:45	
\$143C 3.40 700 700	#7 - GRAME	0144 CFV 14-75	PF [GHT 14.78	NB 12 1CFT 5.75	Put 31:45	
\$143C #56 3.40 GP444 1476.40	9 FOR SAME #7 #8 AME 941-04 ACRY 7503-30 UNIT 7/CC	7 via 7 oraș oraș oraș oraș oraș oraș oraș oraș	PF [GHT 10.76 ACFT 7200.00	12 12 10FT 5,75 49414	Put 31.45	
\$1/3C 856	#7 - GRAME	0144 CFV 14-75	PF [GHT C 14.76 AC#1 7:00.00	12 12 12 12 14 15 15 15 15 16 16 17 17 18 18 18 18 18 18 18 18 18 18 18 18 18	Put 31.45	
\$153C 856	9 FOR SAME	7.10 01A4 7.10 0CFV 14.75	10.70 10.70 ACF1 7:00.00	12 12 12 12 14 15 15 15 15 16 16 17 17 18 18 18 18 18 18 18 18 18 18 18 18 18	Put 31.45	
\$1/3C 856	#T	7.10 01A4 7.10 0CFV 14.75	10.70 10.70 ACF1 7:00.00	12 12 12 12 14 15 15 15 15 16 16 17 17 18 18 18 18 18 18 18 18 18 18 18 18 18	Put 31.45	
\$1/3C 856	#T	01Au 01Au 7-10 0CFV 14-75 404 4E072	10.70 10.70 ACF1 7:00.00	12 12 12 12 14 15 15 15 15 16 16 17 17 18 18 18 18 18 18 18 18 18 18 18 18 18	Put 31.45	
\$1/3C 856	#T	01Au 01Au 7-10 0CFV 14-75 404 4E072	10.70 10.70 ACF1 7:00.00	12 12 12 12 14 15 15 15 15 16 16 17 17 18 18 18 18 18 18 18 18 18 18 18 18 18	Put 31.45	
\$173C \$173C \$560 \$1,4C \$770 \$60	#T	01Au 01Au 7-10 0CFV 14-75 404 4E072	10.70 10.70 ACF1 7:00.00	12 12 12 12 14 15 15 15 15 16 16 17 17 18 18 18 18 18 18 18 18 18 18 18 18 18	Put 31.45	
\$173C \$173C \$560 3.4C 776 GD144 1470-00 VPL CC 464-46 -0. -0. -0. -0. -0. -0. -0. -0.	#T	7.10 01A4 7.10 0CFV 14.75	PFIGHT 10.70 ACF1 7100.00	12 12 12 14 15 15 17 17 17 17 17 17 17 17 17 17 17 17 17	Put 31.45	

	951 	PERCENT	PERCENT	P51 27794.40 .	- GCAM I	PERCENT				
	51G1	\$1 G 2	\$1G3	BEYA DEGREFS	EPSI PERCENT	TPS2 PEPCENT	EPS 3 PE PCENT	EATA DEGREES	EPSOC PEPCENT	GA WOC PEOCENT
	41.40	51.45	51.44		10130-04-		4116-04			9598-04
	51 - 45 51 - 46	51.45 51.45	51.44	43.00	1412F-04 1412E-04 1412E-04 1612E-04 1612E-04			45.000.	. 2	94 9E - 04 959F - 04
	51.46 51.46	91.45 51.45	51.44 51.44	45.00	1812E-04	=-,	812F-34 812F-94 812F-94	45.000.		9596-04 9596-04 9596-04
	51.40		51.44	45.00 .	16126-04	11	M12F-04	45.200.		959E-04
					T MITSY -DATA		•			
-	4001	DEPSY DEPSY	0606543 -	#007	DGAMT	DAMPT PERCENT_	PUC	STRAT	`	
				27844.01 .		.48		· · · · · · · · · · · · · · · · · · ·	· 	
	5161 S 1	51 G 2	\$193 941	SE TA DE CO EES	FPS) PERCENT	EPSZ PERCENT	EP33 WENCENT	EATA OR GREES	EPROC NT	GANDC PERCEN
	51.49	51.45 51.45 51.45	91.42 91.42 91.42	45.00	3697E-04 3697E-04 3697E-04	3	697E-04 697E-04 697E-04	45.003.		0385-04
	51.49 51.49 51.49	51.45	51.42	45.00	3697E - 04	-:3	697E - 04	45.000.		038F-04 038F-04 038E-04 038E-04 038E-04
	51.49	51.45	51.42	45.08 .	369 7E - 04-	3	697E-04	45.000		038F-04
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-			DAMPY	#CO f	TANTS. DATA	DAMPT	3 4 ∪C	STRAT	: .	
	4-10 ¥	DEPSY	DENCENT	27931.72 .	#EDCENT 2226F-03	- PERCENT	<u>-</u>			
	617.		6153	85.14	5051	Fact	6063		E 175 C C	5 4H0 5
	51 th	6142	m T	CEGREES	PERCENT	PEDCANA.	PROCEAL	SEGREES	PERCENT	SAMOC N
	51. 81 61.61 61.61 61.61 61.61	51.45 51.45 51.45 51.45 51.45	51.30 51.39 41.30 41.30	45.00 - 45.00 - 45.00 - 45.00 -	9015F-04- 9015F-34- 9015F-34- 9015F-34- 9015F-34-		015F-04 015E-04 015F-04 015F-04 015E-04	45.000. 44.000. 44.000. 45.000.	- · · · · · · · · · · · · · · · · · · ·	477E = 93 472F = 93 472F = 93 472F = 33 472F = 33 472F = 33
	71.72				##133 efts e4-					
	C IL CUL ATE			LASTIC CONS	TANTS. DATA		•			
	><1 auun	DESCENT	OESCENT	#51 P51 27933-72 -	DGAMY PERCENT	DANDY DEDCFHT	NUC	STOAT		
	41.G1 41.G1	_ 51G2 _ 55L	51G1	- DF GREES	EPS:	ED\$3 DERCEME.	EPS3 DERCEME.	EATA - NE GARES	PERCENT.	GAMINE OF REFM
	91-57	51.45 61.46 51.46	51.33 51.33	45.00 .	1743F-03 1783F-03	E:17	783E-03 781E-03	45.000.	:3	911F-03
	51.57	81-46-	\$1.33 51.33 51.33		1783E-03		783E - 63	45.300	ب ــ ن	911F-03
	51.57 51.57 51.57 51.57 51.57	21:42	51.33	45.00	1783F-03 1783F-03 1783F-03 1783F-03	-:1	783F-03	45.000.	;	911E-03
					TANTS. DATA		_			
	a či	PERCENT	PE PCE NT	#807 P51 27824.20 .	PENCENT	PERCENT		STRAIL		
	31G1 P31	51 G2	5167	9674	# PS 1	FPS2 REPCENT	F0 5 1 PERCENT	EATA DE GREES	E # S OC PEPCENT	Brade
	51.00	51.45	510 22	06 saes s	32015-03-			- 45 .466 .		
		51.45	51 -22	15.00 .	3291E-03-	-:3	291 E ~ G 3 291 E ~ G 3	45.000.	: 3	374E-03
	51.64	91.49	21.53	45.60		- 2				1-7- 71
-	91.68 51.68 51.68 51.68	51.45 51.45 - 51.45 51.45	51.22 51.22 51.22 51.22		3291E-03 3291E-03 3291E-03 3291E-03 3291E-03		2416-03 2416-03 2416-03 2416-03 2416-03 2416-03	45.000.		374E-03 374E-03 374E-03 374E-03

	FALCUI ATFO	SIDF SS E	TATE AND F	LASTIC CEN	STARTS. NATA	SET NO.	•		
	477V	7FP5V	7440V	#C]*	PERCENT	CAMPT BERCENT	-70	57647	
					. 1 #4 OF - 02	.40			
	- 61.21 -	4152 PS1	51G3	SE LT T	EPSI S PERCENT	F0#2	P 0 5 3	FETA	EDETC GAMPE PRECENT DEDCENT
4940	41.06	51.45 51.45 51.45 51.45	50.94 50.94 53.94 53.94 53.94		.749FF-03 .749FF-03 .749FF-03 .749FF-03 .749F-03		4885-63 4885-33 4885-33 4885-33	45.000. 45.000. 45.000.	*1223F-Q2 *1224F-32 *1225F-C2 *1223F-C2 *1223F-32 *1223F-32
4	41.9A 41.3A 41.3A	51.45	53.94	44.30	74 8 AF - 21	-:;	400F-13	45.000.	1227F-C2
•	41.34 . 51.24	-61.45	40.74	_ A*.33	.7498E-03-		4A8E-23	45.302	.1277F-32 .1227F-32
									
	CHICANTEN	C TDF 44 4	TATE AND E		STANTS, DATA				
	-	7F25V	74404					STOAT	
	95 I	PERCENT	PERCENT	051	PERCENT	PERCENT		,,,,,,,	
				27240.63	.3113E-07	. 72			
· •	51 6 4			DEGREE	S DESCENT		SD 53	EATA TEOPERS	FRECE GAMES OFRCENT DESCENT
•	72.32 72.33 62.33 62.33 72.33	41.45 51.45 41.45 41.45 41.45	50.40	45.00	.1261E-02		261E-32	45.303.	.? 35 GF = 0? . 25 46F = 0?
	47.37 42.37	41.45	47.50	44.00	.1261E-02	1	561 E-03	45.303.	.2059F-02
	72.33	41.44	50.60 - 50.60 - 50.50 - 50.50 -	45.00 45.00	.1261E-02 .1261E-02 .1261E-02 .1261E-02		241 E-02 241E-02 261E-02 241E-02 241E-02	45.000. 45.000. 45.000.	27468-07 27468-07 2749-27 2749-07 2749-27 2749-27
	CALC # ATES	STEF 55 5	- TATE AND E	- LASTIC CON	- ISTANTS, NATA	5ET NO.			
-		3=05V	7440V 050CENT		DSAMT GERCENT	DAMPT DESCENT	`⊃رب⊯	STEAT	
	191	SEDCENT			-	_			
				50824-18	-\$149E-C2	90 .			
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,	31G1 144	*162 *****		D CO. EL				DE COPER	EDC(FAT DEDCFAT
	57.41 57.41	41.45	53.09	45.00	. 20 A QF - 32 . 20 69 F - 02	?	0A9F-37	45.000. 45.000. 45.000.	. * * * * * * * * * * * * * * * * * * *
. 7	43.41	51.45 61.45 51.45	50, 30 -		-3040E-05-	?	0405-03 0405-03 0405-03	45.000.	1170F-77 1170F-77 1170F-77 1170F-77 1170F-77
,	57.41 52.41 52.41	51.45	13.09 53.09 50.09 50.09 50.09	45.00	.20 A 9F - 32 + . .20 69 F - 32 .20 69 F - 32 .20 69 F - 02 .20 69 F - 02 + .	_ ::3	040 £ -05 240 £ - 05	45.000.	11705-07
				.==.5=3.1.	STANTS. DATA				
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	च्छी क्यानक ∽	PERCENT	DESCENT		DEDCENT		_ WUC	-	
					• dddde -05	1.20			
,	51G1	51 G 2	5173 PS 1	NF TA OF GP EE		F P5 2 P EB CENT	EBCÉML EBCÉML	FAT & DEGREES	FORME SAMOS DEBUENT DEBUENT
	47.00	51.45	44.91	**.00	. 404 3F - 02		043E-02	45.200.	*********
	53.00 53.10 53.40 43.44	51.45	49.91	45.00	40478-02-		043E-07	45.330.	.AA 7 TF - 7 7 . AF 7 TF - 0 7 . AF 7 TF - 0 7 . AA 7 TF - 0 7 . AA 7 TF - 0 7 . AA 7 TF - 0 7 .
	43.44	51.45 61.45 51.45 61.45 61.45	49.91	45.00	. 4043F-02 .4043E-02 .4047E-02 .4047F-02 .4047F-02	4	843E-67 947E-02 943E-02	45.000.	+603F-02
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	-	410F35 -5	1-41-F 441 9 El	45746- 68 4	57 MIT 5 DAT A	- 			
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-		- 2 40 € € 41 -		24020+64					
	\$ 1 71 P\$1	5 1 G 2	\$163 PS 1	AF TA		E = 52 P EP C ENT	E P S T	#A74_	EOSTE GAMOT OFOCENT OFOCENT
 j	P\$1			DECREE				DEGREE'S	
					4044		044F-02	45.000	94 705 - 02
	55.12	51.45	47.78	44.00	. EQ 44F - 07	4	3448-37	45.200.	,Q470F-03
0	55.12 55.12 55.12 55.12	51.45 51.45 51.45 51.45 51.45	47.78 47.78 47.78 47.78 47.78	45.00	.6048-02 .6048-02 .6348-02 .6348-02		044E-07 044E-07 044E-07	44.000. 45.000. 45.000. 44.000.	.0070F-02 .0070F-02 .0070F-02 .0070F-02 .0070F-02

	1.UA	OF RCENT	PERCENT	₩DOT PS t	DEBCENT	PERCENT	MUC	STRAT		
				21264.75 .	53096-31	2.26	-			
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	56.82	41.45	45. 08		934AE-02		148E-02	45.000.		1577-01
		\$1.49	- 46-04		634 5 - 22-a					15778-01
34	46. 97	51.45	45.08		9148F-07		14 FF - 32	45.000.		1977 - 61
	56.62	51.45	46.04	43.00	914AE-07-	9	14 95 - 22	44.000.		1577E-01
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-	 CM OH MF!		TATE AND EL	ASTIC CONS		9FT 40. 12				
_	CALCULATES SOLVER	STRESS ST	CAMINA SE	MOOT PSE	TANTS, DATA	SET NO. 1:	NUC -	STRAT		
_	4C0A	1FPSV	PEPCENT	MOOT PS (2 GANT	DAMP T PERCENT	- 	STRAT	·	·
·	4C0A	1FPSV	PEPCENT	MOOT PS [PERCENT	DAMPT PERCENT 2459	- 	STRAT EATA DE GREES	ED40C	5497C PERCE
···	4C04 201	TEDSV DECCENT SIGT PGI	CICLE COLORS	MOOT PSI 22569.83 A RETA DESPEES 45.00 .	7GANT PRRCENT ZARLE-01 FDS: _ PERCENT 1166#-01	DANDT PPRCPNT 2459 EDS? PERCENT	F043 PERCENT	EATA DE GREES	-PERCENT	
74	4CDV 7SI 51G1 PS1 57-95 57-95	SIGT	C143 251 251 44.95	MODT PSI 22569.83 A RFTA DESPRES 45.00 .	7GANT PFRCENT 2881E-01 FOS! _ PERCENT !166E-01	DARPY PERCENT 2.59 EDS? PERCENT	F243 PERCENT 164F-01	EATA DE GREES	PERCENT	PERCE 1905F-01 1935F-01
74 29	51.51 95.1 57.35 57.35 57.35	1F05V DFC(FNT 51G? PG1 51.45 51.45	CAMPY PEPCENT CIGI PSI 44.95 44.95	MF74 DESDESS 45.00 .	7GAWT PERCENT 2881E-01 FOS! _ PERCENT 1166#-01 1166E-01	DARDY PPRCENT Za59 EPS? PERCENT	F043 PERCENT (64F-01 (64F-01	EATA DE GREES	_PERCENT	PERCE 1909F-01 1939F-01 1939E-01
74	4CDV 7SI 51G1 PS1 57-95 57-95	SIGT	C143 251 251 44.95	MFTA DE SPEE S 45.00 .	7GANT PFRCENT 2881E-01 FOS! _ PERCENT !166E-01	DANDY PPRCENT 2459 EDS? PERCENT	F243 PERCENT 164F-01	EATA DE GREES		PERCE 1905F-01 1935F-01

APPENDIX B-4 Hollow Cylinder Test No. A26L-2

@ FF 4, 1851 47. A76L-2. 449CH 10, 1975			
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TEST RESULTS FOR SANULE NO. 16			
STG IC MT MEIGHT T NP			
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Millione 1.56			
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1.720 .001P -202020202020000000000			
15.6993 .0196 -903. 21.3333 .0270 -000.			
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49.1300 .3630 -0Q0. 95.2700 .0430 -000.			
74, 4386			
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	- sici	P\$ 1		DEGREES	PERCENT	PERCENT	PERCENT	OF GREES	ME BC E MI	PERCENT
•~		3000			-18445-030-		-56306 -63-			-40715-01
4	55.19	29.40 29.40	24.53	4.	. 18+6E-020.	:	-75416-03 -95106-03		. 431 LE-03 (0-30466, 0-34 995	.2149 -02 .23716-42
	55.19 55.19	72.00 12.00	26.76 29.40	<u>.</u>	.1 8465 -0 20 . -1 844E = 02 0.				*5 1 0E - D 3	. 244 IF - 07
9	55.19	41.45	29.40	٠.	.1846E-070.		.1774E-02	•.	.741 F -04	. 2 97 66 - 02
of.	55 STATE 4	NO ELASTIC	CONSTANTS	. DATA SET	NG. 2					
ij	400 VT	PERCENT	#00A2	PERCENT	#007 #51	DENCENT	C4007 P\$1	GENCEN. CRTAA	#UC	STRAT
9	30910.12	-6154F-02 -6154F-02	13972.58	-56628 -00 -56628 -00			19931.50 14983.37 13935.25			
9	33904.32	.0154E-32	12302.00	-86624+00 -56624+00			11779.00			
ບ້	\$161 P31	821 21es	\$1G3 P\$1	BETA DESPEES	PERCENT	PERCENT PERCENT	EPS3 PERCENT	EATA CEGREES	EP SOC	GAMNC BF#CENT
•	54.77 56.97	29,40 79,40	23.73 25.05 29.37	<u>:</u>	.6154F-020. .6154F-020.		.1943E-02 .2514E-02	÷:	.1404E-02 .1213F-02	.440 3E-32 .72 42 - 02
		79.40 29.40	26 .44-				-34346-02-			
3	54.97	38.00	29.40	• • • • • • • • • • • • • • • • • • •	.6154E-020.			: :	. #39 0F -03 . #044 E-04	. 285 3E-02
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24	39841.97 37749.45	.11285-01 .11285-01 .11285-01 .11285-01	145 09.27	. 571 3F + 00 . 571 3E + 00			13667.03 14631.65			
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-				.5713E+00	······					
43	31.114.07	.11298-01	11763.99	• E 71 SE + 00			10510.06			
ن. •	\$16°	5 1 G2	5 (63	META DEGREES	ano g y	FD 52 0ERCF4T	8993 0E8CF47	FATA	EPSCC	GANDE
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	59,04 59,04						- 440# -02 - 54178-03			
	-7.0	30.91	29.40	٥.	.1120F-010.		- 10945-01		.009@E-03	164 QE -01
.0	59,04	43.34	29.40				- 11 4300 - 71			
1 4 F	44 6545	m a 46.4611 6	- CONSTANT	L, .0434_ 4F	E -8:G4					
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74	27727.29	.144 F-01	1 4678. 70	.5795E+00			15015.00			
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44 ·	9191 9191 971	3172	\$1 53 	<u>.</u>	. [9448-010. [9448-010.		ASRCE 43.	0: 0: - 0:¢weir	we at ent-	
44	90000005 9191 991	3112 29.40 31:11 35.60 40.20	\$153 21.96	0	.19492-010.		A 154F-02	- 00000000		. 21 86F - 91 . P306F-01 . 267 9F-01 . 267 9F-01
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19										
•	35008.85	.2764E-01	13625- 04	-5877E+00			11779.79			
•	29224.49	.27A9E-01	12211-31	-3877E+00	·- 		9806.98			
		6163-			-40SI	4PL				- GAMOC PERCENT
_	95 (- ei es	PSI	DEGREES	BERCENT	PERCENT	PERCENT	DEGREES	PERCFHT	PROCENT
٠.			27.39	<u>.</u>	-276-016.			<u>.</u>	-4316E-02 .5460F-02 .4676F-02 .337@-02 .1974E-02 .3620E-03	- 31 96F -01
29	65.04	32.11	29.40	3:	.2769E-010.		[131E-01 [477E-01	: :	4440E-02	.327W-01
	45.34	36 . 43	29.40		.2769E-010.		17705-01		.337 9E -02	-494 2F - 01
Ġ	65.34	46.26	29.40	ě.	.2769E-010.	•	266 IE - Q I	٥.	. 36206-03	.4474#-31
96	SS STATE	IND ELASTIC	CONST ANTS	. DATA SET	40. 4					,
.,	400VT	DFPSV	#ggvs	EPSV	#077T	04447	C#207	CGANY		STRAT
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	34406.47 37404.13 38123.74	.35506-01 .35906-01	1 4 4 4 4	-5959E+00			13449.43			
19-	13121.10			-5959E +00			10408-00			
9	26799.35	.35907-01	124 19.36	.6424E+00			9460.00			
ii)	51G1 P51	5 f G2	\$ 163 P\$1	BETA DEGREES	PERCENT	PERCENT	EP53 PERCENT	EATA CFSREES	PROCENT	GAUCT REDCENT
•	67.99	29.40 29.40	23, 37 26, 24 29, 40	g.	. 3590E-010.		1130E-01		.R 1975-02 .7078-02	.40 > 7E - 01
	A7.99	33.11	29.46 29.46		.3590E-010. .3590E-010.		1849F -01			
14	67.99	42.45 47.72	29.40	0.	.3590E-010.		20216-01 34496-01		.25e4F-02	.5747E-0
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o ę		ND FLASTIC								
	47.0V7 P31	25.32 A 25.32 A 36.32 A	NO CVS	PERSY	#00 T PS (PERCENT	CN007	PERCENT	4 UC	STOAT
•	23742.69	.4619E~01	15600.07	.68424+00			13626.17			
9	77942.65 31713-11 31033-59 24016-14	.4615F-01 .4615F-01 .4615F-01	14535.74	.60h2E+00 .49h2E+09 .60h2E+09			12729.71 11933-25 12934-79			
•	77 942 4 65 31 71 74 14 33 63 74 2 6 2 9 0 1 64 1 6 2 7 2 6 74 75	•4615F-01 •4615F-01 •4015F-01 •4015F-01	14535.74				12729.71 11833-25 1293-27 12040-77 9147-87			
0	77962.65 31713.14 33633.26 34016.16 27264.75 5151 951	**************************************	14535.74	-6067F+33	PERCENT FOSI	E0 52 E0 52		Fata hegrees	######################################	GANCC DESCENT
40 40 N . 45	5 (51 9 51 7 4 9 4	5 G2 95 29. 40	\$163 \$163 \$163 \$51	BETA DEGREES	EGS! BENCENT	* EP CENT	9147-87 ERS3 ERS3 ERS5 ERS7 1497F-01	nesnees	DEDCEMA	PERCENT
40 40 D ' 95 9	5151 951 70.74 77.74 73.34	29.40 29.40 29.40	\$163 \$163 \$163 \$51	BETA DEGREES	EGS! BENCENT	* EP CENT	9147.47 ERS3 PRECENT -1497#-01 -1995E-01	DEGREES	.1943#-91 .9101#-92 .459#-93	5177F-31 -5177F-31 -5461F-01 -5886F-01
±0.00 €	5 (51 9 51 74. 94 73. 94	5 G2 P 5	146 35 . 74 13347 . 76 17479 . 05 5163 951	BETA DEGREES	PERCENT FOSI	PEPCENT	9147-87 ERS3 ERS3 ERS5 ERS7 1497F-01	nesnees	#PSCF #PSCF = 02 -1910 = 03 -1910 = 03 -1910 = 03 -1910 = 03 -1910 = 03 -1910 = 03	5177F-31 -5177F-31 -5461F-01 -5886F-01
+0 +0 · · · · · · · · · ·	5151 951 70.74 77.74 73.34 70.94	\$162 29.40 20.40 34.43 79.13	13347.70 13347.70 12270.05 \$163 \$51 29.10 29.40 29.40 29.40	.606/76-00 .606/76-00	#031 #ERCENT .6613F-010. .6613F-010. .6613F-010.	PEPCENT	ERS3 PRECENT 	nesetts	.1943#-91 .9101#-92 .459#-93	5177F-31 -5177F-31 -5461F-01 -5886F-01
10 40 W 10 40 40 FF	\$ 251 051 70.94 70.94 70.94 70.94 70.94	5 G2	14.35.74 13343.79 13947.05 3163 951 24.08 29.10 29.40 29.40 29.40	BETA DEGREES 0. 0. 0.	FOS1 PERCENT .46198-01046198-01046198-01046198-01046198-010.	Percent	7144,67 7144,67 6953 6992,647 -114476-01 -21476-01 -21476-01 -21476-01 -3146-01	0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.	.1041F-01 .9101F-02 .7450F-02 .746F-02 .3267F-02	5177F-31 -5177F-31 -5461F-01 -536F-31 -678F-31 -678F-31
10 40 W 10 40 40 FF	5151 951 70.9a 77.9a 70.9a 70.9a 70.94	51G2 951 29.40 29.40 34.42 79.13 44.15	14(35.74 1334-17 12=79.05 \$163 \$51 24.08 29.10 -29.40 29.40 29.40	BETA DEGREES 0. 0.	#031 #ERCENT .661 % # 010 .661 % # 010	PEPCENT	ERS3 PRECENT 	nesetts	10 4 18-01 -10 4 18-02 -7 45 98-03 -4 4 98-02 -3 2 4 78-03 -5 3 7 38-03	5177F-31 -5177F-31 -5461F-01 -5886F-01
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0 40 U 40 40 40 40 40 40 40 40 40 40 40 40 40	9131 73.74 73.74 73.74 70.94 70.94 70.94 70.94 95.574 Ed. of 97.774, ag. 3105-31 3105-31 32071, 22 7277, 22	9162 93.40 90.40 34.43 70.13 44.15 49.15 4	14.33.70.131.70.70.131.70.70.70.70.70.70.70.70.70.70.70.70.70.	BEYN - 30 - 0 - 0 - 0 - 0 - 0 - 0 - 0 - 0 -	FOS1 PERCENT	PEPCENT	EPS3 PTPCCPY -1477-01 -1378-01 -2178-02 -2178-03 -2178-03 -2178-03 -2178-03 -2178-03 -2178-03 -2178-03 -2178-03 -2178-03 -2178-03 -2178-03 -2178-03 -2178-03	DESREES 0. 0. 0. 0. 0.	911C	5177F-31 -5177F-31 -5461F-01 -536F-31 -678F-31 -678F-31
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10 10 Participant (FF III) 740-140	9131 73.74 73.74 73.74 70.94 70.94 70.94 70.94 95.574 Ed. of 97.774, ag. 3105-31 3105-31 32071, 22 7277, 22	9162 93.40 90.40 34.43 70.13 44.15 49.15 4	14.33.70.131.70.70.131.70.70.70.70.70.70.70.70.70.70.70.70.70.	BEYN - 30 - 0 - 0 - 0 - 0 - 0 - 0 - 0 - 0 -	FOS1 PERCENT	PEPCENT	### 1949 - 19	OFFICE S	911C	5177F-0 -5146F-0 -586F-0 -5966F-0 -6927F-0 -6747E-0 -7700F-0
30 30 30 30 30 40	98 474 42-4 91 70-94	5 G2 95 20.40 20.40 34.43 70.13 40.17 40.17 6602E-01 6602E-01 6602E-01 6602E-01 6602E-01 6602E-01	14039.70 13314.70 13314.70 13314.70 13314.70 24.08 29.10 29.40 29.40 29.40 29.40 29.40 1009.33 14009.41 14009.41 14009.41 14009.41 13741.77 13741.77	0.0079-030 -00079-030	F031 PERCENT	COAMY PROCENT	### ### ### ### ### ### ### ### ### ##	CGAMP BY GERMAT	19 19 19 19 19 19 19 19 19 19 19 19 19 1	######################################
740 40	\$151 951 70.94 70.94 70.94 70.94 70.94 70.94 95 STATE	9162 92.40 20.40 20.40 34.43 70.13 44.15 40.15 40.15 40.15 40.22 40.42 4	14.33.70.1011311.70113111.70113111.7011311.7011311.7011311.7011311.7011311.7011311.7011311.7011311.7011311.7011311.7011311.7011311.7011311.7011311.7011311.701	0.00.7% -00.00.00.00.00.00.00.00.00.00.00.00.00.	### ### ### ### ### ### ### ### ### ##	COANT PENCENT.	### ### ### ### ### ### ### ### ### ##	CGAMV PFECENT	19 19 19 19 19 19 19 19 19 19 19 19 19 1	######################################
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APPENDIX B-5 Hollow Cylinder Test No. B29-3

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.400mg-2- -500mg-0- -60mg-0- -	#UC	19.97 - 20.78 - 27.12 - 27.12 - 27.12 - 6198-02 - 71378-02 - 777228-02 - 77728-02 - 77728-02	-1997#-32 -24545-02 -3001#-02 -3001#-02 -1994-03 -1994-03 -1994-03 -1994-03 -21994-03	-4520E-035254E-0361 80E-0371 95E-0371 95E-	157% - 02-3648 - 02-3648 - 02-36-68 - 02-36-		CONSTANTS MTCV5 B51 14090.04 14070.75 14070.75 13349.77 1349.75 1349.75 14090.04 5173 951 14090.06 5173 951 41.01 41.01 41.05	47.91 45.22 77.61 100 FLASTIC PERCENT . A45.00 - 20 . A15.00	20, 47 00, 43 00, 43 00, 43 00, 43 00, 43 01, 43	30 64 9 7 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9
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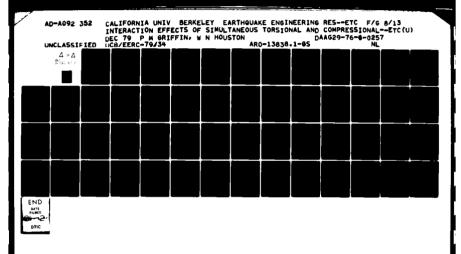
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۰,۰۹	100.70	91-45	***13	.37	-11-25-01-		-4653E-32	2:25	.7745E-02	.125#-01
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29 14 19	4250.42	.1139E -01	14813.56	.94 1 AF +00	177449.54	. 190 W - 32	16703.79	1 #96F-01	~ .A7	1.65473 1.6562 1.7817 1.9277
19	41 190 . 96	11 39E-91	13713.74	94 14F + 00	173489-54 17389-54 173889-54 173889-54 173889-54 173889-54	1903#-32	13933-53	-1762E-01 -1856E-01 -262GE-01 -2105E-01 -2525E-01		7.3949
•	37543.1	.11 TOF -01 .11 TOF -01 .11 TOF -01 .11 TOF -01 .11 TOF -01	12290.45	. 941 AF +00	173949.54	19034-07	1 2598 . 18	. 2675E-01	-,40	7.1347
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. 74	100.4A 101.03 131.03 101.03	51.45 51.55 50.15 60.67	36.37 43.96 51.45	_	16188-01-1671F-01-1732F-01-1806F-01-2000F-01-	3595t-02-	-,4794F-02 -,4791F-02	74.5A 29.44 30.35	.24076~02 .27456~02 .14406~02 .1456~02 .41476~03 .14486~03	.1924F-01 .7049F-01 .2190F-01 .714JF-01
34 19	101.07	30.13	31.33	4,41	1 e c 6E - 01		. 7279F-02	11,20	11405-07	2443F-01
49	101.22	74.20	\$1.45	7.05	. 2000E-01-	86148-02-	1 0 9 4 E -0 I	37.24	-148BF-03	2414-01
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٠,٠	5131 231	51G?	\$1 G3	PEGPEES	EDS! DERCENT	EBCENT DERCENT	PERCENT	DE CHEES	SE OCENT	SE BC ENT
29	101.26	51.45	36.18 43.74 51.30	4.99	.2084 E-01-	3754E-02	895 T - 02	33.94	2 34 4E-02	. 277 FE - 01
14	- 101.39	51.45 51.45	91 + 30	==:::4	2270E-01		- 13095-01	33.37	14205-03	.777 = 01 .297 78 - 01 .370 = -01 .7491 = -01
14	101.01	73.73	51,45 51,45	10.40	.2084E-01- .2173F-01- .2276E-01- .2387E-01- .2542F-01-	9341E-02-	1357F-01 1524F-01	30.96	.712F-07 .734E-07 .1921F-37 .1429E-03 .4492F-03	1831 -01
	35 STATE	AND ELASTIC	COMSTANT	. DATA SF						
٧U	#30YT	DEPSY PERCENT	#00V\$	EPS V PERCENT	#00T	DGAMT PERCENT	C#707	CGAMA	446	STRAT
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 	37655.91 39484.53	10-30421	13039.82		200574.36 200574.36 ED 51	*3563E-02	13074.97 11907.54	.5751 F-31	01 01	4.7604 4.6746 GAMQC

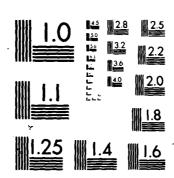
				-	PERCENT			CFGD#RS	SEECEME	
14 79 34 19	131.54 131.74 131.74 132.32 132.75 102.63	51.45 41.45 41.45 58.32 65.71 72.95	35.65 43.17 50.47 61.45 51.45	6.79	. 7640F-01- . 2770F+01- . 2920F-01- . 3096F-01- . 330 5E-01-	.50215-02-	.1411F-01	36.17 14.71 17.27	. 7 #0 TF - 07 . 7 #7 #F - 02 . 1 9# 7F - 62 . 1 #7 #F - 67 . P 78 JF - 0 T	·344 0F - 01 ·347 牙 - 01
19	132.32	5A. 37	41.45 11.45	9.13	. 30 96F -01-	. 7859F-07-	.1467F-01	17.47	.147# -07	
• 9	102.63	72.99		13.77	. 1555E- 0 I-	1181E-01-	.2324€-01	36. 77	· 1 * 2 77 - 63	.9091F-3
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74 74 14 30 44	43556.47 42 br. J. 43	.1250F-31 .1250F-01 .1250F-01 .1250F-01 .1250F-01	15242.60	.9476F+00	197114.76 192114.76 192114.76 192114.76 192114.76	.4702E-02	17575.19	.4767F-31 .5107F-31 .5489F-01 .5939F-31 .6479F-31 .7104E-01	PP PR PR 00	7.745 4.34; 4.148
32	37215.45	1250E-31	13714.14	.94265+33	192114.76	.4702F-02	14106.40	59394-31	90	1112
44	15145.76	.12504-31	13200-A1		192114-74	-A702E-02	- 11793488	71048-01	00	5 . ^ 4 3
u ,	5 () (5 ()	4 1 G2		_ DEGREES	PERCENT.	TERCENT ED45	PERCENT			DEBLER
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30	102.40	41.45	50.45	10.94	. 3446E-01-	.40526-05-	.240AE-01	39.31	15145-33	. 4784F-C
44	193.45	- 73-26	31:45	16-25-	-4237E-41-	-1210E-01-	.2978F-31	19.49	-1646F-03	.4141F-3
	54 4747F	AND FLASTIC	COMETANT	5. DATA SE	NO. 10					
-	41047 -	- DEPS No.	#00VS	- EPSV PERCENT	#057	D SANT THE STATE	E MODT	CGAMA	4 07	57847
2 <u>4</u>	42191-25	-110AE-GL	15239-55	-94115.00	109389.46	.9141 E-43.	19028-93	-7207F-37	:::	• 620
70 34	44247,13	-110AE-01 -110AE-01 -110BE-01 -110AE-01 -110AE-01	14 231 - 63	-411F+00	104389.66 145389.46 145389.46 145389.46 145389.46	. 9141F-03	10575.12	.7207F-07 .771%E-77 .8700F-77 .8980F-77 .5781E-32 .1074F-31	44	• 650 • 674 • 744 • 913
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24					DESCENT	DEDCEME			•	
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4 9	- 100 JA 100 77 100 77 100 77 100 78 100 70	86-45- 51.45 #1.70	30,49 44,10 51,45 51,45	1.39 1.60 1.90 2.32 2.00	12185-01- 12295-01- 1205-01- 1205-01- 1205-01-	DEDCEME			•	
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7 C F	100.74 100.77 100.77 100.77 100.78 100.78 100.78	\$1.45 11.45 11.70 49.71 48.00 74.50	30.40. 61.10 91.45 51.45 51.45 CONSTANTS	1 .39 1 .50 1 .50 1 .90 2 .32 2 .99	12188-01-1229-01-1229-01-1245-01-1250-01-1250-01-1250-01-1250-01-1250-01-1250-01-1250-01-1250-	DPDCENT 041048F-02- 11705-72- 1152-02- 1152-02- 1152-02- 1152-02- 1152-02- 1152-02-	-3507F-02 • 6527F-32 • 5709F-02 • 5709F-02 • 8709F-01 • 1005F-01	18.52 17.42 18.41 19.41 20.77 22.75	07528F-32 07198F-37 01791F-07 013778-07 013778-07 14008-37	1379 -0 11450 -0 1550 -0 11660 -0 11660 -0 1179 -0
7 C F	100.74 100.77 100.77 100.77 100.78 100.78 100.78	\$1.45 11.45 11.70 49.71 48.00 74.50	30.40. 61.10 91.45 51.45 51.45 CONSTANTS	1 .39 1 .50 1 .50 1 .90 2 .32 2 .99	12188-01-1229-01-1229-01-1245-01-1250-01-1250-01-1250-01-1250-01-1250-01-1250-01-1250-01-1250-	DPDCENT 041048F-02- 11705-72- 1152-02- 1152-02- 1152-02- 1152-02- 1152-02- 1152-02-	-3507F-02 • 6527F-32 • 5709F-02 • 5709F-02 • 8709F-01 • 1005F-01	- 16.52 17.42 18.41 19.41 20.72 22.25	.7524F-32 .7184F-32 .1791F-03 .1372F-03 .7912F-03 .1060F-31	-1379F-06 -145F-0 -1559F-0 -1466F-2 -1079F-0 5704T -427 -427 -427 -5372 -579
7 C F	100.74 100.77 100.77 100.77 100.78 100.78 100.78	\$6.45 51.45 21.70 40.71 44.00 74.50	30.40. 61.10 91.45 51.45 51.45 CONSTANTS	1 33 1 39 1 39 1 90 1 90 2 32 2 90 3 32 2 90 3 74 76 90 94 74 90 96 74 90	00000000000000000000000000000000000000	DFDCENT -1008F-0211708F-0211501F-021501F-021501F-021608F-022175E-023097F-038097F-03	1300F-02 - 6227F-12 - 5709F-02 - 7004F-02 - 7004F-02 - 8709F-01 (10.52 17.42 18.42 10.51 10.51 20.77 27.75 CCAMV FACE-27 -70.07-27 -77.07-27	#1C	1375 -0 .1445 -0 .1456 -0 .1566 -0 .1566 -0 .1566 -0 .1566 -0 .1566 -0 .1566 -0 .1566 -0 .157
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70 49 49 40 79 49 40 79 49 40 79 49 40 79 49 40 79 49 40 79 49 40 79 79 79 79 79 79 79 79 79 79 79 79 79	100 JA 100 -77 100 -77 100 -78 103 -78 103 -70 103 -70 103 -70 41144 64 4174 18 4174 1	AMO FLASTIC 174 18 - 01 174 18 - 01 174 18 - 01 174 W - 01	20.40 44.10 11.45 11.45 51.45 51.45 CONSTANTS MODVS 041 14012.40 14012.40 14012.40 14012.41 1301.43	1 33 1 39 1 39 1 90 1 90 2 32 2 90 3 32 2 90 3 74 76 90 94 74 90 96 74 90	00000000000000000000000000000000000000	DFDCENT -1008F-0211708F-0211501F-021501F-021501F-021608F-022175E-023097F-038097F-03	1300F-02 - 6227F-12 - 5709F-02 - 7004F-02 - 7004F-02 - 8709F-01 (17.47 17.47 19.41 10.41 10.77 27.75 27.75 27.76	# 52 AF = 32 ** 7 1 A	1378 -0 .1458 -0 .1458 -0 .155
7 CE WIJ 2716 A WI 7718 A S S S S S S S S S S S S S S S S S S	100-74-10	AMO FLASTIC 174 18 - 01 174 18 - 01 174 18 - 01 174 W - 01	20.60. 20.60. 21		# 120 # 01 - 120 # 01	DFDCENT -1008F-0211708F-0211501F-021501F-021501F-021608F-022175E-023097F-038097F-03	-1500F-021504F-021704F-02704F-02704F-02704F-02704F-02704F-02704F-02704F-02704F-027146F-0	7445F - 07 - 27 - 27 - 27 - 27 - 27 - 27 - 27	#152 NF = 32 *7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1378 -0 .1458 -0 .1458 -0 .155
7 CE WIJ 2716 A WI 7718 A S S S S S S S S S S S S S S S S S S	100-74-10	AMO FLASTIC AMO FLASTIC AMO FLASTIC AFORENT (174) F - 01 1174 W - 01 1174 W - 01 1174 W - 01 1174 S - 01	20.60. 20.60. 21		# 120 # 01 - 120 # 01	DFDCENT -1008F-0211708F-0211501F-021501F-021501F-021608F-022175E-023097F-038097F-03	-1500F-02 -1877F-32 -704F-02 -704F-02 -704F-02 -704F-02 -704F-02 -705F-01 -705F-02 -7170F-02 -7170F-02 -7170F-02 -7170F-01 -7170F-01	17.47 17.47 19.41 10.41 10.77 27.75 27.75 27.76	#17 10 10 10 10 10 10 10	1378 -0 .1458 -0 .1458 -0 .155
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40	51 G1	\$1 G?	5167 P31	MET ≜ DEGREES	F051 PF9CFNT	EPS? PERCENT	F2 53 PERCENT	FATA DEGREES	PERCENT	SAMOC PERCEN
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14	100.62	- 51 - 45 - 53- 74		1-40	- 2494F-GL	-27773E-43	+2445-04	Q.9A	.300RE-02	-317*E-4
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10	117.50	44.71	51.45 51.45	1 - 3 7	- 376 9F -01	4995E-03-	.1967E-01	4.40	.6177 -07	- 485 PF - 01
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240 Mil 250 Mil 400 Mi	118.18. 118.18. 118.19. 118.23. 118.23. 119.23. 400.47. 379.18.72.	107 -31 -43 107 -31 -43 107 -31 -43 107 -31 -43 107 -31 -43 107 -31 -31 -31 -31 -31 -31 -31 -31 -31 -31	14677.04 13920.64 13117.52 E1G3. B91 40.64 49.14 51.45 51.45 51.45	. 1974F + 00 - 1974E + 00 - 1974F + 00 - 1974F + 00 - 1974 - 1974	\$2 92 6 401. \$2 92 6 401. \$3 92 6 701. \$3 92 6 701. \$3 92 6 701. \$3 92 6 701. \$3 92 6 701. \$4 92 701.		##\$3. ##\$CENT •1655E-61 •2140E-01 •2140E-01 •3340F-01 •4117F-01 •5034F-01	-1321E-01 -1430E-01 -1450F-01 -2450F-01 -5472 -5472 -5472 -449 -449 -449 -449 -449 -449 -449 -44	# 109 -01 -03 -03 -03 -03 -04 -1109 -01 -1109 -01 -1109 -01 -1109 -01 -1109 -02 -04 -03	-2521 -774A -7015
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200 MJ 400 MJ 40	118.18. 118.18. 118.19. 118.23. 118.23. 119.23. 400.47. 379.18.72.	107 -31 -43 107 -31 -43 107 -31 -43 107 -31 -43 107 -31 -43 107 -31 -31 -31 -31 -31 -31 -31 -31 -31 -31	14677.04 13920.64 13117.52 E1G3. B91 40.64 49.14 51.45 51.45 51.45	. 1974F + 00 - 1974E + 00 - 1974F + 00 - 1974F + 00 - 1974 - 1974	\$2 92 6 401. \$2 92 6 401. \$3 92 6 701. \$3 92 6 701. \$3 92 6 701. \$3 92 6 701. \$3 92 6 701. \$4 92 701.		##\$3. ##\$CENT •1655E-61 •2140E-01 •2140E-01 •3340F-01 •4117F-01 •5034F-01	-1321E-01 -1430E-01 -1450F-01 -2450F-01 -5472 -5472 -5472 -449 -449 -449 -449 -449 -449 -449 -44	# 109 = 41 - 1109	-2524 -2524
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400 MA ADRIGATION TO ATTACAN TO A	6124 051 114.14 117.16 114.21 114.21 114.21 114.23 114.21 114.23 114.21 114.23 114.32 13091,72 13091,72 13091,73 13193,87 121,46 121	102 - 01 - 02 - 02 - 02 - 02 - 03 - 03 - 03 - 03	14677.04 13117.02 13117.02 13117.02 13117.02 140.64 49.114 51.45 51.45 51.45 10027.04 10027.07 11027.0		### ### ##############################	F082 DFDCENT 	Entj	- 1371E-01 - 1430E-01 - 1400F-01 - 5474 - 6484 - 64	# 10 SF - 03 03 03 03 03 03 04 05 04 05	-2521 -27 AA 77 -01 -27 AA 77 AA 77 -27 AA 77 AA 77 -27 AA
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٠u	5151 951	\$1.G2	5167 P51	RFTA DESRFFS	EPS 1 PF RCE NT	EPS 2 PERCENT	EPS3 PERCENT	FATA CEGRFES	EPSOC PERCENT	GAMPC PERCENT
. > •	129.34	51.45	43.27	1.23	. PAA W - 01-	. 4992#13-	- 2650F = 01		-19145-41	
16	129.3A	58.29	51.49	1.49	-94505-01	.49928-03- -5715E-03- -6636E-03-	.347 AF - 01	4.70	.1655-01	.94746-01 .99966-01 .104 6-30
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, 74	17417.20	.477#-01	17003.60	. 10? 75 +01	240074.74	.73495-03	15056.00	.1335F-01	-,07	+1 366
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44	77314.Je	. 9773E-01	17003.60 16556.46 15957.11 15299.55 14549.77	-1028E+31	7800 74 - 74 -2 400 74 - 74	**************************************	11114.00	. 1612F-01	94	. 1 454
40.1	3141 3141	954	5163 544		PERCENT	- PERCENT	- secent	DE GORE S	- PERCENS	GANCC PERCENT
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#1 76 10 11 16 17 64	Mn.wt ust 36272.31 15713.46 16302.39 35670.39 18301.65 20215.75	0F054 - 0605ENT - 3028 - 04 - 04 - 05 - 06 - 06 - 07 - 07 - 07 - 07 - 07 - 07	NGEMS - 051 14630-14 1945-10 17897-24 17271-50 14640-04 13650-74	FD SW 0 ERCENT - 10 30 E + 01 - 10 30 E + 01	MCOT- DK 1945A0-85 194780-85 194780-11 1945A0-11 1945A0-11	DGAMI OFR (FNT -6417-02 -6477-02 -6477-02 -6477-02 -6477-02	CMODT P51 1A609.80 1349.63 1769.63 1769.44 11729.79 10754.12 9403.95	-7280E-01 -7280E-01 -7797-01 -8389E-01 -9070E-01 -9880F-01 -1085F-09	00 01 01 02 02	.7701 .7613 .8001 .9001 .9005 .9005
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## 1 24 4 7 14 1 14 4 4 4 4 4 4 4 4 4 4 4 4 4	WT, WT 36272, 31 36272, 31 36272, 31 36272, 31 36272, 34 36272, 34 36272, 34 36272, 34 36272, 34 36272, 34 36272, 34 36272, 36 362	30.00 - 34.00	WGCMS - 051 16420-18-1 1943-10-1 1943-1-1 1943-	FOSW OF FOSW O	#COT.	DGAMI DGAMI DGAMI OFFICE OFFI OFFI OFFI OFFI OFFI OFFI OFFI OFF	CMODT 051 1A400.80 13769.04 11729.70 10729.12 9403.95 EN33 000000000000000000000000000000000	CGAMV 0FECFMT .77805-01 .77905-01 .790705-01 .90705-01 .98867-01 .10857-00 .10857-00 .10857-00 .20.30 .70.30 .71.16 .73.77 .75075-01 .80.307-01 .80.307-01 .75075-01 .80.307-01 .75075-01 .80.307-01 .75075-01 .75075-01 .75075-01 .75075-01 .75075-01 .75075-01 .75075-01 .75075-01 .75075-01	- 90 - 91 - 91 - 91 - 92 EPSAC OF OFF - 91 - 196, 75 - 91 - 91 - 97 - 91 - 90 - 90 - 90 - 90 - 90 - 90 - 90 - 90	- 7704 - 010 - 000





MICROCOPY RESOLUTION TEST CHART NATIONAL BUREAU OF STANDARDS-1963-A

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-	32987.14 32987.14 32349.97 33454.97 29674.14 2024.70	- 08384 - 08-9CENT - 11248-33- - 11248-33- - 11248-33- - 11248-93- - 11248-93- - 11248-98-	1620. A3 1620. A3 1620. A3 15709. 22 15137. 75 14323.04 13533. 67	- ERSY. PEPCENT - 1043 - 601 - 1043 - 601 - 1043 - 601 - 1043 - 601 - 1043 - 601	150597.00 150597.00 150597.00 150597.00 150597.00 150597.00	2GART	1348 1-27 12476-19 12551-10 10474-07 9806-94 8975-85	-1473E+80 -1479F+30 -164F+33 -2041F+30 -2199F+30	- -	1.756
-	32987-14 32987-14 32949-97 33989-94 2022-79-16 2028-79-94 2028-79-	0836M — 0879CENT	1620.A1 1620.A2 1620.A3 15705.22 15137.73 16323.04 13533.67	- task	150592.00 150592.00 150592.00 150592.00 150592.00 150592.00	36ANT	1340 1.27 12426.10 12421.10 10474.07 00476.04 0075.05	.1475E+80 .1979F+80 .194F+09 .184F+99 .2081F+99 .2199F+30		1.7044 1.3044 1.704 1.774 1.774 1.407 SANOC SERFENT
-	32987-14 32987-14 32984-97 32984-97 29974-16 29974-76 2499-96	0836M — 0879CENT	1620.A1 1620.A2 1620.A3 15705.22 15137.73 16323.04 13533.67	- task	150592.00 150592.00 150592.00 150592.00 150592.00 150592.00	36ANT	1340 1.27 12426.10 12421.10 10474.07 00476.04 0075.05	.1475E+80 .1979F+80 .194F+09 .184F+99 .2081F+99 .2199F+30		1.7644 1.3944 1.7944 1.7734 1.7734 1.9479 SA400
-	22987-16 32987-16 32984-97 32984-97 27079-16 28276-70 28589-96 8171 144-77 194-20 134-78	0836M — 6FRCENT -1129E-33 -1129E-33 -1127F-03 -1127F-03 -1127E-00 -1127E-00	1620.A1 1620.A2 1620.A3 15705.22 15137.73 16323.04 13533.67	- task	150592.00 150592.00 150592.00 150592.00 150592.00 150592.00	36ANT	1340 1.27 12426.10 12421.10 10474.07 00476.04 0075.05	.1475E+80 .1979F+80 .194F+09 .184F+99 .2081F+99 .2199F+30		1.7044 1.3044 1.704 1.774 1.774 1.407 SANOC SERFENT
-	32987-16. 32987-16. 32949-97 3398-9. 2072-79-16. 2072-79-16. 2072-79-16. 3171-77-179-20. 139-20.	05784 6790287 11780-03 1127-03 1127-03 1127-03 1127-05 1128-06 1178-06 1178-06 1178-06	1620.A1 1620.A2 1620.A3 15705.22 15137.73 16323.04 13533.67	- task	150597.00 150597.00 150597.00 150597.06 150597.06	36ANT	1340 1.27 12426.10 12421.10 10474.07 00476.04 0075.05	.1475E+80 .1979F+80 .194F+09 .184F+99 .2081F+99 .2199F+30	F0 40 03 03 01 01	1.7044 1.3044 1.704 1.774 1.774 1.407 SANOC SERFENT
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APPENDIX B-6 Hollow Cylinder Test No. B31-1

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**************************************	#70 VS	DEPCENT		- EPEN SPOCFHY				- CSIMU - GEOCENT - 2947E-01 - 2757E-01 - 2756E-01 - 3756E-01		
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Derivations

The following derivations were made for use in this study:

- C-1 Strain Tensor for Resonant Column Specimen
- C-2 Strain Tensor for Hollow Cylinder Specimen
- C-3 Calculation of Moduli for Resonant Column Tests
- C-4 Principal Stresses for Resonant Column Specimen
- C-5 Principal Stresses for Hollow Cylinder Specimen
- C-6 Calculation of Compression Modulus for Hollow Cylinder Tests
- C-7 Calculation of Volume of Hollow Cylinder Specimen
- C-8 Calculation of Relative Density

Strain Tensor for Resonant Column Specimen

From the expressions for the strains contained in Equations 2.5 through 2.11 and from the boundary conditions presented in Equations 2.12 and 2.13, the following expressions may be written:

$$\varepsilon_{\theta} = \frac{\mathbf{u}}{\mathbf{r}}$$
, because $\frac{\partial \mathbf{v}}{\partial \theta} = 0$ (C-1.1)

$$\varepsilon_{R\theta} = \frac{1}{2} \left(\frac{\partial u}{r \partial \theta} + z \Delta \theta_{o} - z \Delta \theta_{o} \right) = \frac{1}{2} \frac{\partial u}{r \partial \theta}$$
 (C-1.2)

$$\varepsilon_{RZ} = \frac{1}{2} \frac{\partial u}{\partial z}$$
, because $\frac{\partial w}{\partial r} = 0$ (C-1.3)

$$\varepsilon_{\theta \dot{z}} = \frac{1}{2} \frac{v}{z} = \frac{1}{2} r\Delta\theta$$
, where $\frac{\partial w}{\partial \theta} = 0$ (C-1.4)

and u is unknown, but

$$u = \int \varepsilon_R dr$$
 (C-1.5)

From Hooke's Law, it is known that:

$$\varepsilon_{\mathbf{x}} = \varepsilon_{\mathbf{y}} = -\mu \varepsilon_{\mathbf{z}}$$
 (C-1.6)

and
$$\varepsilon_{x} + \varepsilon_{y} + \varepsilon_{z} = \varepsilon_{vol} = (1 - 2\mu)\varepsilon_{z}$$
 (C-1.7)

Also,

$$\varepsilon_{\text{vol}} = \varepsilon_{\text{R}} + \varepsilon_{\theta} + \varepsilon_{\text{z}} = (1 - 2\mu)\varepsilon_{\text{z}}$$
 (C-1.8)

or
$$\varepsilon_{R} + \varepsilon_{\theta} = -2\mu\varepsilon_{z}$$
 (C-1.9)

Now, if one considers the new volume, $\mathbf{V}_{\mathbf{N}}$, and relates it to the old volume, $\mathbf{V}_{\mathbf{O}}$, it can be stated that:

$$v_{N} = v_{o}(1 + \epsilon_{vol}) = v_{o}[1 + (1 - 2\mu)\epsilon_{z}]$$
 (C-1.10)

But V_N is also equal to:

$$V_{N} = H_{O}(1 + \varepsilon_{z}) \cdot \pi R_{N}^{2} \qquad (C-1.11)$$

where H $_{\rm O}$ is the old height and R $_{\rm N}$ the new radius. We also know that V $_{\rm O}$ is a function of H $_{\rm O}$ and R $_{\rm O}$, the old radius, as follows:

$$V_{O} = H_{O} \cdot \pi R_{O}^{2}$$
 (C-1.12)

Combining the last three equations, C-1.10 through C-1.12, the following expression for $\mathbf{R}_{_{\mathbf{N}}}$ may be developed:

$$R_{N} = R_{O} \cdot \left[\frac{1 + (1 - 2\mu)\epsilon_{z}}{1 + \epsilon_{z}} \right]^{1/2}$$
 (C-1.13)

and it is also known that:

$$R_{N} = R_{O}(1 + \epsilon_{R}) \qquad (C-1.14)$$

It is now possible to write an expression for $\boldsymbol{\epsilon}_{R}$ as follows:

$$\varepsilon_{R} = \left[\frac{1 + (1 - 2\mu)\varepsilon_{z}}{1 + \varepsilon_{z}}\right]^{1/2} - 1$$
 (C-1.15)

Now, from Equation C-1.5:

$$u = \int \varepsilon_{R} dr \qquad (C-1.5)$$

or

$$u = r \cdot \left(\left[\frac{1 + (1 - 2\mu)\epsilon_z}{1 + \epsilon_z} \right]^{1/2} - 1 \right)$$
 (C-1.16)

It is possible now to write expressions for ϵ_{θ} , $\epsilon_{R\theta}$, and ϵ_{Rz} , and as follows:

$$\varepsilon_{\theta} = \frac{\mathbf{u}}{\mathbf{r}} = \left[\frac{1 + (1 - 2\mu)\varepsilon_{z}}{1 + \varepsilon_{z}} \right]^{1/2} - 1 = \varepsilon_{R}$$
 (C-1.17)

$$\varepsilon_{R\theta} = \frac{1}{2} \frac{\partial u}{r \partial \theta} = 0 \tag{C-1.18}$$

and

$$\varepsilon_{RZ} = \frac{1}{2} \frac{\partial u}{\partial z} = 0 \tag{C-1.19}$$

It is also known that:

$$\varepsilon_{R} + \varepsilon_{\theta} = -2\mu\varepsilon_{z}$$
 (C-1.19)

so that one may write:

$$\varepsilon_{R} = \varepsilon_{\theta} = -\mu \varepsilon_{z}$$
 (C-1.20)

The strain tensor may now be written:

$$\varepsilon = \begin{bmatrix} -\mu \varepsilon_{z} & 0 & 0 \\ 0 & -\mu \varepsilon_{z} & \frac{1}{2} r \Delta \theta_{o} \\ 0 & \frac{1}{2} r \Delta \theta_{o} & \varepsilon_{z} \end{bmatrix}$$
 (2.14)

It should be further noted that the $\epsilon_{\theta z}$ component is a function of the radius r. For practical application purposes, the average $\epsilon_{\theta z}$ will be assumed to exist at $r=\frac{2}{3}$ R, and be considered a constant at that value.

Strain Tensor for Hollow Cylinder Specimen

From the expressions for the strain contained in Equations 2.5, 2.8 through 2.11, 2.15, and 2.16, and from the boundary conditions expressed in Equations 2.12 and 2.13, the following expressions may be written:

$$\varepsilon_{R\theta} = \frac{1}{2} \left(\frac{\partial u}{r \partial \theta} + \Delta \theta_{o} z - \Delta \theta_{o} z \right) = \frac{1}{2} \frac{\partial u}{r \partial \theta}$$
(C-2.1)

$$\varepsilon_{R\dot{z}} = \frac{1}{2} \frac{\partial u}{\partial z}$$
, because $\frac{\partial w}{\partial r} = 0$ (C-2.2)

$$\varepsilon_{\theta z} = \frac{1}{2} \frac{\partial v}{\partial z} = \frac{1}{2} R_{avg} \Delta \theta_{o}$$
, where $\frac{\partial w}{\partial \theta} = 0$ (C-2.3)

and u is unknown, but

$$u = \int \varepsilon_{R} dr \qquad (C-2.4)$$

If it is assumed that Hooke's Law applies, the following expressions may be written:

$$\varepsilon_{R} = \frac{1}{E} \sigma_{R} - \frac{\mu}{E} \sigma_{Z} - \frac{\mu}{E} \sigma_{\theta}$$
 (C-2.5)

$$\varepsilon_{z} = \frac{1}{E} \sigma_{z} - \frac{\mu}{E} \sigma_{R} - \frac{\mu}{E} \sigma_{\theta}$$
 (C-2.6)

and

$$\varepsilon_{\theta} = \frac{1}{E} \sigma_{\theta} - \frac{\mu}{E} \sigma_{R} - \frac{\mu}{E} \sigma_{z}$$
 (C-2.7)

If ε_{A} = 0, Equation C-2.7 may be rewritten as follows:

$$\sigma_{\theta} = \mu (\sigma_{R} + \sigma_{Z}) \tag{C-2.9}$$

Now ε_{z} and ε_{R} may be written:

$$\varepsilon_{R} = \frac{1}{E} \sigma_{R} - \frac{\mu}{E} \sigma_{Z} - \frac{\mu}{E} \cdot [\mu (\sigma_{R} + \sigma_{Z})]$$
 (C-2.9)

$$= \left(\frac{1 - \mu^2}{E}\right) \sigma_R - \left(\frac{\mu + \mu^2}{E}\right) \sigma_Z$$

and

$$\varepsilon_{z} = \frac{1}{E} \sigma_{z} - \frac{\mu}{E} \sigma_{R} - \frac{\mu}{E} \cdot [\mu (\sigma_{R} + \sigma_{z})]$$
 (C-2.10)

$$= \left(\frac{1 - \mu^2}{E}\right) \sigma_z - \left(\frac{\mu + \mu^2}{E}\right) \sigma_R$$

Now, the change in the radial strain, $\Delta\epsilon_{\rm p},$ during loading may be written:

$$\Delta \varepsilon_{R} = \left(\frac{1 - \mu^{2}}{E}\right) \Delta \sigma_{R} - \left(\frac{\mu + \mu^{2}}{E}\right) \Delta \sigma_{Z}$$
 (C-2.11)

And because $\Delta\sigma_{R}$ = 0 in this testing program,

$$\Delta \varepsilon_{R} = \left(\frac{\mu + \mu^{2}}{E}\right) \Delta \sigma_{z} \tag{C-2.12}$$

Similarly,

$$\Delta \varepsilon_{z} = \left(\frac{1 - \mu^{2}}{E}\right) \Delta \sigma_{z} \tag{C-2.13}$$

Combining the above two equations, C-2.12 and C-2.13 gives:

$$\Delta \varepsilon_{R} = -\left(\frac{\mu + \mu^{2}}{1 - \mu^{2}}\right) \Delta \varepsilon_{z} \tag{C-2.14}$$

Now, because

$$u = \int \varepsilon_{R} dr \qquad (C-2.4)$$

for $R_{avg} \leq r \leq R_2$,

$$u = \int_{R_{avg}}^{r} -\left(\frac{\mu + \mu^{2}}{1 - \mu^{2}}\right) \cdot \Delta \varepsilon_{z} dr \qquad (C-2.15)$$

$$= -\left(\frac{\mu + \mu^{2}}{1 - \mu^{2}}\right) \cdot \Delta \varepsilon_{z} \cdot [r - R_{avg}]$$

for $R_1 \leq r \leq R_{avg}$

$$u = \int_{R_{avg}}^{r} -\left(\frac{\mu + \mu^{2}}{1 - \mu^{2}}\right) \cdot \Delta \varepsilon_{z} dr$$

$$= -\left(\frac{\mu + \mu^{2}}{1 - \mu^{2}}\right) \cdot \Delta \varepsilon_{z} \cdot [r - R_{avg}]$$
(C-2.16)

The expressions for $\epsilon_{R\theta}$ and ϵ_{Rz} may now be written:

$$\varepsilon_{R\theta} = \frac{1}{2} \frac{\partial u}{r \partial \theta} = 0 \tag{C-2.17}$$

$$\varepsilon_{RZ} = \frac{1}{2} \frac{\partial u}{\partial z} = 0 \tag{C-2.18}$$

The strain tensor may now be written:

$$\varepsilon = \begin{bmatrix} -\left(\frac{\mu + \mu^2}{1 - \mu^2}\right) \varepsilon_z & 0 & 0 \\ 0 & 0 & \frac{1}{2} R_{avg} \Delta \theta_o & 0 \\ 0 & \frac{1}{2} R_{avg} \Delta \theta_o & \varepsilon_z \end{bmatrix}$$
 (2.17)

It is also worth noting that for the special condition where $\Delta\theta_{0}=0, \text{ Equation (2.17) is transformed so that the values of } \epsilon_{R}, \ \epsilon_{\theta},$ and ϵ_{Z} are also the principal strain values, $\epsilon_{3}, \ \epsilon_{2}, \ \text{and } \epsilon_{1}$ respectively. In this special case the strain tensor is that of a plane strain condition.

Calculation of Dynamic Moduli for Resonant Column Tests Compression Modulus

The basic expression for the dynamic compression modulus, E, is as follows:

$$E = \rho \cdot V_p^2 \tag{C-3.1}$$

where ρ is the mass density of the specimen material, and where V_p is the "P-wave" (compression wave) velocity of propagation. For the solid cylindrical specimen,

$$\rho = \frac{W}{V \cdot g} \tag{C-3.2}$$

where W is the specimen weight, V is the specimen volume, and g is the acceleration of gravity. The value of $\mathbf{V}_{\mathbf{p}}$ may be calculated from the following expression:

$$v_{p} = \frac{2 \cdot \pi \cdot f_{v} \cdot L}{\psi_{v}}$$
 (C-3.3)

where f_v is the resonant frequency in compression, L is the length (or height) of the cylindrical specimen and where ψ_v is the root of the frequency equation:

$$\psi_{\mathbf{V}} \cdot \tan \psi_{\mathbf{V}} = \frac{\mathbf{W}}{\mathbf{W}_{\mathbf{TOP}}}$$
 (C-3.4)

where $W_{\overline{TOP}}$ is the top cap weight. The value of $W_{\overline{TOP}}$ in this testing series is 1576.6 gm.

During the resonant column test, the resonant frequency is influenced by dynamic coupling between the sample and the cap system. If a test could be conducted under ideal conditions, where the top cap weight was

negligible when compared with the specimen weight, there would be no coupling effect and f_v would reflect the true resonant frequency of the soil specimen. In this special case, $\psi_v = \pi/2$, and Equation C-3.3 reduces to the following:

$$v_{p} = 4 \cdot f_{v} \cdot L \qquad (C-3.5)$$

In the more general case, the dynamic compression modulus may be calculated directly from Equations C-3.1 through C-3.4. The only difficulty in this calculation, albeit a minor one, is that the value of $\psi_{_{\mbox{$V$}}}$ must be calculated from Equation C-3.4 by an iterative process.

These calculations are made in Computer Program RC, which is included as Appendix D-1.

Shear Modulus

The basic expression for the dynamic shear modulus, G, is as follows:

$$G = \rho \cdot V_s^2 \tag{C-3.6}$$

where V_g is the "S-wave" (shear wave) velocity of propagation, and where ρ is the mass density of the specimen material and is defined in Equation C-3.2. The value of V_g may be calculated as follows:

$$v_{s} = \frac{2 \cdot \pi \cdot f_{s} \cdot L}{\psi_{c}}$$
 (C-3.7)

where f_g is the resonant frequency in torsional shear, L is the length (or height) of the cylindrical specimen, and where ψ_g is the root of the frequency equation:

$$\psi_{s} \cdot \tan \psi_{s} = \frac{J}{J_{o}}$$
 (C-3.8)

where J is the torsional moment of inertia of the specimen, and J_{o} is the torsional moment of inertia of the top cap system. The units of J and J_{o} are mass-length-time². For a cylindrical specimen, J may be calculated as follows:

$$J = \frac{W \cdot D^2}{8 \cdot q} \tag{C-3.9}$$

where D is the diameter of the specimen.

As discussed earlier, the factor $\psi_{\rm g}$ is necessary to correct for the dynamic coupling influence of the top cap system upon the measured resonant frequency. If the torsional moment of inertia of the top cap system were negligible when compared with the torsional moment of inertia of the specimen, $\psi_{\rm g} = \pi/2$, and Equation C-3.7 would reduce to:

$$V_{S} = 4 \cdot f_{S} \cdot L \qquad (C-3.10)$$

In the more general case, the dynamic shear modulus may be calculated directly from Equations C-3.6 through C-3.9. The value of J_{0} in this testing series is:

$$J_{\odot} = 31.45 \text{ gm-cm-sec}^2$$
 (C-3.11)

The value of $\psi_{\mathbf{g}}$ must be calculated from Equation C-3.8 by an iterative process.

These calculations are made in Computer Program RC, which is included as Appendix D-1.

Principal Stresses for Resonant Column Specimen

The Mohr's circle diagram in Figure C4-1 represents the state of stress within the resonant column specimen, viewed from the " θz plane", for the condition of simultaneous maximum vertical compression and torsional shear stress. The principal stresses are shown on this figure as σ_1 , σ_2 , and σ_3 . The values $\Delta \sigma_z$ and $\Delta \tau_{\theta z}$ as recorded raw data were double amplitude values and thus must be divided by 2 for plotting on Mohr's circle.

The center of the Mohr's circle, σ_{ct} , may be calculated as follows

$$\sigma_{\rm ct} = \sigma_{\theta} + \frac{\Delta \sigma_{\rm z}}{4} \cdot \sin(\omega_{\rm z}t)$$
 (3.18)

where $\omega_{_{\mathbf{Z}}}$ is the rotational frequency of vertical loading and is constant. Note that:

$$\sigma_{\theta} = \sigma_{R} = \sigma_{1c} = \sigma_{2c} = \sigma_{3c}$$
 (C-4.1)

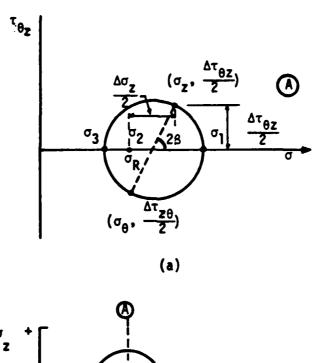
for this testing series, where σ_{1c} , σ_{2c} , and σ_{3c} are the principal stresses during consolidation.

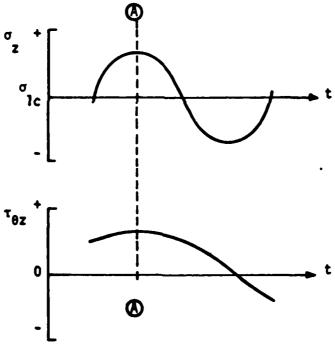
From the figure,

$$\tan(2\beta) = \begin{bmatrix} \frac{\Delta \tau_{\theta z}}{2} \cdot \sin(\rho \cdot \omega_{z} \cdot t) \\ \frac{\Delta \sigma_{z}}{4} \cdot \sin(\omega_{z} \cdot t) \end{bmatrix}$$
 (C-4.2)

where $\rho = \frac{f_T}{f_z} = \frac{\omega_T}{\omega_z}$

Now:
$$\beta = \frac{1}{2} \tan^{-1} \left[\frac{2 \cdot \Delta \tau_{\theta z} \cdot \sin(\rho \cdot \omega_{z} \cdot t)}{\Delta \sigma_{z} \cdot \sin(\omega_{z} \cdot t)} \right]$$
(3.17)





(b)

FIGURE C4-1 MOHR'S CIRCLE IN STRESS (a) AND STRESS TIME HISTORIES (b) FOR SIMULTANEOUS DYNAMIC LOADING OF RESONANT COLUMN SPECIMENS UNDER CONDITION OF MAXIMUM INSTANTANEOUS VERTICAL AND TORSIONAL STRESS

Now:

$$\sigma_1 = \sigma_R + \frac{\Delta \sigma_z}{2} \cdot \sin(\omega_z t) + \frac{\Delta \tau_{\theta z}}{2} \cdot \sin(\rho \cdot \omega_z \cdot t) \cdot \tan(\beta)$$
 (3.14)

Note that because of the geometry of the Mohr's circle, $\boldsymbol{\sigma}_1$ may also be written:

$$\sigma_1 = \sigma_R + \frac{\Delta \sigma_z}{4} \cdot \sin(\omega_z \cdot t)$$

$$+\sqrt{\left[\frac{\Delta\sigma_{z}}{4}\cdot\sin(\omega_{z}t)\right]^{2}+\left[\frac{\Delta\tau_{\theta z}}{2}\cdot\sin(\rho\cdot\omega_{z}\cdot t)\right]^{2}}$$
 (C-4.3)

The remaining two principal stresses are as follows:

$$\sigma_2 = \sigma_R \tag{3.15}$$

and

$$\sigma_3 = 2\sigma_{ct} - \sigma_1 \tag{3.16}$$

It should be noted that σ_1 and σ_3 are the major and minor principal stresses at the condition of simultaneous maximum vertical compression and torsional shear stress as shown in Figure C4-1(a). As time goes on, however, the amplitude and direction of these principal stresses will change, and at some point " σ_1 " will be numerically smaller than " σ_3 ". At that point the principal stress directions "reverse" and σ_3 becomes the new σ_1 .

Principal Stresses for Hollow Cylinder Specimen

The Mohr's circle diagram in Figure C5-1 represents the state of stress within the hollow cylinder specimen, viewed from the " θz plane", for the condition of simultaneous maximum vertical compression and torsional shear stress. The principal stresses are shown on the figure as σ_1 and σ_2 . The minor principal stress, σ_3 , is unchanged during loading and is:

$$\sigma_3 = \sigma_{3c} = \sigma_{R} \tag{C-5.1}$$

The vertical and torsional stresses shown in Figure C5-1(a) may be written

$$\frac{\partial \sigma_{z}}{\partial t} = \frac{\Delta \sigma_{z}}{2} \cdot \sin(\lambda \cdot t) \tag{C-5.2}$$

and

$$\frac{\partial \tau_{\theta z}}{\partial t} = \frac{\Delta \tau_{\theta z}}{2} \cdot \sin \left[(\lambda - \alpha) \cdot t \right]$$
 (C-5.3)

where λ is the rotational frequency of loading and is constant.

The center of the Mohr's circle, σ_{ct} , at any time may be calculated as follows:

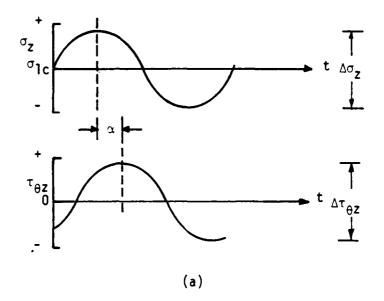
$$\sigma_{\text{ct}} = \frac{\left[\sigma_{1c} + \frac{\Delta\sigma_{z}}{2} \cdot \sin(\lambda \cdot t)\right] + \left[\sigma_{2c} + \frac{\Delta\sigma_{\theta}}{2} \cdot \sin(\lambda \cdot t)\right]}{2}$$
 (C-5.4)

and because $\sigma_{2c} = \mu(\sigma_{1c} + \sigma_{3c})$, and $\Delta\sigma_{\theta} = \mu\Delta\sigma_{z}$,

$$\sigma_{ct} = \frac{\left[\sigma_{1c} + \frac{\Delta\sigma_{z}}{2} \cdot \sin(\lambda \cdot t)\right] + \left[\mu(\sigma_{1c} + \sigma_{3c}) + \frac{\mu\Delta\sigma_{z}}{2} \cdot \sin(\lambda \cdot t)\right]}{2}$$

which reduces to:

$$\sigma_{ct} = \left(\frac{1+\mu}{2}\right)\sigma_{1c} + \left(\frac{1+\mu}{4}\right)\Delta\sigma_{z} \cdot \sin(\lambda \cdot t) + \frac{\mu\sigma_{3c}}{2} \qquad (3.37)$$



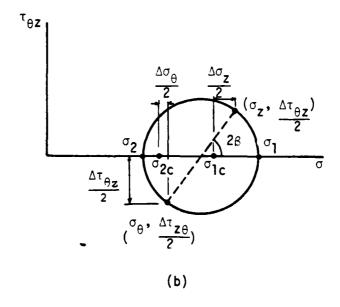


FIGURE C5-1 STRESS TIME HISTORIES (a) AND MOHR'S CIRCLE IN STRESS (b) FOR SIMULTANEOUS DYNAMIC LOADING OF HOLLOW CYLINDER SPECIMENS UNDER CONDITION OF MAXIMUM INSTANTANEOUS VERTICAL AND TORSIONAL STRESS

Now, from Figure C5-1, the following expression may be written:

$$\tan(2\beta) = \begin{cases} \frac{\Delta \tau_{\theta z}}{2} \cdot \sin[(\lambda - \alpha) \cdot t] \\ \sigma_{1c} + \frac{\Delta \sigma_{z}}{2} \cdot \sin(\lambda \cdot t) - \sigma_{ct} \end{cases}$$
 (C-5.6)

or
$$\beta = \frac{1}{2} \tan^{-1} \left\{ \frac{\Delta \tau_{\theta z} \cdot \sin[(\lambda - \alpha) \cdot t]}{2[\sigma_{1c} + \frac{\Delta \sigma}{2} \cdot \sin(\lambda \cdot t) - \sigma_{ct}]} \right\}$$
(3.36)

The principal stresses may now be written:

$$\sigma_1 = \sigma_{1c} + \frac{\Delta \sigma_z}{2} \cdot \sin(\lambda \cdot t) + \frac{\Delta \tau_{\theta z}}{2} \cdot \sin[(\lambda - \alpha) \cdot t] \cdot \tan(\beta)$$
 (3.33)

$$\sigma_2 = 2\sigma_{ct} - \sigma_1 = \sigma_1 - 2(\sigma_1 - \sigma_{ct})$$
 (3.34)

and
$$\sigma_3 = \sigma_{3c} = \sigma_R$$
 (3.35)

It should be noted that with time, the numerical values of the major and minor principal stresses will change. If $\frac{\Delta\sigma_z}{2}$ exceeds $0.85 \cdot \sigma_{3c}$ at any time, then the principal stress directions will "reverse" for a period during each cycle of loading, and σ_3 will become the new σ_1 for that period.

Calculation of Compression Modulus for Hollow Cylinder Tests

From Hooke's Law the strains may be written as follows:

$$\epsilon_{\mathbf{z}} = \frac{1}{E} \sigma_{\mathbf{z}} - \frac{\mu}{E} (\sigma_{\mathbf{R}} + \sigma_{\theta})$$
 (C-6.1)

$$\varepsilon_{R} = \frac{1}{E} \sigma_{R} - \frac{\mu}{E} (\sigma_{z} + \sigma_{\theta})$$
 (C-6.2)

and

$$\varepsilon_{\theta} = \frac{1}{E} \sigma_{\theta} - \frac{\mu}{E} (\sigma_{z} + \sigma_{R})$$
 (C-6.3)

Because ε_{θ} = 0 in this testing series, Equation C-6.3 may be rewritten as follows:

$$\sigma_{\theta} = \mu (\sigma_{z} + \sigma_{R}) \tag{C-6.4}$$

Now Equation C-6.1 may be written:

$$\varepsilon_z = \left(\frac{1 - \mu^2}{E}\right) \sigma_z - \left(\frac{\mu + \mu^2}{E}\right) \sigma_R$$
 (C-6.5)

During dynamic loading, the change in vertical strain may be written as follows:

$$\Delta \varepsilon_{z} = \left(\frac{1 - \mu^{2}}{E}\right) \Delta \sigma_{z} - \left(\frac{\mu + \mu^{2}}{E}\right) \Delta \sigma_{R}$$
 (C-6.6)

but because $\Delta \sigma_{R} = 0$,

$$\Delta \varepsilon_{z} = \left(\frac{1 - \mu^{2}}{E}\right) \Delta \sigma_{z} \tag{C-6.7}$$

or

$$E = (1 - \mu^2) \frac{\Delta \sigma_z}{\Delta \varepsilon_z}$$
 (4.13)

Calculation of Volume of Hollow Cylinder Specimen

For most tests in this testing program, the specimen height, H, and thickness, T, were measured immediately following the initial isotropic increment of consolidation. For some earlier tests, only the sample height was accurately measured. Assuming the specimen in Figure C7-1(a) is typical, then:

$$T = R_2 - R_1$$
 (C-7.1)

 R_{avg} is approximately 10.0 cm (3.937 in) for all samples.

The volume of a specimen, V, is:

$$V = H \cdot \pi \cdot (R_2^2 - R_1^2)$$
 (C-7.2)

but because $R_2 = R_{avg} + \frac{T}{2}$, and $R_1 = R_{avg} - \frac{T}{2}$,

$$V = H \cdot \pi \cdot \left[\left(R_{avg} + \frac{T}{2} \right)^2 - \left(R_{avg} - \frac{T}{2} \right)^2 \right] \qquad (C-7.3)$$

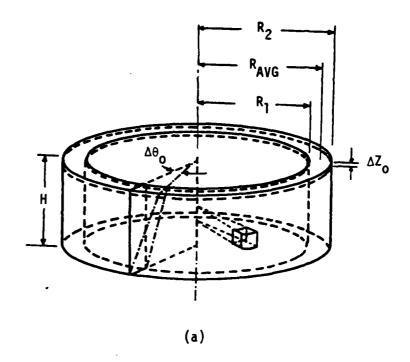
Expanding this expression gives:

$$V = H \cdot \pi \cdot \left[\left(R_{avg}^2 + T \cdot R_{avg} + \frac{T^2}{4} \right) - \left(R_{avg}^2 - T \cdot R_{avg} + \frac{T^2}{4} \right) \right]$$
(C-7.4)

or
$$V = H \cdot \pi \cdot 2 \cdot T \cdot R_{avg}$$
 (C-7.5)

To calculate the volume in cubic inches, since T and H are measured in inches, Equation C-7.5 may be written:

$$V = H \cdot T \cdot 24.73695$$
 (C-7.6)



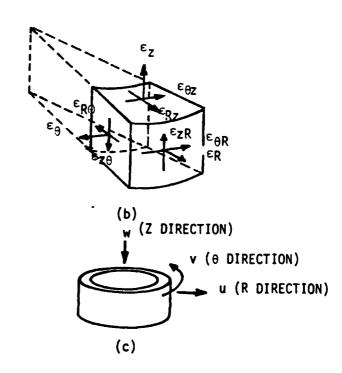


FIGURE C7-1 FREE BODY DIAGRAM AND STATE OF STRAINS FOR HOLLOW CYLINDER SPECIMENS

If T was not recorded, an average value of T = 0.779 in. may be used with good accuracy, and Equation C-7.5 may be written:

 $V \simeq H \cdot 19.270084$

(C-7.7)

Equations C-7.6 and C-7.7 are used in Computer Program HC, included as Appendix D-2, to calculate the volume of hollow cylinder specimens.

Appendix C-8

Calculation of Relative Density

The soil used in this study was Monterey No. 0 sand, a uniformly graded, fine grained quartz sand processed from beach sand. A gradation analysis of this sand is shown in Figure C8-1.

In order to calculate relative densities for the various specimens constructed through this study, laboratory tests were performed to determine the maximum and minimum densities, $\gamma_{d,max}$ and $\gamma_{d,min}$, which could be achieved. These two limiting densities were found to be as follows:

$$\gamma_{\rm d \ max} = 1.70709 \ \rm gm/cc$$
 (C-8.1)

and

$$\gamma_{d \min} = 1.42532 \text{ gm/cc}$$
 (C-8.2)

The relative density of a sand, D_{R} , is defined as follows:

$$D_{R} = \frac{e_{max} - e}{e_{max} - e_{min}} \times 100\%$$
 (C-8.3)

where e is the void ratio and is defined as:

$$e = \frac{v}{v}$$
 (C-8.4)

where $\mathbf{V}_{\mathbf{V}}$ is the volume of voids, and $\mathbf{V}_{\mathbf{S}}$ the volume of solids, which may also be written:

$$V_{s} = \frac{W_{s}}{G_{s} \cdot \gamma_{w}}$$
 (C-8.5)

In this equation, W_S is the weight of soil (weight of voids is assumed negligible), G_S is the unit weight of the solid materials, and γ_W is the unit weight of water, $\gamma_W = 1$ gm/cc. The volume, V, may be written:

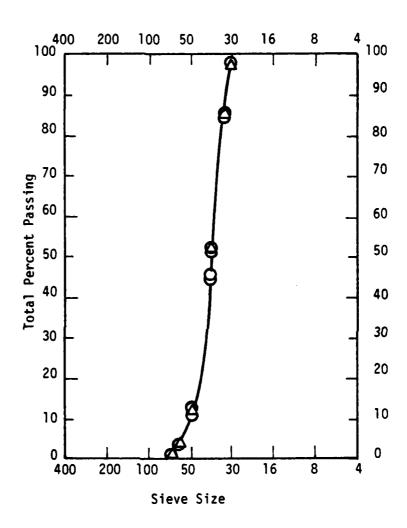


FIGURE C8-1 GRADATION ANALYSIS OF MONTEREY NO. O SAND USED IN THIS STUDY

$$V = V_v + V_s \tag{C-8.6}$$

Now e may be rewritten:

$$e = \frac{V_{v}}{V_{s}} = \frac{V_{v} \cdot G_{s} \cdot \gamma_{w}}{W_{s}}$$
 (C-8.7)

Also, because $\gamma_d = \frac{W_s}{V}$,

$$v_{v} = \frac{w_{s}}{\gamma_{d}} - v_{s} \qquad (C-8.8)$$

and

$$e = \frac{G_s \cdot Y_w \cdot W_s}{Y_d \cdot W_s} - 1$$
 (C-8.9)

which reduces to:

$$e = \frac{G_s \cdot \gamma_w}{\gamma_d} - 1 \tag{C-8.10}$$

The relative density may now be written:

$$D_{R} = \frac{\left(\frac{G_{S} \cdot \gamma_{W}}{\gamma_{d \min}} - 1\right) - \left(\frac{G_{S} \cdot \gamma_{W}}{\gamma_{d}} - 1\right)}{\left(\frac{G_{S} \cdot \gamma_{W}}{\gamma_{d \min}} - 1\right) - \left(\frac{G_{S} \cdot \gamma_{W}}{\gamma_{d \max}} - 1\right)} \times 100\%$$
 (C-8.11)

or

$$D_{R} = \frac{\left(\frac{1}{\gamma_{\text{dmin}}}\right) - \left(\frac{1}{\gamma_{\text{d}}}\right)}{\left(\frac{1}{\gamma_{\text{dmin}}}\right) - \left(\frac{1}{\gamma_{\text{dmax}}}\right)} \times 100$$

This Equation further reduces to:

$$D_{R} = \frac{0.701597 - \frac{1}{\gamma_{d}}}{0.1158046} \times 100$$
 (C-8.13)

Equation C-8.13 was used to calculate relative density during this study.

Appendix D

Computer Programs

The following computer programs were developed for use in this study:

- D-l Program RC--A Program to Reduce Data from the Triaxial Resonant

 Column Testing Series.
- D-2 Program HC--A Program to Reduce Data from the Thin-Walled Hollow Cylinder Testing Series.

APPENDIX DI

Program RC--A Program to Reduce Data from the Triaxial Resonant Column Testing Series

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PROGRAM RC (IMPUT-OUTPUT-PUMCH)
       A COMPUTER PROGRAM TO REDUCE DATA FROM THE TRIAXIAL RESUNANT COLUMN
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AND THE RESONANT FREQUENCY IN MEDICAL AND THESIONAL LOADING. THE
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APPENDIX D2

Program HC--A Program to Reduce Data From the Thin-Walled Hollow Cylinder Testing Series

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DROGRAM HE (THOUT . OUTDUT)
         A COMPUTER PROGRAM TO REDUCE DATA FROM THE HOLLOW CYLINDER TESTING
         CAREN BY DAT SOIFFIN -- ACTAGED DE. 1070
        THIS PROSPAN IS RASED ON PLASSICAL ELASTIC THEORY AND HORKE'S LAW-
THE INDUT PARRYTERS INCLUDE THE CONFINING DEFERUPE, THE VERTICAL
AND TORSIONAL SIRESS AND STRAIN READINGS, AND THE PHASE LEG RETWEEN
VERTICAL AND TORSIONAL LOADING. THE PROGRAM CALCULATES THE
MAGNITURE AND DIRECTION OF THE ORIGINAL STREESS AND INCOMMENTAL
DRINCIPAL STRAINS. THE TANGENT AND SECANT COMPRESSION MODULE, THE
SHEAR MODULUS, AND THE INCREMENTAL OCTAMEDRAL STREEN AS A SUNCTION
OF THE POISSON'S RATIO. WITH THE COMPINED LOADING TESTS THIS
DROGRAM ALSO CALCULATES AN EFFECTIVE POISSON'S RATIO. THE CHEAR
STRAIN, AND THE OCTAMEDRAL SHEARING STRAIN.
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